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FEASIBILITY STUDY OF A SOLAR ENERGY POWERED
SORPTION DEHUMIDIFICATION SYSTEM

by

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NOMENCLATURE

| <u>SYMBOL</u> | <u>DEFINITION</u> |
|-----------------------|--|
| <u>English Letter</u> | |
| A | Solar Collector Area, m ² |
| c | Drying constant |
| C | Flag indicating income-producing or nonincome-producing |
| C _A | Cost per area of solar collector |
| C _E | Equipment cost |
| C _f | Cost of fuel |
| d | Market discount rate |
| D | Ratio of downpayment to initial investment |
| DEP | Depreciation tax deduction |
| F | Fraction of energy supplied by Solar |
| i | Inflation rate |
| i _f | Fuel inflation rate |
| L | Load on Dryer |
| LCC _C | Life cycle cost - conventional system |
| LCC _S | Life cycle cost - solar energy system |
| m | Annual mortgage interest rate |
| M _E | Commodity initial equilibrium moisture content, Dry Basis |
| M _s | Ratio of first year miscellaneous costs to initial investment |
| n | Drying constant |
| N _d | Depreciation lifetime, in years |
| N _e | Equipment lifetime, in years |
| N _L | Term of loan, in years |

| <u>SYMBOL</u> | <u>DEFINITION</u> |
|---------------------|---|
| N_{min} | Years over which mortgage payments contribute to the analysis |
| N'_{min} | Years over which depreciation tax deductions contribute to the analysis |
| P_1 | Ratio of life cycle fuel costs to the first year fuel energy cost |
| P_2 | Ratio of the owning cost to the initial cost |
| PWF | Present Worth Factor |
| Q_s | Heat Gain in the Solar Collector |
| RH | Relative Humidity |
| R_v | Ratio of resale value at end of the period of the analysis to the initial investment |
| t | Property tax rate based on assessed value |
| \bar{t} | Effective tax rate |
| V | Ratio of assessed valuation of the system in the first year to the initial investment in the system |
| <u>Greek Letter</u> | |
| α | Slope of constant F_i lines on a psychrometric chart |
| γ | Specific capacity ratios between the moist silica gel and the air water vapor mixture |

CHAPTER I

INTRODUCTION

Dehumidified air finds a multitude of applications, including humidity control for marine storage areas, in the production of foods such as breakfast cereals and dried fruits and vegetables, moisture removal from paper and the surface of fresh citrus fruit, reduction of lumping in powdered substances manufactured by chemical and pharmaceutical companies, air conditioning for human comfort and the drying of agricultural and lumber products. These applications conventionally utilize fossil fuels, and the rising cost of these fuels has encouraged interest in utilizing solar energy as an alternative.

Two of the researchers in this area, Singer (36) and Ananth (3), have developed experimental and computer models of a solar energy powered sorption dehumidification system to produce dehumidified air. This current study takes one step further towards developing a working system for commercial use.

Since the system is powered by solar energy, the location and time of year of operation will affect the performance of the system. Also, the economic feasibility of the system will be a deciding factor before it will be chosen to be built.

Therefore, the objective of this thesis is to develop a computer model that provides weather data to the computer model of the system, and to incorporate that with an economic analysis. These three parts of the computer program - Weather Data, System and Economic Analysis - may be carried out independently of each other with data supplied from one to the other, or they may be combined into one large computer program.

CHAPTER II

SOLAR ENERGY POWERED SORPTION DEHUMIDIFICATION SYSTEM

The system in this study at Kansas State University was initially developed by Singer (36) and further enhanced by Ananth (3). It utilizes a desiccant to dehumidify air through adsorption, and solar energy to regenerate the desiccant.

The sorption process has been described as the movement of water vapor molecules towards the desiccant surface caused by unbalanced molecular forces occurring at the surface of the desiccant. On a macroscopic level, the mass transfer can be thought of as taking place because of a difference in water vapor partial pressures in the desiccant and in the air. When an active, anhydrous desiccant is brought in contact with moist air, the water vapor partial pressure of the air exceeds the water vapor partial pressure within the desiccant. Passing a heated air stream through the desiccant evaporates the sorbed moisture, thereby regenerating the desiccant.

Sorbent is the term referring to the desiccant which sorbs, in this case, moisture from the moist air stream. The sorbate is the substance sorbed, i.e. moisture from the air. The sorbent can be solid or liquid. Sorption with solid sorbents may be accomplished by either absorption or adsorption, where sorption with liquid sorbents occurs only by absorption. Through absorption, the desiccant reacts with the sorbate, changing either chemically or physically. Lithium chloride is one such desiccant which undergoes a chemical change to a hydrate state, where additional moisture absorption would cause the sorbent to dissolve into solution. Ordinary table salt is another example of an absorbing sorbent. With adsorption, the sorbent will not react chemically or physically with the sorbate. Examples of adsorbing desiccants are activated

alumina, silica gel, activated bauxites and activated charcoals.

Much research has been conducted in the areas of solar dehumidification utilizing liquid and solid sorbents. Disadvantages of a liquid system are the solution leakage possibilities, solution entrainment in the processed air, cooling tower requirements, bulky size and increased number of pumps, valves and heat exchangers which would also require maintenance. Disadvantages of a solid desiccant system compared to a liquid desiccant is a higher air stream pressure drop through the desiccant requiring increased fan power. The liquid system can operate isothermally, giving a low dry bulb temperature exiting from the dehumidifier. This is preferable over a solid desiccant system for an application of air conditioning. The solid desiccant system operates adiabatically, yielding a high exit dry bulb temperature, which is better adapted to drying operations. For this study, because of the simplicity of design and additional advantage of inexpensive materials utilized, the solid desiccant system was chosen.

The selection of a solid desiccant should be based on the ease of regeneration, cost, effective lifetime and reactivity with air which would yield undesirable gases such as NO_x , SO_2 and CO_2 . The ease of regeneration depends on a regeneration temperature obtainable with flat plate solar collectors and the capacity to adsorb moisture from an air stream. The desiccants taken into consideration for this study included activated alumina, molecular sieve, lithium chloride on a honeycomb matrix, silica gel in a packed bed and silica gel on Teflon. One of the disadvantages of a silica gel bed is the relatively low cost, almost infinite lifetime of regeneration/dehumidification cycling, inert characteristics and superior ease of regeneration, Singer chose a bed of silica gel as the desiccant.

The bed of solid desiccant can either be fixed, or it can rotate. If

fixed, the bed is first regenerated and then can process unconditioned air. If a continuous operation is desired, two fixed beds or a rotating bed must be employed. Using two fixed beds, one processes the moist air while the other is regenerated. If a rotating bed is used, one half of the wheel undergoes regeneration while the other half dehumidifies moist air. Disadvantages of these techniques are that the fixed bed requires more ductwork, directional dampers, timers and dew point sensing devices, and with a rotating bed, the air leaks around the wheel seals. Because of the simplicity, the rotating bed was chosen for these initial studies.

SOLAR DEHUMIDIFICATION UTILIZING A ROTATING, SOLID DESICCANT

The system is composed of a rotating sensible heat exchanger, a rotating desiccant bed, flat plate solar collectors, an auxiliary heater, fans and ductwork. The dehumidifier shown schematically in Figure 1 houses the heat exchanger and the desiccant bed. To allow for continuous operation, it is split into two isolated flow paths - the process side and the regeneration side. On the process side flows air which is to be dehumidified, and the regeneration flow stream is used to regenerate the desiccant. The desiccant bed adsorbs moisture on the process half of the bed and is dried by concurrently flowing heated air from the solar collectors on the regeneration side. If cool, dry air is desired, i.e. for air conditioning, the sensible heat exchanger is used. This cools the process air, and preheats regeneration air entering the solar collector. To maintain a particular minimum regeneration air temperature entering the desiccant bed, the auxiliary heater supplies any additional energy required during hours of low insolation.

Figure 2 is a plot of each state of the dehumidification process on a psychrometric chart. On the process side, ambient air to be dehumidified

FIGURE 1
SCHEMATIC OF THE SOLAR
DESICCANT DEHUMIDIFIER SYSTEM

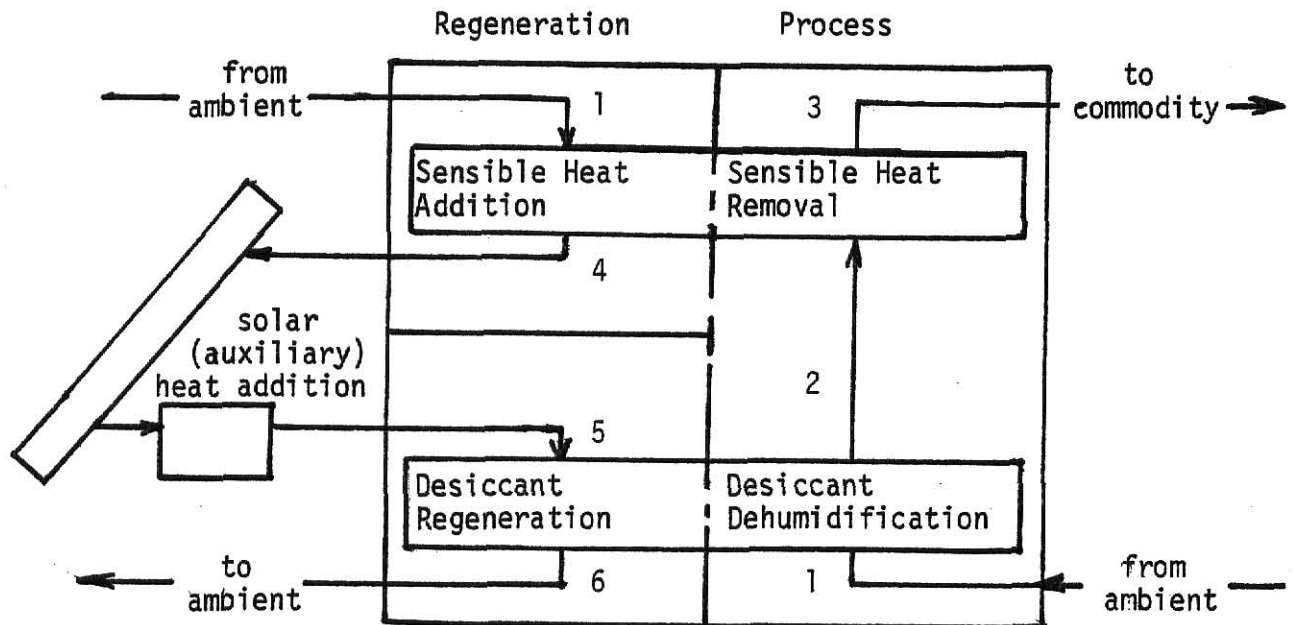
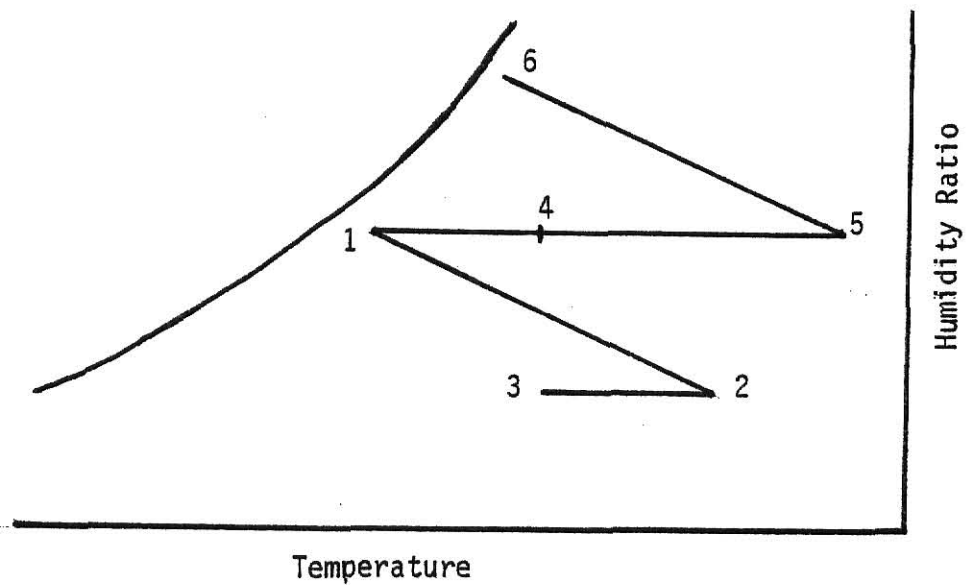


FIGURE 2
PSYCHROMETRIC PLOT OF
DEHUMIDIFICATION PROCESS



enters the dehumidifier at condition 1 and passes through the desiccant bed. The desiccant adsorbs moisture from the air, decreasing the humidity ratio and raising the air dry bulb temperature. This increase in temperature results from the heat of condensation, released as moisture changes from a vapor to liquid state. The air is dehumidified almost adiabatically, which very nearly follows a constant wet bulb line on the psychrometric chart, and exits the desiccant bed at condition 2.

If the sensible heat exchanger is used, the dry bulb temperature of the air is reduced and exits the dehumidifier at condition 3. For applications desiring hot dry air, the heat exchanger would not be utilized. In this case, the air would exit the dehumidifier hot and dry at condition 2, flowing on to a dryer.

On the regeneration side, ambient air at condition 1 enters the dehumidifier, passes through the sensible heat exchanger gaining the heat lost by the process stream air. The flow stream is then directed through the solar collector followed by the auxiliary heater. Through some combination of the heat exchanger, solar collector and auxiliary heater, the regeneration air is heated to a temperature sufficient for desiccant regeneration at condition 5. The air then flows through the desiccant, regenerating the bed while humidifying the air. Regeneration air exiting the desiccant bed at condition 6 is released to the surrounding atmosphere.

COMPUTER PROGRAM

A major portion of Singer's work was the development of a computer program to predict the performance of the dehumidification system. It is this program which is incorporated as subroutine SYSTEM into the main Weather Data and Economic Analysis computer program of the present study. Singer wrote the

program in Fortran Watfiv to use on the KSU IBM 370 digital computer. Ananth rewrote the program in Basic language utilizing the HP 9845 B desk top computer. Ananth also modified the program to accept different humidity ratios, mass flow rates and peripheral leakage rates for the process and regeneration streams. But, to combine this program with the TMY weather data tapes, requires use of the KSU IBM 370 computer. Ananth's program was rewritten in Fortran Watfiv, modified to reduce computation time and combined with the weather data and economic analysis parts of the program. Due to the length, intricacy and modifications made, subroutine SYSTEM deserves some discussion.

The original program consisted of a main program, MAIN and seven subroutines, HUMID, EGV, ALFAV, LEAK, COLPER, COLEFF, and HEATX. The MAIN, renamed as SYSTEM, serves as a source for calling subroutines HUMID, COLPER and HEATX and also allows for data input and output. Subroutine HUMID models the properties of the silica gel bed utilizing subroutines EGV and ALFAV. LEAK subroutine calculates the outlet temperature and humidity conditions of the air streams considering leakages around the rotating wheels. The solar collector performance, or useful heat gain, is calculated in COLPER and calls COLEFF to calculate the collector efficiency. HEATX models the sensible heat exchanger performance.

A flowchart of the modified SYSTEM program is included in Appendix A. The original program required much computation time, hence, most modifications were geared towards decreasing the execution time. All Call statements for subroutines were either simplified or replaced with the actual subroutine called. Since subroutines LEAK, COLEFF and HEATX were fairly short and were called only one or two times, they were deleted as subroutines and replaced the Call statements in their entirety. The remaining subroutines HUMID, EGV, ALFAV, and COLPER were originally called with a list of parameters. To

simplify these Call statements, the parameters were deleted. Some difficulty arose because of the freedom within the Fortran Watfiv and Basic languages allowing different names for the same parameter between a subroutine and the main part of the program. This was alleviated by careful manipulation, renaming a multitude of parameters and by setting the appropriate variables in the subroutine equal to their counterpart found previously in the list of parameters. This must be done immediately preceeding and following each Call statement.

Subroutine SYSTEM allows for data input and output. The input read on data cards includes:

1. The program option, indicating if a minimum regeneration temperature for the desiccant is desired.
2. Collector data - air mass flow rates, collector area and slope, latitude and longitude. Note that the slope of the collector is the actual slope from the ground, minus the latitude.
3. Dehumidifier data - duct temperature drops, silica gel bed properties, rotational speed of the wheel, the minimum regeneration temperature, leakage rates and heat exchanger effectiveness.

Input provided by the Weather Data program through the common statement are:

4. Calendar data - month, day, year, hour, number day of the year and number of hours in the study.
5. Ambient dry bulb and dew point temperatures and ambient humidity ratios.
6. Solar radiation, percent diffuse radiation, percent direct radiation and the ground reflectance ratio.
7. Wind direction and velocity.
8. Station Number

Output provided through the common statement for the Economic Analysis is the hourly humidity ratio of the process air exiting the dehumidifier. Other output printed in the program consists of temperature and humidity maps for the process and regeneration sides, heat balance of the dehumidifier and the collector performance, all on an hourly basis. The input parameters included above in items 1-3 are also output to verify correct values.

Singer originally developed the program with the option of using one of four collector types - with manufacturer's performance curves, water collector, air collector with metal/glass flow channels and air with metal/metal flow channels. This option was deleted in the present program and utilizes the manufacturer's performance curves for the Solaron Series 3000 collector.

The HUMID subroutine analytically models the simultaneous heat and mass transfer occurring within the desiccant bed. Solving the equations which model this process is difficult due to the coupled and non-linear characteristics. The coupled nature can be attributed to the dependency of the sorbent equilibrium capacity and liberation of heat of sorption. The equations are non-linear because equilibrium concentrations of the sorbate and the sorbent are not related linearly, and also because the air and desiccant specific heats and the heat of sorption are not linear.

One method to predict desiccant performance was initially developed by Banks (5) in 1971. Transforming the energy and mass conservation equations to characteristic form, they were found to represent two potential kinematic wave equations. Each of these kinematic wave equations describe the propagation of a change in potential, and depend on sorbate concentration and the temperature. Bank's analysis models the transfer of a single sorbate between a stationary porous sorbent and a single phase fluid mixture. The reader is

encouraged to refer to Ananth and Singer for further discussion.

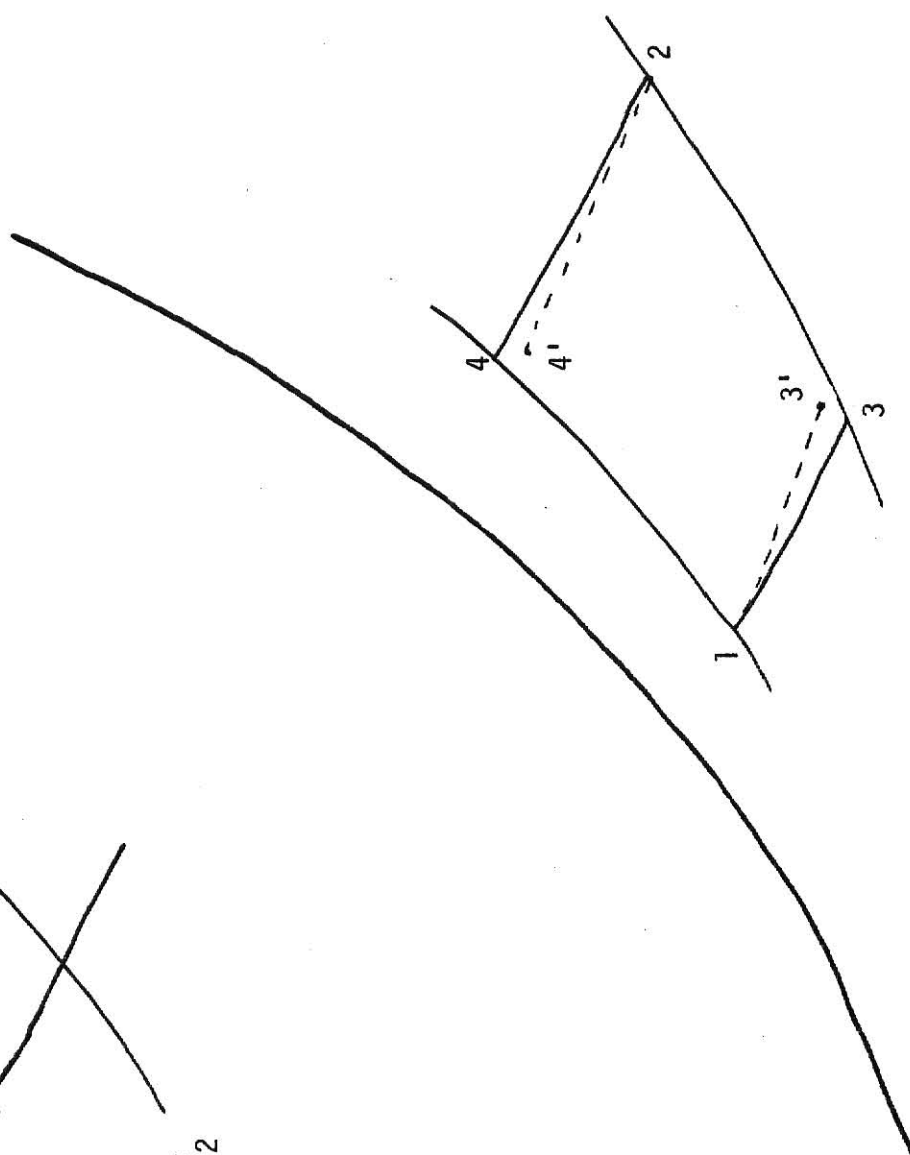
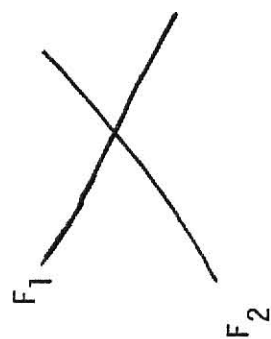
With further study by Close and Banks (11,12) for heat and mass transfer equations between silica gel and an air vapor mixture, they were able to theoretically determine equilibrium outlet conditions of a stationary bed of silica gel, with a step change in inlet air conditions. Their model predicts two wave fronts passing through the bed, the first producing new outlet conditions, which remain constant until the second wave front flows through. After the second wave front passes through the bed, the outlet conditions are identical to the inlet conditions. These wave fronts follow constant F lines on a psychrometric chart as shown in Figure 3. With a rotary dehumidifier, the process air stream changes from state 1 to state 3', and the regeneration air stream proceeds from condition 2 to 4'.

MacLaine-Cross and Banks (30) developed this model further for a rotating bed. This method was computerized by Nelson (33), providing process and regeneration outlet temperatures and humidity ratios. The model does not provide time-temperature-position conditions, but this information is not required for the performance of the dehumidifier for these studies.

Singer based subroutines HUMID, EGV and ALFAV on Nelson's program making minor modifications. Subroutine ALFAV supplies values for α_1 , α_2 , γ_1 and γ_2 for both the silica gel bed and the air vapor mixture. α_1 and α_2 are the slopes of constant F_1 and F_2 lines respectively on a psychrometric chart. γ_1 and γ_2 are the specific capacity ratios between the air vapor mixture and the moist silica gel. Put another way, this subroutine determines the equilibrium properties of the moist silica gel bed and air water vapor mixture, and determines the path the two air streams will follow when the step change occurs. As these paths of constant F_1 and F_2 lines are followed, subroutine EGV (Equilibrium Gama Values) determines the incremental

FIGURE 3

PSYCHROMETRIC PLOT OF SORPTION PROCESS WITH SILICA GEL



HUMIDITY RATIO

TEMPERATURE

values for temperature and humidity ratio. The HUMID subroutine first calculates inlet points 1 and 2, then it estimates the true points 3 and 4 as the intersection of F_i lines through points 1 and 2 having slopes α_i . Many incremental steps along the constant F lines are taken and the values of α_i for each new state are determined. This process is repeated until the intersection of the F_1 and F_2 lines are found, reaching points 3 and 4. Then the rotating effect of the wheel are corrected, yielding points 3' and 4'.

CHAPTER III

WEATHER DATA COMPUTER MODEL

The second portion of the computer program deals with supplying the necessary weather data. How well a solar energy powered system performs depends not only on properties of the system, but also on the weather conditions. The performance of such a system should be estimated on an hour by hour basis, which then required hourly weather data.

The method of estimating hourly insolation utilizes empirical relationships based on measured daily total insolation, the sunset hour angle, the time away from solar noon and the daily clearness index. The daily clearness index is the ratio of terrestrial to extraterrestrial radiation for a particular day and latitude. This method agrees fairly well with weather data. But one disadvantage is that these estimations of insolation do not account for actual hourly variations in weather, such as storms or sudden cloud cover.

Solar radiation together with other meteorological data have been recorded since 1953 at weather stations across the United States. Insolation data were recorded either on a daily or an hourly basis, depending on the station. Most of these horizontal surface radiation data were recorded instantaneously, and were then either graphically or electronically integrated. Through the years, errors in the data were detected due to instrument sensor deterioration, changes in pyrheliometric scales, calibration errors and moves in station location. To identify and correct these errors, the United States Weather Service developed the SOLMET program in 1978. The purpose of SOLMET was to identify errors in individual instruments and location moves, to correct or 'rehabilitate' earlier data recorded, and to combine all available data into a single source. This was achieved by recording all available data, plus rehabilitated data, on computer tapes in SI units. Details of particular

rehabilitation measures taken can be found in Volume I of the SOLMET User's Manual (40).

The SOLMET tape collection comprises an enormous amount of information. In order to use these data in predicting the hourly performance of a solar energy system, one difficulty arises. Since weather on a particular day in one year can vary greatly from weather occurring in another year, the data do not account for annual variations in the weather.

Hall, et al (20) solved this problem with the Typical Meteorological Year tapes. Twenty six SOLMET weather stations (Table 1 and Figure 4) were chosen as Typical Meteorological Year (TMY) sites. Each station recorded 23 years of hourly meteorological and solar radiation data beginning in 1953 and continuing through 1975. The process used statistical methods to select a TMY for a given location, selecting one typical meteorological month for each of the twelve calendar months. Data for one year consists of 12 specific months, i.e. January 1975, February 1955, etc. (Table 2). These months were then catenated to form a typical meteorological year.

The criteria used in selecting a month closest to the composite of all 23 years were dry bulb temperature, dew point temperature, wind velocity and solar radiation on a horizontal surface. Cloud cover and precipitation were not included as criteria for selection of a typical month. Since discontinuities exist between the end of one month in one year and the beginning of the next month in a different year, the pressure, temperatures and wind velocity for the six hourly points on either side of the interface were smoothed.

The TMY data, then, were chosen for this study over another form of weather data for the following reasons:

1. Complete weather data are available on an hourly basis.
2. Daily variations in local cloud cover and yearly fluctuations are

TABLE 1

TYPICAL METEOROLOGICAL YEAR STATIONS

| <u>STATION NAME</u> | <u>STATION NUMBER</u> | <u>TAPE LABEL</u> | <u>DATE OF RECORD</u> | <u>LATITUDE</u> | <u>LONGITUDE</u> |
|---------------------|-----------------------|-------------------|--|--------------------|--------------------|
| Albuquerque, NM | 23050 | 13 | 7/ 1/52 - 12/31/75 | N 35 03 | W 106 37 |
| Apalachicola, FL | 12832 | 4 | 7/ 1/52 - 5 /19/75 5/20/75 - 12/31/75 | N 29 44 N 29 44 | W 84 59 W 85 02 |
| Bismarck, ND | 24011 | 17 | 7/ 1/52 - 12/31/75 | N 46 46 | W 100 45 |
| Boston, MA | 94701 | 24 | 7/ 1/52 - 6/ 5/64 6/ 6/64 - 11/30/68 | N 42 21 N 42 22 | W 71 04 W 71 03 |
| Brownsville, TX | 12919 | 6 | 7/ 1/52 - 12/31/75 | N 25 54 | W 97 26 |
| Cape Hatteras, NC | 93729 | 22 | 7/ 1/52 - 2/28/57 3/ 1/57 - 12/31/75 | N 35 13 N 35 16 | W 75 41 W 75 33 |
| Caribou, ME | 14607 | 10 | 7/ 1/52 - 12/31/75 | N 46 52 | W 68 01 |
| Charleston, SC | 13880 | 7 | 7/ 1/52 - 12/31/75 | N 32 54 | W 80 02 |
| Columbia, MO | 03945 | 3 | 7/ 1/52 - 1/23/70 1/24/70 - 12/31/75 | N 38 58 N 38 49 | W 92 22 W 92 13 |
| Dodge City, KS | 13985 | 9 | 7/ 1/52 - 12/31/75 | N 37 46 | W 99 58 |
| El Paso, TX | 23044 | 12 | 7/ 1/52 - 12/31/75 | N 31 48 | W 106 24 |
| Ely, NV | 23154 | 14 | 12/ 1/51 - 12/31/75 | N 39 17 | W 114 51 |
| Fort Worth, TX | 03927 | 1 | 7/ 1/52 - 4/30/53 5/ 1/53 - 7/31/74 | N 32 49 N 32 50 | W 97 21 W 97 03 |
| Fresno, CA | 93193 | 21 | 7/ 1/52 - 12/31/75 | N 36 46 | W 119 43 |
| Great Falls, MT | 24143 | 18 | 7/ 1/52 - 12/31/75 | N 47 29 | W 111 22 |

TABLE 1, continued

TYPICAL METEOROLOGICAL YEAR STATIONS

| <u>STATION NAME</u> | <u>STATION NUMBER</u> | <u>TAPE LABEL</u> | <u>DATE OF RECORD</u> | <u>LATITUDE</u> | <u>LONGITUDE</u> |
|---------------------|---------------------------|-----------------------|---|--------------------|----------------------|
| Lake Charles, LA | 03937 | 2 | 7/ 1/52 - 10/31/61 11/ 1/61 - 12/31/75 | N 30 13 N 30 07 | W 93 09 W 93 13 |
| Madison, WI | 14837 | 11 | 7/ 1/52 - 12/31/75 | N 43 08 | W 89 20 |
| Medford, OR | 24225 | 19 | 12/ 1/51 - 12/31/75 | N 42 22 | W 122 52 |
| Miami, FL | 12839 | 5 | 7/ 1/52 - 12/31/75 | N 25 48 | W 80 16 |
| Nashville, TN | 13897 | 8 | 7/ 1/52 - 12/31/75 | N 36 07 | W 86 41 |
| New York, NY | 94728 | 25 | 7/ 1/52 - 12/31/75 | N 40 47 | W 73 58 |
| Omaha, NE | 94918 | 26 | 6/ 1/57 - 12/31/75 | N 41 22 | W 96 01 |
| Phoenix, AZ | 23183 | 15 | 7/ 1/52 - 12/31/75 | N 33 26 | W 112 01 |
| Santa Maria, CA | 23273 | 16 | 7/ 1/52 - 10/31/54 11/ 1/54 - 3/31/69 | N 34 56 N 34 54 | W 120 25 W 120 27 |
| Seattle-Tacoma, WA | 24233 | 20 | 12/ 1/51 - 12/31/75 | N 47 27 | W 122 18 |
| Sterling, VA | 93734 | 23 | 8/ 1/53 - 12/31/60 11/ 1/60 - 12/31/75 | N 38 50 N 38 59 | W 76 57 W 77 28 |

FIGURE 4

SOLAR RADIATION DATA REHABILITATION STATIONS

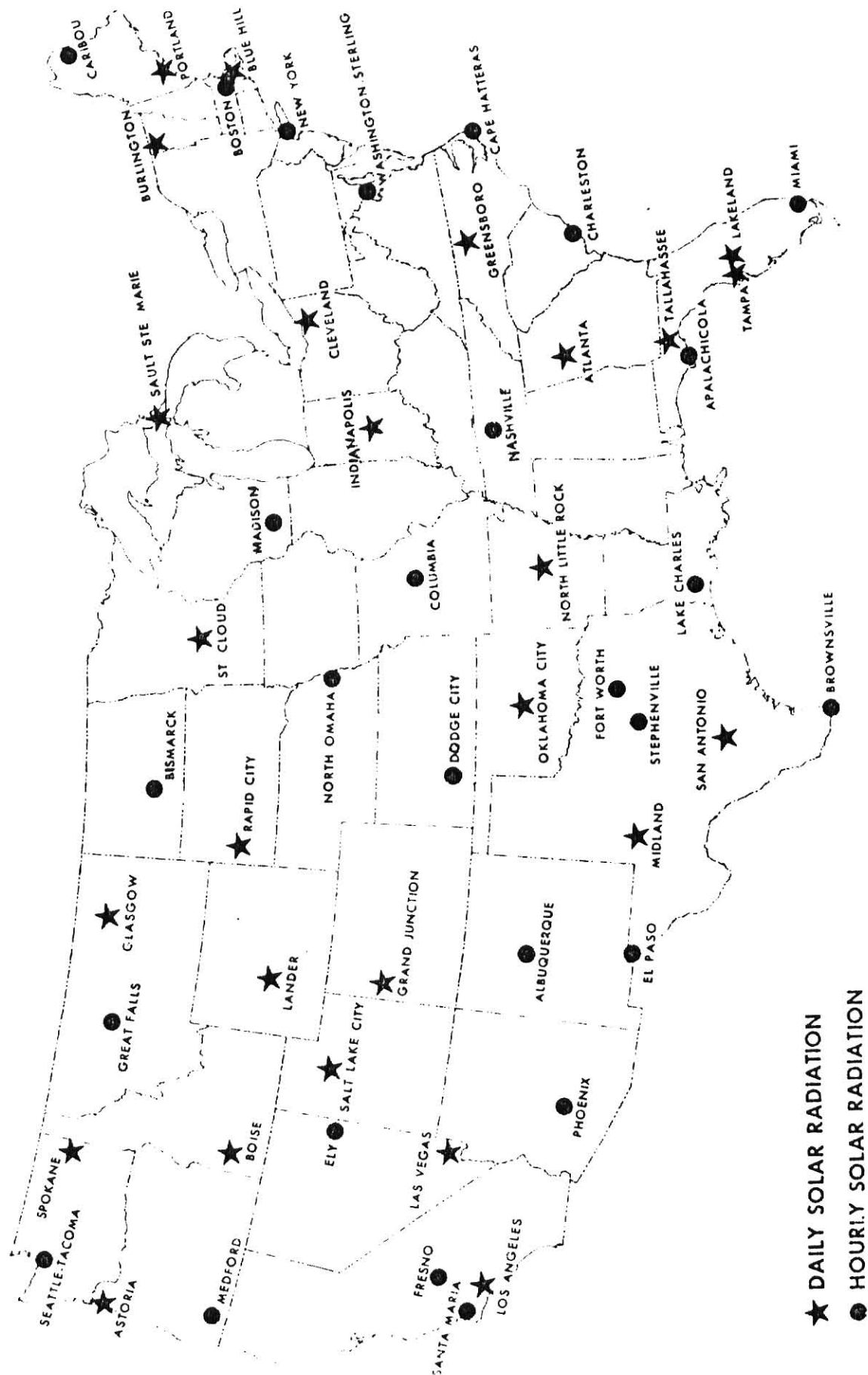


TABLE 2

THE YEAR OF THE
TYPICAL METEOROLOGICAL MONTHS COMPRISING
TYPICAL METEOROLOGICAL YEAR FOR 26 STATIONS

| <u>Location</u> | <u>Jan</u> | <u>Feb</u> | <u>Mar</u> | <u>Apr</u> | <u>May</u> | <u>Jun</u> | <u>Jul</u> | <u>Aug</u> | <u>Sep</u> | <u>Oct</u> | <u>Nov</u> | <u>Dec</u> |
|-----------------|------------|------------|------------|------------|------------|------------|------------|------------|------------|------------|------------|------------|
| Albuquerque | 58 | 53 | 65 | 66 | 64 | 69 | 57 | 54 | 67 | 65 | 59 | 54 |
| Apalachicola | 60 | 60 | 64 | 63 | 67 | 63 | 58 | 55 | 66 | 55 | 59 | 65 |
| Bismarck | 60 | 71 | 57 | 54 | 57 | 75 | 69 | 55 | 57 | 68 | 67 | 67 |
| Boston | 66 | 53 | 61 | 60 | 62 | 65 | 53 | 56 | 62 | 57 | 61 | 61 |
| Brownsville | 64 | 61 | 56 | 65 | 71 | 64 | 61 | 66 | 68 | 53 | 68 | 75 |
| Cape Hatteras | 65 | 55 | 56 | 69 | 54 | 69 | 68 | 71 | 64 | 66 | 74 | 66 |
| Caribou | 59 | 70 | 70 | 69 | 71 | 53 | 74 | 53 | 72 | 73 | 74 | 60 |
| Charleston | 60 | 60 | 75 | 59 | 73 | 61 | 53 | 67 | 60 | 65 | 71 | 75 |
| Columbia | 65 | 55 | 54 | 70 | 58 | 64 | 62 | 63 | 67 | 74 | 71 | 64 |
| Dodge City | 60 | 73 | 54 | 62 | 55 | 63 | 61 | 60 | 66 | 54 | 71 | 67 |
| El Paso | 74 | 67 | 75 | 74 | 54 | 61 | 71 | 61 | 71 | 67 | 71 | 56 |
| Ely | 74 | 71 | 71 | 71 | 56 | 75 | 58 | 73 | 66 | 66 | 63 | 65 |
| Fort Worth | 72 | 61 | 62 | 66 | 66 | 59 | 65 | 55 | 57 | 68 | 71 | 62 |
| Fresno | 64 | 75 | 68 | 53 | 68 | 62 | 54 | 73 | 68 | 66 | 74 | 68 |
| Great Falls | 68 | 65 | 71 | 63 | 70 | 59 | 54 | 62 | 73 | 71 | 71 | 65 |
| Lake Charles | 67 | 72 | 56 | 74 | 64 | 59 | 70 | 55 | 64 | 72 | 67 | 65 |
| Madison | 65 | 60 | 72 | 64 | 53 | 57 | 73 | 63 | 58 | 74 | 65 | 54 |
| Medford | 66 | 62 | 53 | 69 | 64 | 59 | 69 | 63 | 62 | 60 | 62 | 61 |
| Miami | 62 | 74 | 67 | 59 | 64 | 53 | 57 | 63 | 57 | 65 | 61 | 68 |
| Nashville | 54 | 59 | 64 | 74 | 63 | 68 | 75 | 73 | 58 | 69 | 66 | 66 |
| New York | 58 | 59 | 59 | 74 | 74 | 61 | 60 | 72 | 58 | 56 | 71 | 67 |
| Omaha | 72 | 59 | 59 | 63 | 58 | 72 | 61 | 60 | 66 | 68 | 74 | 62 |
| Phoenix | 68 | 75 | 63 | 57 | 68 | 56 | 74 | 72 | 72 | 68 | 59 | 66 |
| Santa Maria | 63 | 59 | 57 | 57 | 56 | 64 | 53 | 62 | 61 | 62 | 53 | 62 |
| Seattle | 75 | 71 | 62 | 72 | 61 | 59 | 62 | 70 | 62 | 66 | 68 | 56 |
| Sterling | 65 | 70 | 67 | 67 | 56 | 75 | 73 | 65 | 73 | 73 | 73 | 61 |

included. The weather data are most representative of weather one could anticipate at any location.

3. The data include 26 locations within the United States. This gives a good choice of locations for a variety of dehumidification applications.
4. The data are stored on a magnetic tape and easily incorporated into a computer program.

TMY WEATHER TAPE

The tape available through the KSU Meteorological Service is 1600 bpi density, 9 track, EBCDIC mode and can be executed using either Fortran or Watfiv compilers. Each block of the tape contains the data for one day, is 3168 bytes long and consists of 24 logical records. Each logical record is 132 bytes long and contains the data for one hour. The tape can be read with or without a format; see Figure 5 for the JCL necessary to read the tape unformatted, and Figures 6a and 6b for a sample of an unformatted block.

However, to use the data in calculations requires a format. The tape density, block length, record length, track and station location (Table 1) must be specified in the JCL of the program. The JCL specifies which tape the computer should read and the physical characteristics of the tape. The fortran deck controls when and in what format the tape is read. See Figures 7 and 8 for the required Watfiv and Fortran JCL.

The tape positions can be read as integers (I5 for a 5 character integer), as real numbers (F5.2 or 2PF5.0 for a real number with two decimal places) or can be skipped over (5X to skip five positions). Refer to Table 3 for the contents of the TMY tape. The first five positions on the tape contain the number assigned to the station. This acts as a check to assure that the correct

FIGURE 5

LISTING OF PROGRAM FOR
UNFORMATTED TMY WEATHER DATA

```

/*TAPE9
// EXEC DSLIST,PARM=(,,3)
//SYSIN DD DSN=TMY,UNIT=TAPE1600,VOL=SER=9TMY03,LABEL=26

```

Where:

PARM=(,,3) - refers to the number of blocks read,
in this case, three

UNIT=TAPE1600 - the 1600 specifies the tape bpi density

VOL=SER=9TMY03 - refers to the name of the tape

LABEL=26 - is the tape label for the station, 26 is
for Omaha, Nebraska (Table 1)

FIGURE 6a

TMY UNFORMATTED DATA FOR
Jan. 3, 11:00 at Omaha Nebraska

| RADIATION | | | | | | | | | | | | | |
|----------------|----------------|----------------|---------------|------------|------------|------------------|---------|----------|-----------|-----------|----------|------------------|-----------------------|
| Station Number | Year | Month | Day | Solar time | Clock time | Extraterrestrial | Direct | Diffuse | Net | Tilt | Observed | Engin. corrected | Stand. year corrected |
| 9491872 | 1 | 3 | 1100 | 1128 | 1946 | 70002 | 99999 | 99999 | 99999 | 569 | 539 | 483 | 53999999 |
| Min. of Sun | Time of Obser. | Ceiling Height | Sky Condition | Visibility | Weather | Sea level | Station | Dry bulb | Dew point | Direction | Velocity | Clouds | Snow cover |
| 9911 | 99999 | 999999 | 99999 | 9999999999 | 10265 | 9892 | -99 | -147 | 27 | 75 | 99999 | 0 | |

FIGURE 7

WATFIV JCL FOR TMY WEATHER DATA TAPE

```

/*ROUTE PRINT VM_--
/*TAPE9
/*REGION 400K
// EXEC WATFIV,PARM='SIZE=(1000K,8K)'
//SYSIN DD *
$JOB AFA,TIME=(,30),PAGES=30
C
C
C
C
C
C
C
C
C
C
FCRTRAN DECK
C
$ENTRY
C
C
C
C
C
C
DATA SOURCE
C
C
C
C
C
C
//GO.FT03F001 DD DSN=TMY,UNIT=TAPE1600,VOL=SER=9TMY03,LABEL=(3,,,IN),
// DCB=(RECFM=FB,LRECL=132,BLKSIZE=3168)

```


TABLE 3

CONTENTS OF TMY TAPE

| <u>TAPE POSITION</u> | <u>FORMAT TO READ OFF TAPE</u> | <u>ELEMENT</u> |
|--------------------------|--|---|
| 1-5 | I5 | Station Number |
| 6-7 | I2 | Year of observation- last two digits only |
| 8-9 | I2 | Month of observation |
| 10-11 | I2 | Day of observation |
| 12-15 | I4 | Solar time- end of the hour in hours and minutes |
| 16-19 | I4 | Clock time- local standard time in hours and minutes |
| | | RADIATION - in kJ/m^2 |
| 20-23 | I4/F4.0 | Extraterrestrial |
| 24-28 | I5/Ix,I4 | Direct- with data code indicator |
| 29-33 | I5 | Diffuse |
| 34-38 | I5 | Net |
| 39-43 | I5 | Tilt |
| 44-48 | I5/F5.0 | Observed |
| 49-53 | I5/F5.0 | Engineering corrected |
| 54-58 | I5/F5.0 | Standard year corrected |
| 59-68 | I10 | Additional |
| 69-70 | I2 | Minutes of Sunshine |
| | | METEOROLOGICAL DATA |
| 71-72 | I2 | Time of observation |
| 73-76 | I4 | Ceiling height |
| 77-81 | I5 | Sky condition- coded in ascending layers: 0 = clear or less than 10% cover 1 = thin scattered (10-50% cover) 2 = opaque scattered (10-50% cover) 3 = thin broken (60-90% cover) 4 = opaque broken (60-90% cover) 5 = thin overcast (100% cover) 6 = opaque overcast (100% cover) 7 = obscuration 8 = partial obscuration |

TABLE 3, continued

CONTENTS OF TMY TAPE

| <u>TAPE POSITION</u> | <u>FORMAT TO READ OFF TAPE</u> | <u>ELEMENT</u> |
|--------------------------|--|---|
| 82-85 | I4 | Visibility- prevailing horizontal visibility |
| 86-93 | I8 | Weather |
| 86 | | Occurrence of Thunderstorm, tornado or squall 0 = none 1 = Thunderstorm 2 = Heavy Thunderstorm 3 = Tornado reported 4 = Squall |
| 87 | | Occurrence of rain, rain showers or freezing rain ranging from 0 - 8 with 0 = none |
| 88 | | Occurrence of drizzle, freezing drizzle ranging from 0 - 8 with 0 = none |
| 89 | | Occurrence of snow, snow pellets or ice crystals ranging from 0 - 8 with 0 = none |
| 90 | | Occurrence of snow showers and snow grains, ranging from 0 - 8 with 0 = none |
| 91 | | Occurrence of sleet, sleet showers or hail, ranging from 0 - 8 with 0 = none |
| 92 | | Occurrence of fog, blowing dust or blowing sand 0 = none 1 = fog 2 = ice fog 3 = ground fog 4 = blowing dust 5 = blowing sand |
| 93 | | Occurrence of smoke, haze, dust, blowing snow, blowing spray |
| 94-98 | F5.2/2PF5.0 | Pressure - sea level in kPa, atmospheric |
| 99-103 | F5.2/2PF5.0 | Pressure - at the station, atmospheric |
| 104-107 | F4.1/PF4.0 | Dry bulb temperature |
| 108-111 | F4.1/PF4.0 | Dew point temperature |
| 112-114 | I3 | Wind direction, measured clockwise from the North |
| 115-118 | I4 | Wind speed |
| 119-122 | I4 | Clouds, coded |
| 123 | I1 | Snow cover indicator |
| 124-132 | | Positions left blank |

portion of the tape is read. Refer to Table 1 for the stations and their corresponding numbers.

The year of observation lists only the last two digits, so that a 66 is actually 1966. The solar time refers to the end of the hour in hours and minutes. Clock time is the local standard time in hours and minutes corresponding to the solar time.

All radiation values are in kilojoules per meter squared. The extra-terrestrial radiation is the radiation received at the top of the atmosphere during the solar hour based on a solar constant $1377 \text{ J/m}^2\text{-s}$. Any elements encoded with a '9' beginning with the extraterrestrial radiation to the end of the tape indicate missing or unknown data.

For the following radiation data, the first position contains a data code indicator and the next four positions contain the recorded insolation. The data code indicator describes whether the value is observed, or if it was estimated from another model. The data code indicator is not always included.

The direct radiation is that portion of insolation received directly from the sun at the pyrheliometer. Diffuse radiation is the amount of radiant energy received indirectly from reflection and scattering. The net radiation refers to the difference of incoming and outgoing radiation. A constant of 5000 was added to all net radiation data. Tilt radiation indicates the global radiation (total of direct and diffuse) received on a tilted surface. Observed radiation is that observed value on a horizontal surface. Engineering corrected data is the observed value corrected for known scale changes, recorder and sensor calibration changes, station moves, etc. Standard year corrected radiation is the observed value adjusted by a regression estimate based on cloud, sky condition and sunshine data. This models the expected

clear sky irradiance received on a horizontal surface. The Additional radiation allows tape positions for any supplemental radiation measurements that may have been taken at any particular station.

The time of observation refers to the Local Standard hour which comes closest to the midpoint of the solar hour. Ceiling height in dekameters is defined as 0.6 or greater sky cover. A recording of 0000-3000 equals 0 to 30,000 meters, 7777 is coded as unlimited or clear, and 8888 refers to an unknown height of cirroform ceiling. Sky condition codes four layers of cloud cover in ascending order. Visibility indicates the horizontal visibility in hectometers ($hm = m \times 100$). If encoded with 8888, the visibility is unlimited.

The weather portion of the tape indicates the type and quantity of precipitation, the occurrence of thunderstorms, tornados, fog, etc. Refer to Table 3 for details of the code.

The pressures are measured in kilopascals. The range from 08000 to 10999 indicate pressures of 80 to 109.99 kPa. Temperatures range from -70 to 60 °C, and are recorded as -700 to 0600.

The wind direction is the azimuth angle measured in degrees. The wind speed is in tenths, so that 1500 equals 150.0 m/s. The code for clouds describes the type and height of cloud cover. A more detailed description can be found in SOLMET User's Guide (40). The snow cover indicator records a '0' for no snow or just a trace of snow on the ground, and a '1' for more than a trace of snow. The last nine positions of the record are empty.

THE COMPUTER PROGRAM

The program must first locate the desired date. The segment of tape for each station begins at the first hour of the first day of the year and must

be scanned until the desired day is found. Since the shortest readable unit is one block, the hour immediately preceeding the desired date must be found. The following segment will then be the record length of the next block corresponding to the first hour of the desired day. This method must be followed because of the two separate Read statements in the program (Appendix B). Data is assigned to variables in the form of arrays for later computations in subroutine SYSTEM. The Read statement that lies inside two nested Do loops assigns the data to the arrays. One Do loop controls the hour and the other controls the day. The Read statement preceeding the Do loops locates the desired day. If the first record for the desired day is used to locate the beginning of the study, then the weather data within that record cannot be assigned to an array.

After locating the correct date, the program then proceeds to read one entire record length which contains weather data for one hour. For the first hour of the day, the number of day of the year is calculated, which is required later in subroutine SYSTEM. For every hour of the day, the program uses weather data to calculate the percentage of direct and diffuse radiation following equations from Duffie and Beckman (15), and the humidity ratio following ASHRAE (4) equations. Since the TMY tape contains several values for radiation, one is specified as the horizontal radiation desired for later calculations.

At this point, the computer should have all of the weather data necessary for one day and calls subroutine SYSTEM for that day. For the general case, subroutine SYSTEM can be any computer program, written in Fortran Watfiv describing the daily hour by hour performance of a solar energy powered system. Subroutine SYSTEM is called for every day, internally computes and prints the hourly performance and returns to the main program. The moisture

evaporated each hour is calculated next. Chapter four contains the discussion for the method used to calculate the evaporated moisture. The system operating totals are tabulated for the moisture evaporated, solar heat gain and the auxiliary heat supplied. It then proceeds to read the weather data for the next day and repeats this procedure. The Economic Analysis follows after completing all calculations for all days included.

The Weather Data program need not be accompanied by a SYSTEM subroutine. It may be of value to the user to scan the weather of a particular month for the location in mind, before choosing acceptable system operating dates. Because of this, the Weather Data program was developed to read the TMY tape and print out hourly insolation, atmospheric pressures, dry bulb temperature, dew point temperature, humidity ratio, wind direction and speed and the amount of snow cover. This program is listed in Appendix C, accompanied by two days of output.

In order to use this program, the user must specify the beginning and the ending month, days and hours in the data cards, the tape label corresponding to the desired station and the tape specifications in the JCL, the station name in Format statement 10, and correctly dimension the parameter arrays. The Dimension and Integer statements dimension parameter arrays for the number of days and hours, i.e. `DBT(30,24)` dimensions the dry bulb temperature for thirty days with twenty four hours per day. These arrays should be declared to allow adequate storage for the number of days and hours desired. Two limitations exist with this program. First, the beginning and ending month must be identical. The program, as is, does not allow for a study beginning, for example on June 30 and ending July 3. Because of this, and the necessity to locate the hour preceeding the first day of the study, the second limitation is that the first day of the month cannot be read. These

limitations may be annoying to the user, but can be dealt with.

CHAPTER IV

ECONOMIC ANALYSIS COMPUTER MODEL

Once the performance of a solar energy system with a particular length of operation and location has been estimated, the next step is to determine the economic feasibility. Unit costs may be an indicator of this feasibility, but to adequately evaluate any system, it must be incorporated with a system in operation and related to current energy costs. This can be done by comparing the life cycle cost of a solar energy powered system to that of a conventional system.

The P_1 , P_2 method outlined by Duffie and Beckman (15) relates present worth factors for the life cycle costs to the first year costs in one ratio. One can then determine the life cycle cost, which is merely the present worth of all costs. This method applied to both a conventional and a solar energy powered system provides a foundation for comparing the costs of these systems.

The cost of either system includes two parts, the cost of fuel and the cost of equipment. P_1 is the ratio of life cycle fuel costs to the first year fuel cost and yields the present worth of an inflating series. P_1 multiplied by the present day cost of fuel and the load, determines the total fuel cost. P_2 is the ratio of life cycle expenditures due to any additional capital investment, divided by the initial investment. This yields the present worth of equipment mortgage payments, depreciation tax deduction, etc. The life cycle equipment cost is calculated by multiplying P_2 times the cost of equipment.

For the conventional system, the Life Cycle Cost can be written:

$$LCC_c = P_{2c} C_{Ec} + P_{1c} C_{fc} L \quad (4.1)$$

A solar energy system must consider two additional points. First, part

of the equipment has a fixed size, hence also fixed cost, and part varies with the size of the system. Second, the fraction of auxiliary fuel used must be taken into consideration. The Life Cycle Cost for a solar energy system with a conventional system as a backup can be written as

$$LCC_s = P_{2s}(C_A A + C_{Es}) + P_{2c}C_{Ec} + P_{1s}C_f L (1 - F) \quad (4.2)$$

The present worth factor is given by:

$$PWF(N, i, d) = \begin{cases} \frac{1}{(d - i)} \left[1 - \left(\frac{1 + i}{1 + d} \right)^N \right] & i \neq d \\ \frac{N}{(1 + i)} & i = d \end{cases} \quad (4.3)$$

Any cost that is proportional to the first year fuel cost can be included in P_1 . For this study,

$$P_1 = (1 - C\bar{t}) PWF(N_e, i_f, d) \quad (4.4)$$

where \bar{t} is the effective tax rate given as

$$\bar{t} = \frac{\text{Federal tax}}{\text{Rate}} + \frac{\text{State tax}}{\text{Rate}} - \left[\frac{\text{Federal tax} \times \text{State tax}}{\text{Rate} \times \text{Rate}} \right] \quad (4.5)$$

Likewise, any capital cost that is proportional to the initial cost can be included in P_2 . For the general case:

$$\begin{aligned} P_2 = & D + (1-D) \left[\frac{PWF(N_{\min}, \emptyset, d)}{PWF(N_{\min}, \emptyset, m)} \right] + (1 - C\bar{t}) M_s PWF(N_e, i, d) \\ & - (1-D)\bar{t} \left[PWF(N_{\min}, m, d) \times \left(m - \frac{1}{PWF(N_L, \emptyset, m)} \right) + \frac{PWF(N_{\min}, \emptyset, d)}{PWF(N_L, \emptyset, m)} \right] \\ & + t(1-\bar{t}) V PWF(N_e, i, d) - DEP - \frac{R_v}{(1+d)^{N_e}} \end{aligned} \quad (4.6)$$

This can be more understandably written as:

$$P_2 = \text{Downpayment} + \text{Cost of the mortgage principal and interest} + \text{Miscellaneous costs (maintenance, insurance, parasitic power, etc.)}$$

$$- \text{Income Tax Deductions from the interest} + \text{Net Property tax costs} - \text{Depreciation tax deduction} - \text{Resale value}$$

For straight line depreciation,

$$DEP = \frac{C\bar{t}}{N_d} \text{PWF}(N_{\min}, 0, d) \quad (4.7)$$

The terms used for double declining balance and sum of digits depreciation are given by Duffie and Beckman. It should be noted that the double declining balance method includes the resale value. For accelerated cost recovery, a system depreciating in five years follows this schedule:

| <u>Year</u> | <u>Percent of Initial Value</u> |
|-------------|---------------------------------|
| 1 | 20 |
| 2 | 32 |
| 3 | 24 |
| 4 | 16 |
| 5 | 8 |

The percentage for each year, multiplied by the present worth of that specific year, yields the total present worth. So that the term can be expressed as

$$DEP = C\bar{t} \left[0.20 \times \frac{1}{(1+d)^1} + 0.32 \times \frac{1}{(1+d)^2} + 0.24 \times \frac{1}{(1+d)^3} + 0.16 \times \frac{1}{(1+d)^4} + 0.08 \times \frac{1}{(1+d)^5} \right] \quad (4.8)$$

All terms in P_1 and P_2 include only the present worth factors and ratios in proportion to the initial investment. Any applicable term may be added to or deleted from P_1 or P_2 . They are dependant only on the economic parameters

available and do not vary with the size of the system. The conventional system uses the general forms of P_1 and P_2 , given previously in equations 4.4 and 4.6. The solar energy system includes an additional term in P_2 for tax credits, which are eligible to such a system, and uses the general form for P_1 .

The life cycle cost of a solar energy system varies with the performance of the system and with the fraction of energy supplied by auxiliary methods. The required auxiliary energy in turn depends on the collector area or the load. Optimizing the life cycle cost can be done by one of two methods. Referring to equation 4.2, either the load or the collector size must be fixed. With a fixed load, the collector area can be increased or decreased, which respectively decreases or increases the fraction of auxiliary energy required, $(1-F)$. With a fixed collector area, the load must be altered. A change in load yields a directly proportional change in the fraction of auxiliary energy required.

The latter method was chosen for this study to facilitate the computer program. A change in collector area changes the performance of the system. Because of this, subroutine SYSTEM would be called every time the collector area changed. This would require much computer time and expense. In contrast, varying the load can be accomplished entirely within the economic analysis portion of the main program.

For a drying operation, the energy required to evaporate moisture from the commodity determines the load. The amount of moisture removed by the air from the commodity is a function of the commodity initial moisture content and drying properties, and of the process air dry bulb and wet bulb temperatures. Air will come to equilibrium with the moisture content of a hygroscopic commodity described by an equilibrium moisture curve for that commodity at a specific temperature. If the air relative humidity is lower than the

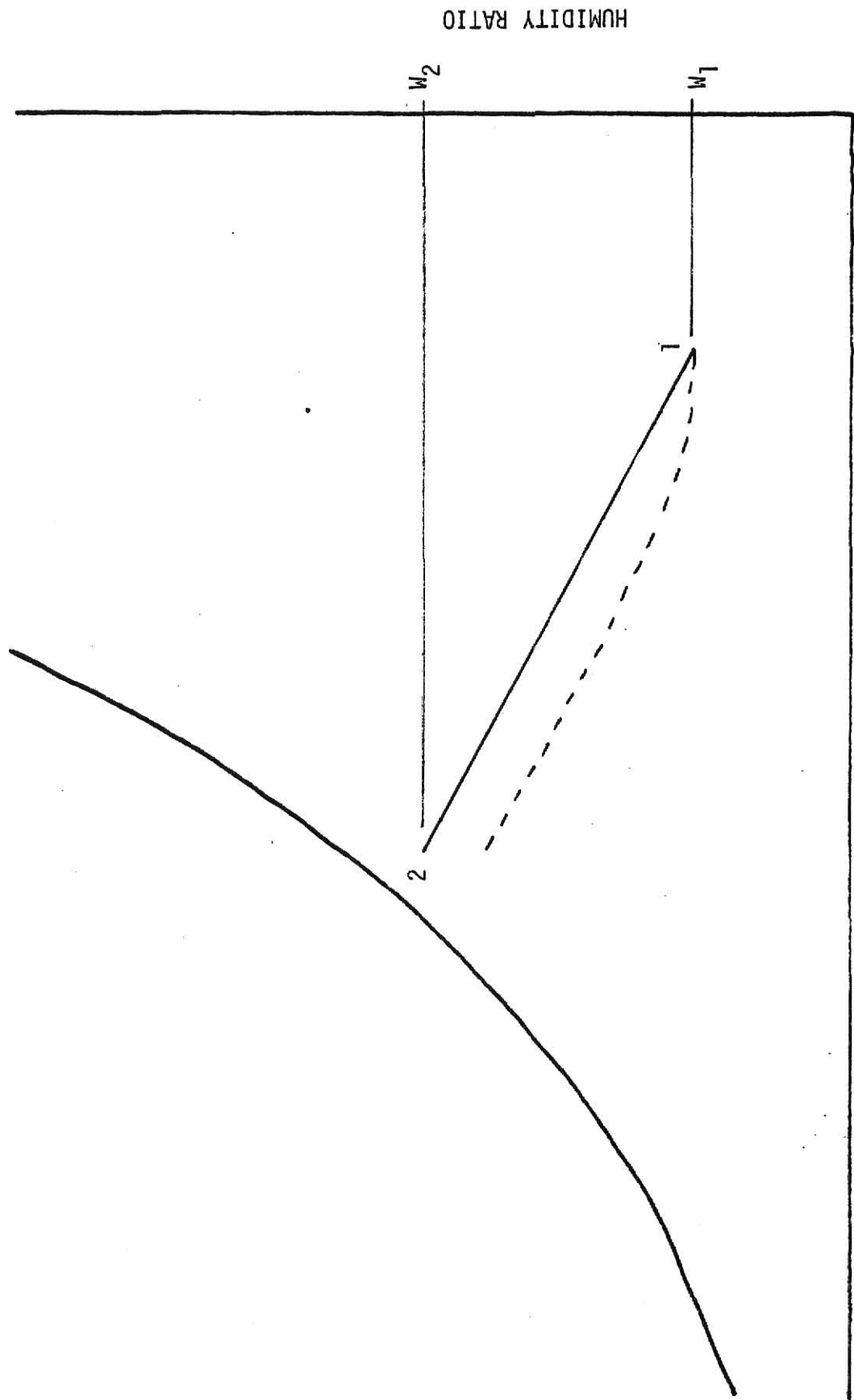
equilibrium moisture content of the commodity, then the air will gain moisture from the commodity. When dry air is passed through a moist commodity, the relative humidity of air exiting the dryer will be in equilibrium with the initial moisture content of the commodity. The relative humidity can then be determined with an equation which models the equilibrium moisture curves. Such equations have been developed by Henderson (21) and by Chung and Pfof (9). Henderson's equation gives valid results for relative humidities ranging from 4 to 70% for a variety of commodities, see Table 4. The equation developed by Chung and Pfof is applicable for 7 to 90% relative humidity for shelled corn, corn products and wheat. Henderson's equation was chosen due to the wider range of commodities and the equation describes the equilibrium moisture curves adequately for these initial studies. This equation is given by:

$$1 - R.H. = e^{(-c T M_E^n)} \quad (4.9)$$

where the temperature is the dry bulb temperature in degrees Rankine, and c and n are drying constants for a particular commodity.

Air entering a dryer at state 1 in Figure 9 will ideally follow a constant wet bulb temperature line and exit at state 2 where the relative humidity is in equilibrium with the commodity initial moisture content. The moisture evaporated will be the difference in humidity ratios of states 1 and 2 multiplied by the airflow rate. However, two assumptions are made, of which the reader should be aware. The air will follow a constant wet bulb temperature line assuming the drying process is adiabatic, and assuming the temperature of the commodity is already the same as the wet bulb temperature of the air entering the dryer. More likely, the commodity will gain some heat, thereby lowering the air wet bulb temperature and the air will follow the dotted line in Figure 9. The air will exit the dryer with a lower relative

FIGURE 9
PSYCHROMETRIC PLOT OF DRYING PROCESS



TEMPERATURE

TABLE 4

EQUILIBRIUM CONSTANTS FOR
HENDERSON'S EQUATION

| <u>MATERIAL</u> | <u>c x10⁻⁵</u> | <u>n</u> |
|-------------------|---------------------------|----------|
| Corn, shelled | 1.10 | 1.90 |
| Cotton | 4.91 | 1.70 |
| Eggs, spray-dried | 2.95 | 2.00 |
| Flaxseed | 0.689 | 2.02 |
| Natural clay | 7.53 | 1.72 |
| Peaches, dried | 41.1 | 0.564 |
| Prunes, dried | 12.5 | 0.865 |
| Raisins | 7.13 | 1.02 |
| Sorghum | 0.340 | 2.31 |
| Soybeans | 3.20 | 1.52 |
| Wheat | 0.0559 | 3.03 |
| Wood | 5.34 | 1.41 |

humidity since the temperature is lower (Equation 4.9), hence, a lower humidity ratio. This will cause a lower efficiency than that predicted by the equations.

Another problem arises in determining the wet bulb temperature of the air entering the dryer. The condition of the air exiting the dehumidifier and entering the dryer are the same. Because of hourly differences in ambient temperature, humidity ratio and insolation, the dehumidifier air exits the dehumidifier at various wet bulb temperatures. Figure 10 shows, that for October 21 in Omaha, Nebraska, the wet bulb temperature varies from 11.2 to 20 °C. With an air flow rate of 0.02836 kg/sec, this is a difference of 0.3675 kilograms of water for every hour, see Table 5. The wet bulb temperature can be determined analytically by numerical methods or by trial and error which can be incorporated into a computer model, or with use of a psychrometric chart, which is not easily incorporated into a computer model.

As the reader can see, much work is needed to develop useable methods predicting the process of commodity drying. Additional study should incorporate transient energy and mass balances on the commodity and the drying air with changing inlet air conditions. With given initial commodity conditions, and varying inlet air conditions, the method developed should accurately predict transient outlet air and commodity conditions and be incorporated into a computer model.

For the present study, it is assumed that the drying process follows a constant wet bulb temperature line and an alternative to estimating the wet bulb temperature was chosen. This alternative method chooses a set wet bulb temperature and either cools or heats the air to that condition before it enters the dryer. Figure 11 shows this process on a psychrometric chart for several hours of operation. Under practical considerations, the air will be heated and not cooled. Once the wet bulb temperature is set, the dry bulb

FIGURE 10

PSYCHROMETRIC PLOT OF
DEHUMIDIFIER OUTLET AIR CONDITIONS
OCTOBER 21 Omaha, Nebraska

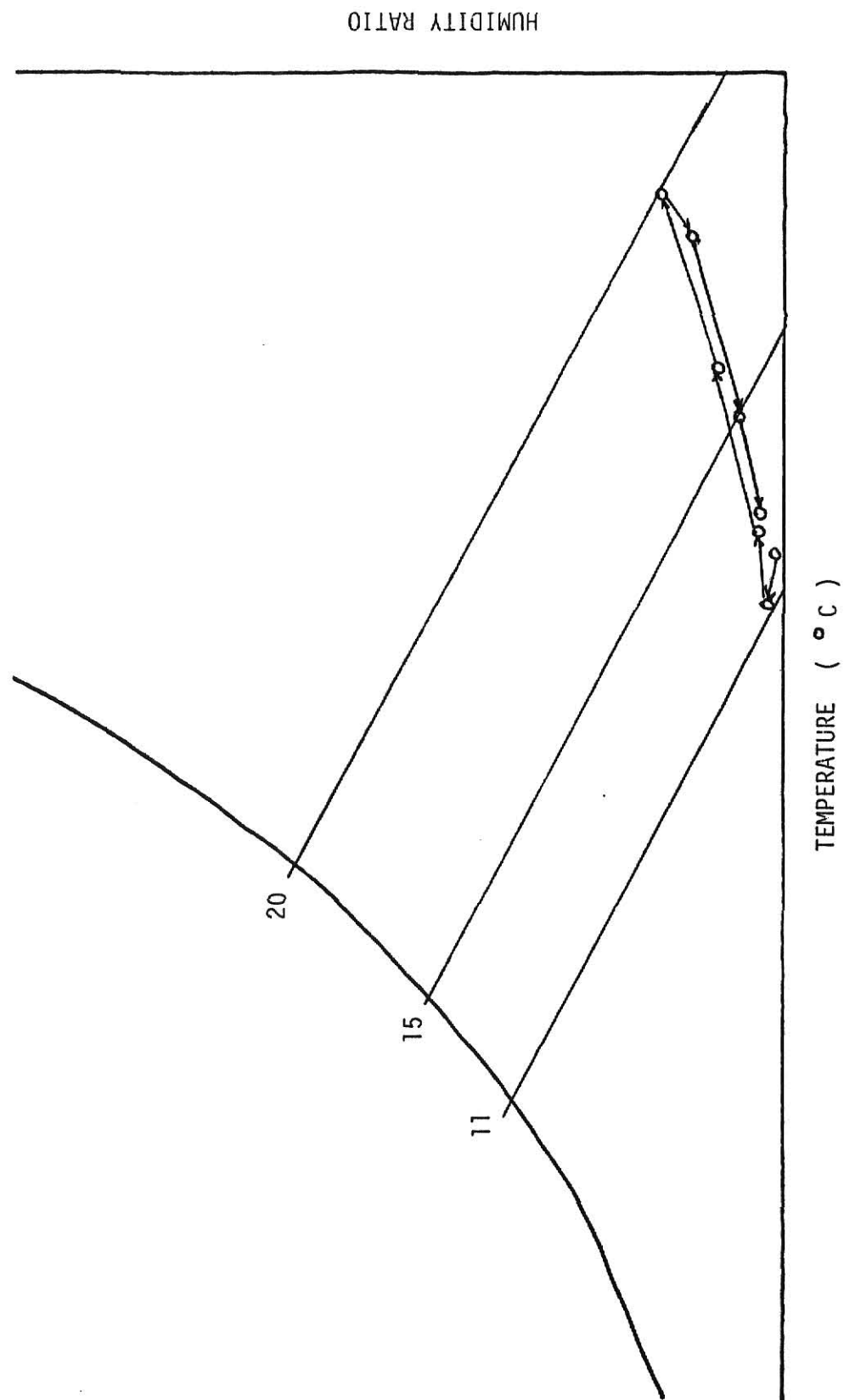


TABLE 5

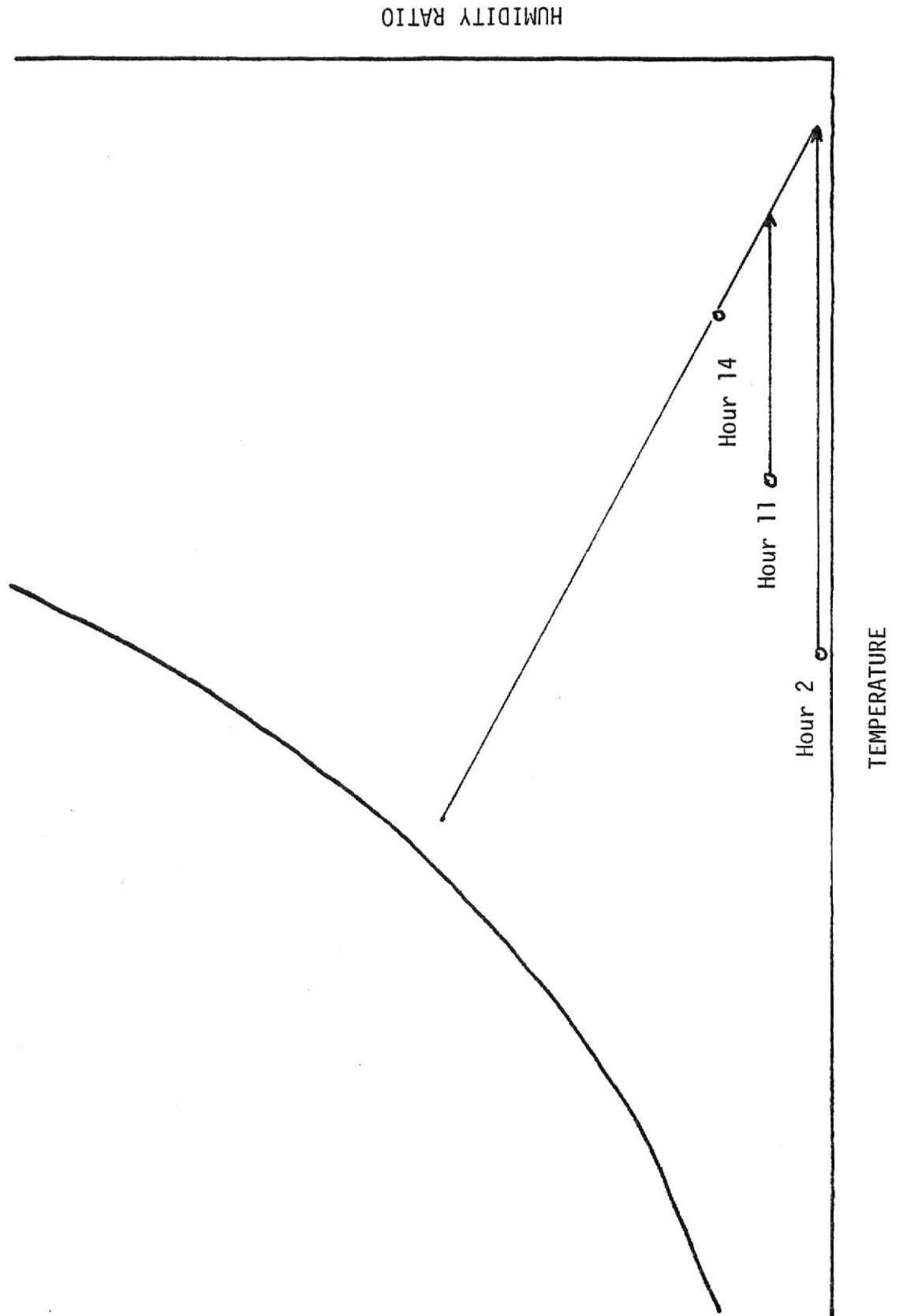
October, 21 Omaha, Nebraska
Moisture Evaporated With Various
Constant Wet Bulb Temperatures

| Hour | Dehumidifier Outlet Temperature | Dehumidifier Outlet Humidity | Wet Bulb Temperature | Sat. Humidity Ratio | Change in Humidity Ratio | Moisture Evaporated * |
|------|------------------------------------|---------------------------------|-------------------------|---------------------------|--------------------------------|--------------------------|
| 2 | 32.5 | .0005 | 12.0 | .0088 | .0083 | 0.8473 |
| 5 | 30.4 | .0009 | 11.2 | .0083 | .0074 | 0.7555 |
| 8 | 33.1 | .0009 | 12.8 | .0093 | .0084 | 0.8576 |
| 11 | 39.8 | .0020 | 16.2 | .0116 | .0096 | 0.9801 |
| 14 | 46.4 | .0038 | 20.0 | .0148 | .0110 | 1.123 |
| 17 | 44.6 | .0029 | 18.6 | .0136 | .0107 | 1.092 |
| 20 | 37.7 | .0017 | 15.0 | .0106 | .0089 | 0.9086 |
| 23 | 33.8 | .0009 | 13.0 | .0093 | .0084 | 0.8576 |

Temperatures in °C

* Moisture Evaporated in Kilogram per Hour,
based on an air flowrate of 0.02836 kg/s

FIGURE 11
PSYCHROMETRIC PLOT OF DRYING MODEL
FOR COMPUTER PROGRAM



temperature for each hour can be calculated for use in equation 4.9 and the moisture evaporated during one hour can be determined.

Then, multiplying the system grand total of moisture evaporated by the latent heat of vaporization determines the load handled by the silica gel dehumidifier. The supplementation of an additional heater increases the load. The fraction of energy supplied by solar, F , equals the solar collector heat gain divided by the total load supplied by the silica gel dehumidifier and the additional heater. The quantity, $1 - F$, is then the amount of auxiliary heat required, which when multiplied by the load, the cost of fuel and P_{1s} , determines the life cycle cost of the fuel (equation 4.2).

This method of determining the load is developed with one configuration of the system, that is, a continuous operation with one desiccant bed and one set of solar collectors. However, this does not take full advantage of many configurations possible with this system. As presented so far in this study, this system would be used only at harvest time and would otherwise sit idle. But, the solar collectors could be utilized in a variety of ways throughout the rest of the year. Two of these uses are to heat buildings, for human or livestock habitation, or to regenerate beds of desiccant to seal, store and save for later use during harvest. Aldis, et al (2) studied the use of silica gel in drying corn and milo, and report a 15% loss of drying potential when the silica gel is stored for 87 days. Utilizing one set of solar collectors and several desiccant beds would take advantage of sunny days throughout the spring and summer, allow for more continuous operation during harvest independent of the weather, and, in general, better utilize the facility. Further study and incorporation of this configuration into the Economic Analysis computer model would shed light on some interesting alternatives and applications for such a system.

THE COMPUTER PROGRAM

The computer program calculates the life cycle costs for a solar energy and a conventional drying system so that they may be compared. Computations are carried out independently, allowing different economic parameters for each system. The parameters common to both systems are:

1. Inflation rates - fuel
 - general
2. Tax rates - Federal
 - State
3. Term of economic analysis
4. Fuel cost
5. Income or Non-income producing operation

The parameters that can be different for either system are denoted by an S or C for Solar or Conventional respectively, and are:

6. Market discount rate
7. Mortgage - term
 - downpayment
 - interest rate
8. Depreciation - life
 - method
9. Equipment cost
10. Property Tax rate
11. Assessed Valuation
12. Resale value
13. Miscellaneous costs
14. Federal Tax Credit - Note that this applies only to the
 Solar Energy System

These data are read at the beginning of the Economic Analysis portion of the program. Data required to vary the load are provided both through the common block and by cards. The parameters supplied in the common block are:

15. Solar collector heat gain
16. Solar collector area
17. Hourly dehumidifier exit humidity ratios
18. Air mass flowrate

Data provided by cards include:

19. Commodity equilibrium moisture content
20. Drying constants
21. Specified Wet Bulb Temperature
22. Enthalpy of air corresponding to the wet bulb
Temperature
23. Hourly station atmospheric pressure

The economic analysis portion of program MAIN can be used without calling subroutine SYSTEM. In this case however, the variables 15-23 listed above must be provided as input to the program, along with the number of hours per day and number of days included in the study: Parameters are dimensioned in arrays with the number of days and hours similar to the arrays in the Weather Data program. The program first calculates the hourly moisture evaporated utilizing Henderson's equation. The total daily moisture evaporated is summed and printed along with the hourly results. This is repeated for every day, and the grand total evaporated moisture for the time period studied, is summed.

The procedure for calculating the life cycle cost is identical for the conventional and solar energy systems. After the evaporated moisture is estimated, the program determines the years over which mortgage payments contribute to the analysis. This is the minimum of the term of the loan and equipment lifetime. If the term of the loan is less than the equipment lifetime, all payments will contribute to the analysis. This method was used

to choose N_{\min} for this study. But if the equipment life is chosen to be less than the term of the loan, then the term depends on the rationale behind that choice. If the discounted cash flow is calculated over the equipment lifetime without regard for costs outside of that time period, N_{\min} equals the term of the equipment lifetime. If all payments are expected to continue past the equipment lifetime, N_{\min} equals the term of the loan. If the equipment lifetime is chosen as the time of sale of the facility, then N_{\min} would equal the equipment lifetime, the remaining loan principal would be repaid at that time, and the principal balance would be deducted from resale value.

This same argument is used to determine the minimum years over which depreciation deductions contribute to the analysis. For this study, the depreciation life is chosen to be less than the equipment lifetime.

Next, the effective tax rate, \bar{t} , is calculated as given by equation 4.5. This accounts for state income taxes being deductible from income for federal tax purposes, but federal taxes are not deductible for state tax purposes.

The present worth factors included in calculating P_1 and P_2 utilize a Fortran statement function. These statement functions, PWF and PWFP, define the functions given by equation 4.3, using one statement and dummy variables. After calculating values for present worth factors, all the terms for P_1 and P_2 are calculated. P_1 and P_2 for the solar energy system are calculated first, followed by P_1 and P_2 for the conventional system.

Life cycle costs can then be calculated, depending on the load. The base load is set at the moisture that the silica gel dehumidifier alone can evaporate. Increasing the load by addition of a conventional heater varies the fraction of energy supplied by solar energy. This Economic Analysis program accompanied by a sample of the output is listed in Appendix D.

CHAPTER V

APPLICATIONS

The sorption dehumidification system finds applications wherever process air is utilized for dehumidification or humidity control. Besides a variety of agricultural and food processing applications, pharmaceutical and chemical manufacturing utilizes dehumidified air to reduce caking and lumping of powdered products. Paper manufacturing and lumber drying also require dehumidified air. Humidity control in marine environments is required to reduce rot, mildew and corrosion in storage areas. Humidity control also finds an application in the area of human comfort. This chapter discusses some of these areas, and is provided as a springboard for further study. Many other applications for this system are left to the reader and his imagination.

GRAIN AND OILSEEDS

Safe storage of cereal grains and oilseeds depends on the equilibrium relative humidity (E.R.H.) and the length of time stored. Grain and oilseeds are susceptible to mold growth and insect infestation if the equilibrium relative humidity exceeds 70%. The equilibrium relative humidity, temperature, history, condition and type of commodity all have an effect on the moisture content. Weather conditions occasionally force the farmer to harvest grain before it is at a moisture content suitable for storage and some grains give higher yields when harvested at higher moisture contents. Grain drying is then necessary.

The moisture content of grain will be in equilibrium with the relative humidity of the surrounding air as described by isotherms on an equilibrium moisture curve for a particular commodity. Equilibrium moisture curves also depend on the direction of moisture movement. Grain undergoing desorption

has a higher moisture content for a given relative humidity than grain undergoing adsorption, as shown in Figure 12. Figures 13, 14 and 15 show several isotherms for desorption of corn, soybeans and peanuts (1). With a given relative humidity, the moisture content will vary for different commodities due to structural and chemical compositional differences. For example, at 75% relative humidity and 30°C, the moisture content of corn is 14%, of soybeans is 13% and of peanuts is 8%.

Figure 16 shows that the allowable storage time for corn decreases as the grain temperature increases and as the moisture content increases (23). For a temperature of 60°F, the difference in storage time is twice as long for 18% moisture content as for corn with 20% moisture content. Likewise, at 20% moisture content, the allowable storage time at 70°F is 20 days, whereas at 50°F, it is 60 days. For minimum spoilage, drying should start immediately after the grain is harvested.

Table 6 lists the safe maximum moisture contents for storage for one and five years for the eight major grains produced in the United States. Some grains such as wheat and soybeans rarely require extra drying. However, corn is an example which requires drying more frequently. It can be seen in Figure 17 that the states with high corn production also required large amounts of energy to dry crops in 1974. In order to dry corn at 28% moisture content to a safe storage level of 12%, 11.9 pounds of water must be removed per bushel (7). Present figures estimate that 2000 BTU are required to evaporate one pound of water from grain (22).

Caution should be exercised when drying so that the grain is not overheated. High grain temperatures can reduce grain quality for many uses through: killing the germ, thereby decreasing germination or malting qualities; changing the nature of the chemical constituents - gelatinize the starch of grain with

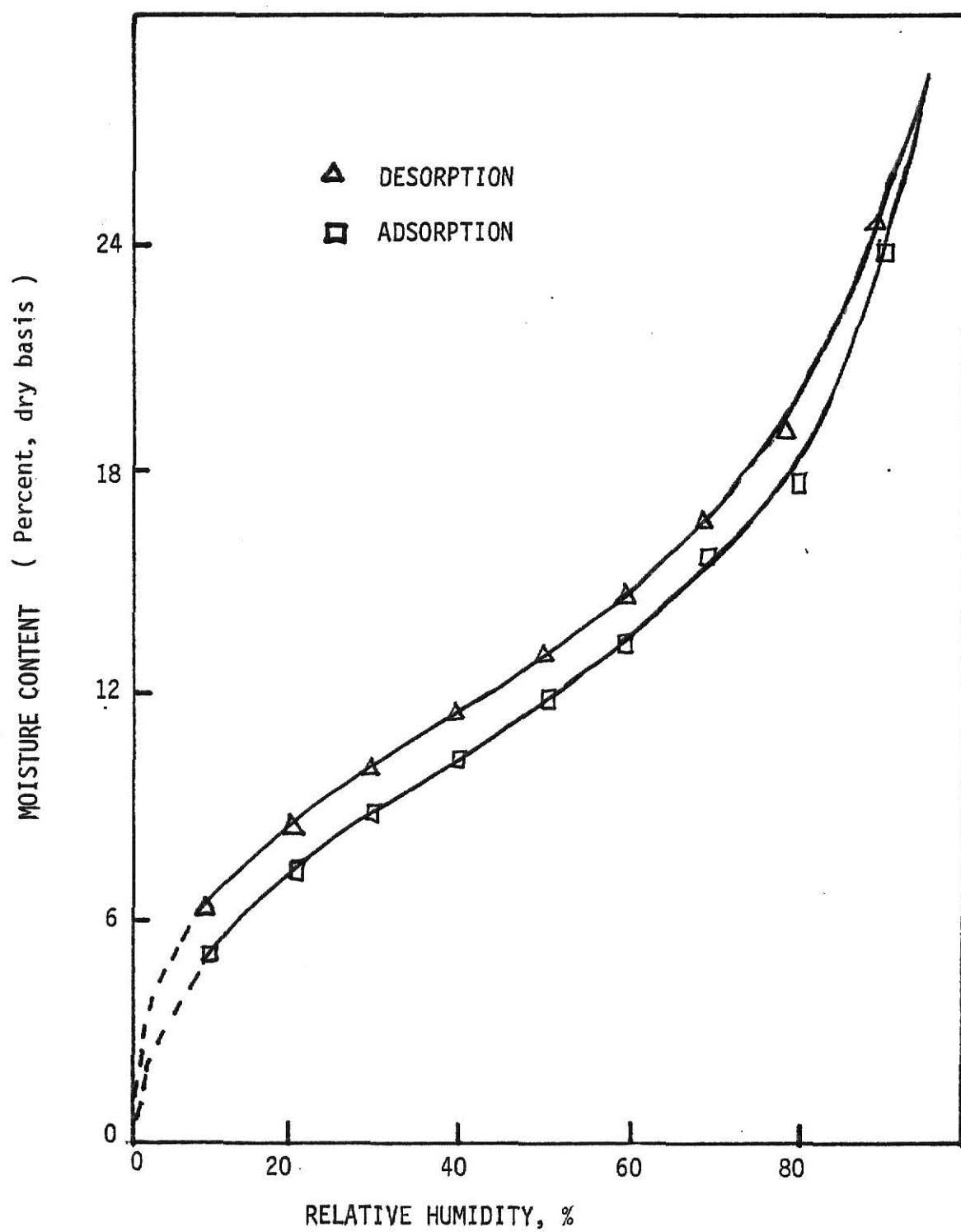


FIGURE 12. Adsorption - desorption isotherms of corn at 22° C.
Chung (1966).

FIGURE 13

EQUILIBRIUM MOISTURE CURVES
YELLOW DENT CORN

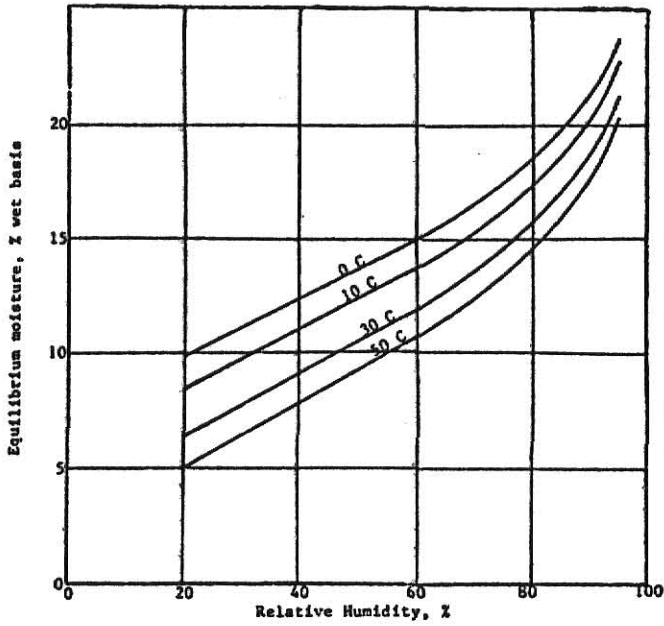


FIGURE 14

EQUILIBRIUM MOISTURE CURVES
SOYBEANS

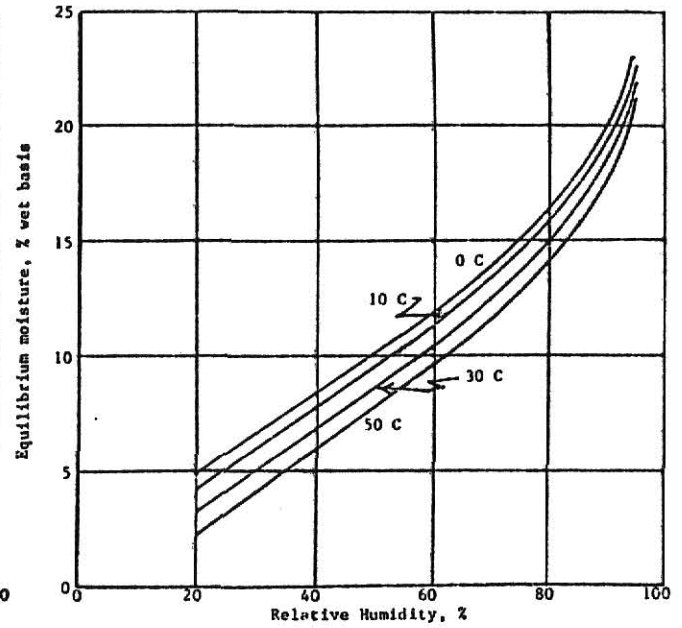


FIGURE 15

EQUILIBRIUM MOISTURE CURVES
PEANUT KERNEL

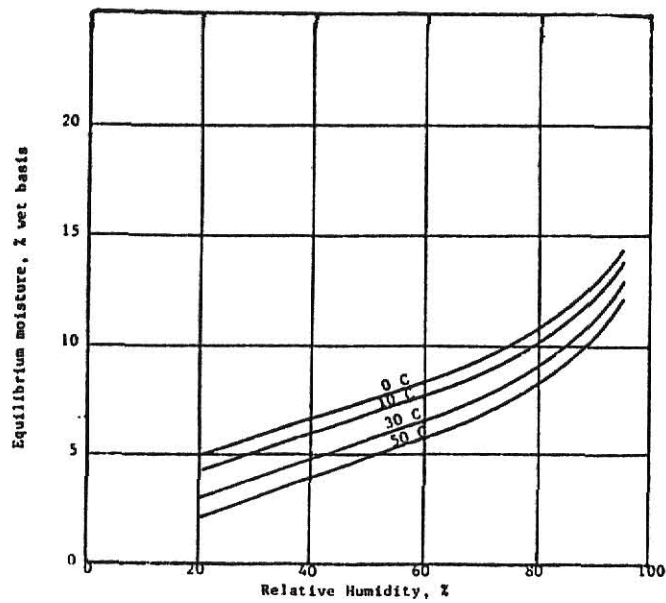


FIGURE 16

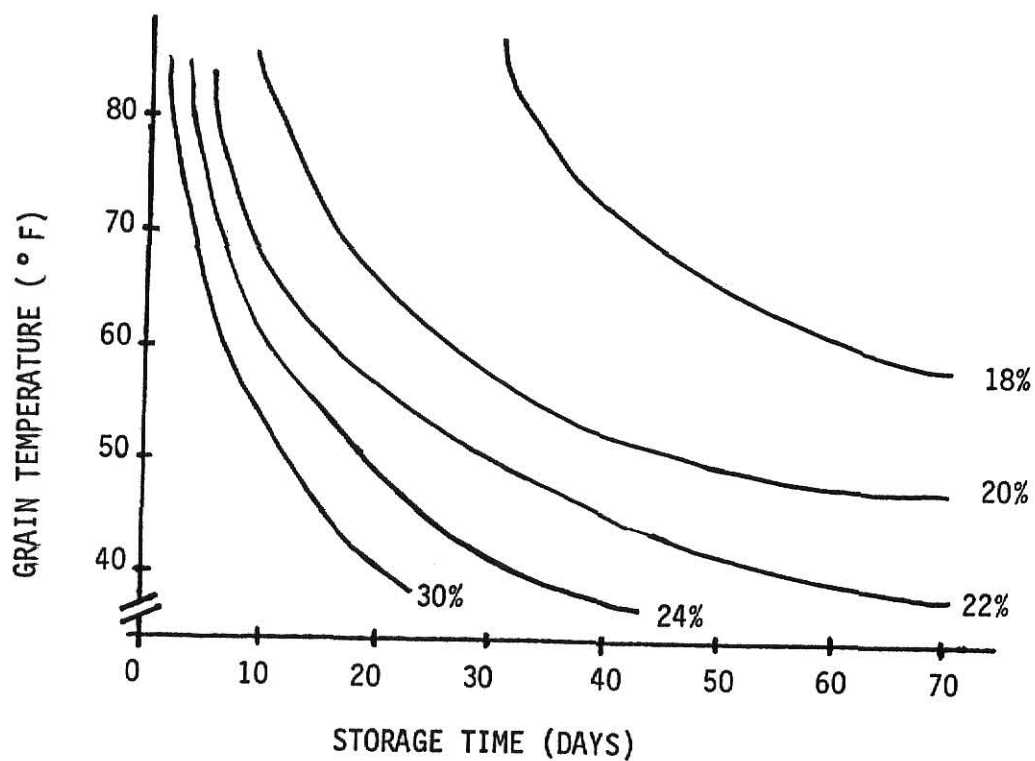
ALLOWABLE STORAGE TIME FOR CORN

TABLE 6

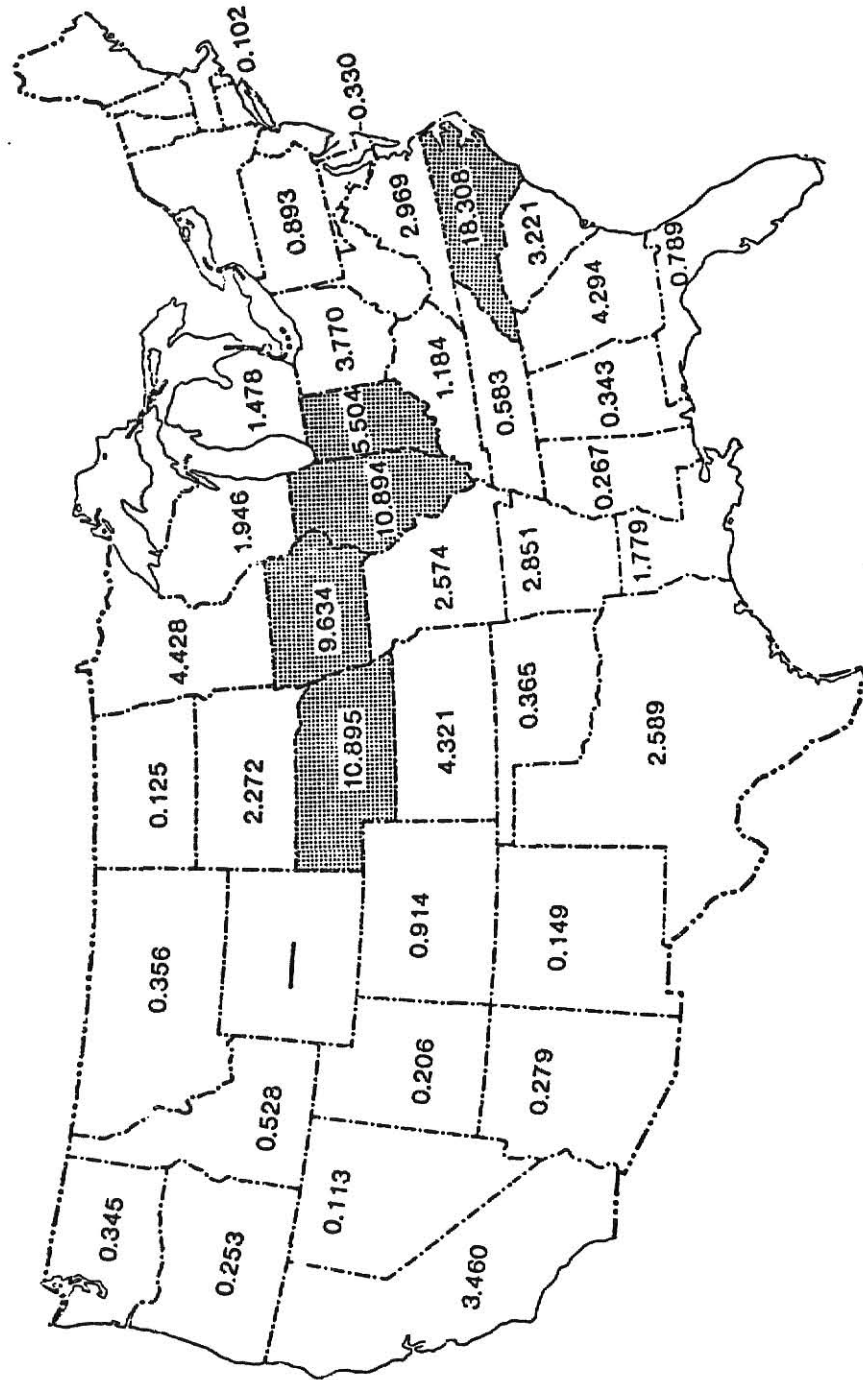
MOISTURE CONTENT (%) OF VARIOUS GRAINS

| <u>CEREAL</u> | <u>BEST AT HARVEST</u> | <u>USUAL AT HARVEST</u> | <u>1 YEAR STORAGE</u> | <u>5 YEAR STORAGE</u> |
|---------------|------------------------|-------------------------|-----------------------|-----------------------|
| BARLEY | 18-20 | 10-18 | 13 | 11 |
| CORN | 28-32 | 14-30 | 13 | 10-11 |
| OATS | 15-20 | 10-18 | 14 | 11 |
| RICE | 25-27 | 16-25 | 12-14 | 10-12 |
| RYE | 16-20 | 12-18 | 13 | 11 |
| SORGHUM | 30-35 | 10-20 | 12-13 | 10-11 |
| SOYBEANS | | 12-13 | 11 | 10 |
| WHEAT | 18-20 | 9-17 | 13-14 | 11-12 |

FIGURE 17

ENERGY CONSUMED FOR CROP AND GRAIN DRYING - 1974
 (10¹² BTU/YEAR)

Shaded Areas Represent 50% of Total Consumption



high moisture content, increase fat acidity, denature the protein, etc.; and through cracking the endosperm. These stress cracks result when a porous body is dried due to thermal expansion through temperature gradients and contraction caused by moisture losses (18). Thus, high temperatures or quick moisture removal lead to grain fissuring. The moisture removal rate can be explained by the diffusion of moisture from the center of the commodity. Drying occurs through water vaporization from the grain surface when the water vapor pressure exceeds the partial pressure in the surrounding air. Diffusion of water takes place from the center to the surface, with the rate depending on the nature of the commodity. Once the seed coat is broken, the grain becomes much more susceptible to mold and insect invasion. Table 7 lists temperatures safe for various commodities and their uses.

High temperatures can cause fissures in grain, but so can extremes of relative humidity. White (41) reports fissuring in soybeans caused by air with low relative humidities, whereas Kunze and Prasad (28) found that high relative humidity air causes rice to fissure. Use of the heat exchanger in the sorption dehumidification system allows control of the air temperature and relative humidity so that high temperatures and extreme relative humidities may be avoided.

Conventional grain dryers heat air with propane or natural gas to high temperatures and blow the hot air through the grain, but vary in configuration. Continuous flow dryers convey grain either with a horizontal belt, or vertically by gravity. Since continuous, they can handle large amounts of grain with relatively uniform moisture contents, but have low thermal efficiencies. Batch drying can be done either in a bin or with a portable batch dryer. As the name implies, a batch of grain is dried at one time, then transferred to another container for storage. This method is restricted to smaller quantities

TABLE 7

AIR TEMPERATURES UTILIZED TO DRY
CEREAL GRAINS AND OILSEEDS

| <u>GRAIN</u> | <u>PURPOSE</u> | <u>MAXIMUM SAFE TEMPERATURE - °F</u> |
|---------------------------|----------------|--|
| CORN | DISTILLING | 140 |
| | FEED | 280 |
| | MILLING-DRY | 120 |
| | MILLING-WET | 130-140 |
| | SEED | 100-110 |
| OILSEEDS | | 115 |
| RICE | | 120-130 |
| SORGHUM | | 140-150 |
| WHEAT | FLOUR | 160 |
| | SEED | 100-110 |
| OTHER CEREAL GRAINS | SEED | 100-110 |

of grain, yet operates with a higher thermal efficiency than a continuous dryer. Dryeration is yet another method employed to dry grain. It operates in two stages, drying grain from 20-30% moisture content to 16-18% with air temperatures ranging from 90-115 °C in the first bin. The grain is then transferred to a second bin, allowed to set for 4-8 hours, and cooled with ambient air which simultaneously removes an additional 2 percent moisture. The advantages of dryeration are a high thermal efficiency, slower moisture removal, reduced grain breakage and a higher dryer capacity.

Natural air drying has found increased interest since the advent of higher fuel prices. Ambient air, that is 10 °F lower than the grain temperature, is blown through a bed of grain. This method is effective only if the air relative humidity is lower than 62% (29). This method requires good weather conditions and long periods of time to dry completely. Supplementing the natural air dryers with solar preheating has been found to reduce the drying time (32).

Due to the long drying time with natural air dryers, even with solar preheating, researchers have studied alternative methods to dry grain. Rodda and Rode (34) studied the use of a desiccant dehumidifier with an air conditioner to dry grain and found the energy efficiency comparable to that of heated air dryers. Danziger, et al (14) studied the effect of drying field corn with air dehumidified with silica gel and also with the corn mixed directly with silica gel. They report an increase of quality when drying corn with silica gel at room temperature. Aldis, et al (2) compared corn and milo dried with natural air to corn and milo dried with air dehumidified by silica gel. Drying time was considerably shorter using the dehumidified air than with the natural air. Mold growth also decreased with use of the dehumidified air. No differences were found in energy requirements or susceptibility to breakage. Silica gel has also been studied as a method to dry and maintain a dry moisture

content of grain in humid climates. Hsiao (24) placed bags of silica gel in grain bins and determined the ratios of silica gel and grain required with different initial grain moisture contents, and the frequency of regeneration necessary for the silica gel.

ALFALFA

Alfalfa meal is fed to cattle, swine, sheep and poultry. Fresh cut alfalfa ranges in moisture content from 65-85% and should be dried immediately to 6-8% for pelleted feed (26). Currently, alfalfa is dried in rotary type drums, either in single or multiple drums at a high rate, removing 6000 pounds of water per hour at temperatures around 800-1000°C. However, research shows that alfalfa dehydrated at 180 °C, compared to 130 °C, reduced digestibility, nitrogen retention and growth of sheep (19). Krause and Klopfenstein (27) studied the effect of freeze dried, sun cured and oven dried alfalfa, and of soybean meal on steer and lamb production. They report a desirable low ammonia concentration in alfalfa dehydrated with temperatures of 80-120 °C. Therefore, a sorption dehumidification system would produce a better quality of air for dehydration. Table 8 lists the geographical areas and number of dehydrating drums in operation in 1981. Alfalfa is cut approximately four times during a season at 30 day intervals beginning in early June (38).

MISCELLANEOUS AGRICULTURAL PRODUCTS

Distillers grain, occasionally fed to cattle and swine, is another area where drying improves allowable storage time (13), and could be dried with a sorption dehumidification process. Distillers grain is the byproduct of fermentation in the production of alcohol which ranges in moisture content from 85-95%. It has been found that 16-18 pounds of distillers grain is recoverable

TABLE 8

ALFALFA DEHYDRATING DRUMS
IN OPERATION IN 1981

| <u>STATES</u> | <u>NUMBER OF DRUMS</u> |
|---|----------------------------|
| NEBRASKA | 99 |
| KANSAS | 41 |
| IOWA, MINNESOTA, WISCONSIN, NORTH DAKOTA, SOUTH DAKOTA | 31 |
| MARYLAND, PENNSYLVANIA, OHIO, MICHIGAN | 30 |
| CALIFORNIA, NEVADA, ARIZONA, WASHINGTON, OREGON | 15 |
| COLORADO, UTAH, MONTANA, IDAHO | 14 |
| ARKANSAS, TENNESSEE, ALABAMA | 6 |
| ILLINOIS, MISSOURI, INDIANA, KENTUCKY | 5 |
| TEXAS, OKLAHOMA, NEW MEXICO | 4 |

from a 56 pound bushel of grain. The net energy values for dry distillers grain and corn do not differ, and most alcohol fuel byproducts contain 28-30% protein, about three times that found in corn.

The citrus industry utilizes several drying operations. Miller (31) reports that surface moisture removal from fresh citrus fruit ranks as the principal consumption of energy on the packinghouse line, requiring 6.59 kW per 40.8 kilogram box of fruit. The production of juice leaves citrus pulp and peel as byproducts (6). When dried to 10% moisture, it is almost equivalent to the nutritional value of corn for livestock feed.

Grapes are sun-dried on racks or on the vine, or dehydrated in tunnel dryers for raisin production in California. Raisins produced by artificial dehydration vary in flavor and color from sun-dried fruit and are less desirable. But weather conditions in California's Central Valley during the fall do not always cooperate with the sun-dried operation. Studer and Olmo (39) report that a combination of partial drying on the vine completed by artificial dehydration would produce a fruit similar to the sun-dried raisin while diminishing the risk of bad weather.

Approximately 57% of the potato crop undergoes processing, with 22% of that being dehydrated (37). Dehydrated potatoes take forms of granules, flakes, flakelets and diced. Washington and Idaho are the major producers of processed potatoes, followed with high production in Maine, North Dakota, Minnesota, Oregon and Michigan.

Other agricultural crops that require some degree of drying are:

| | |
|----------|-----------|
| Apricots | Onions |
| Apples | Parsley |
| Bananas | Peaches |
| Carioca | Pineapple |
| Carrots | Plums |
| Cassava | Yams |
| Figs | |

In drying fruit, careful consideration must be given to the sensitivity of the crop to temperature, ultraviolet light and bacterial action (29). Some of the applications listed above may be a sort of novelty, nevertheless, they are presently dried with conventional systems.

One application reported by Farmer, et al (17) is in solar drying of paunch contents. In the processing of meat, the rumen of a mature, slaughtered animal contains 25-30 kilogram of partially digested feed with high moisture contents. If dried, paunch contents can be used as animal feed. They developed an enclosed dryer utilizing direct solar energy. Use of a sorption dehumidification system could insure continuous operation for this application.

PAPER AND PAPERBOARD

Another industry which uses large quantities of fossil fuels for drying process heat is in the manufacture of paper and paperboard. Following chemical, refinery and primary metal production, the paper industry is the fourth leading energy consumer in the manufacturing industry and the leading consumer of fuel oils (10). Wood is pulped by either mechanical grinding or by chemical reduction, and the pulp is combined with water to form the correct consistency (25). The pulp fibers are fed onto a fast moving wire mesh to form a sheet of paper. At this point, the paper contains substantial amounts of water, with a ratio of 200:1 of water to fiber. The major portion of this water falls through the wire due to gravity as the mesh moves, and a suction section placed near the end of the mesh removes an additional amount of moisture. The paper then passes through a series of presses, reducing the moisture content from about four to two kilograms of water per kilogram of fiber. The presses can only reduce the moisture content to a minimum of 55-60 percent moisture, otherwise the fibers would be crushed and the paper quality reduced. Heat must then be

added to reduce the finished paper moisture content to 8 percent.

The actual drying process requires from 7.6 to 14.5×10^6 Joules per kilogram of the finished product. Water must be removed from the paper itself, but the felt presses also require drying to some degree. Higher temperatures ranging from 300 to 450°C may be used.

LUMBER

Lumber drying is still another area where a dehumidification process may be quite feasible. Dehumidification drying of spruce studs studied by Cech and Pfaff (8) was found to consume less energy than the conventional steam dried kiln (Figure 18). A refrigerant system was used to condense air vapor and produce dry, hot air. It was found that the three dehumidification charges consumed 46-68 percent less energy than the conventional kiln dried sample with no appreciable quality difference. However, some means to provide stress relieving of the studs must be made, generally some additional humidification system. A lower temperature was used in the dehumidification process, 93°F instead of 150 - 180°F used for kiln drying. This lower temperature is well suited to temperatures produced by flat plate collectors. However, the lower temperature lengthens the drying time, as shown in Figure 19. One solution to decrease the drying time would be to combine the two processes, first using a dehumidification process, then at about 25% moisture content, finish drying with a conventional kiln. This could simultaneously provide stress relief for the lumber. Areas of major lumber production are the Pacific West, South Atlantic and Rocky Mountain states.

AIR CONDITIONING

The area of air conditioning has been studied by Rush and Macriss (35)

FIGURE 18

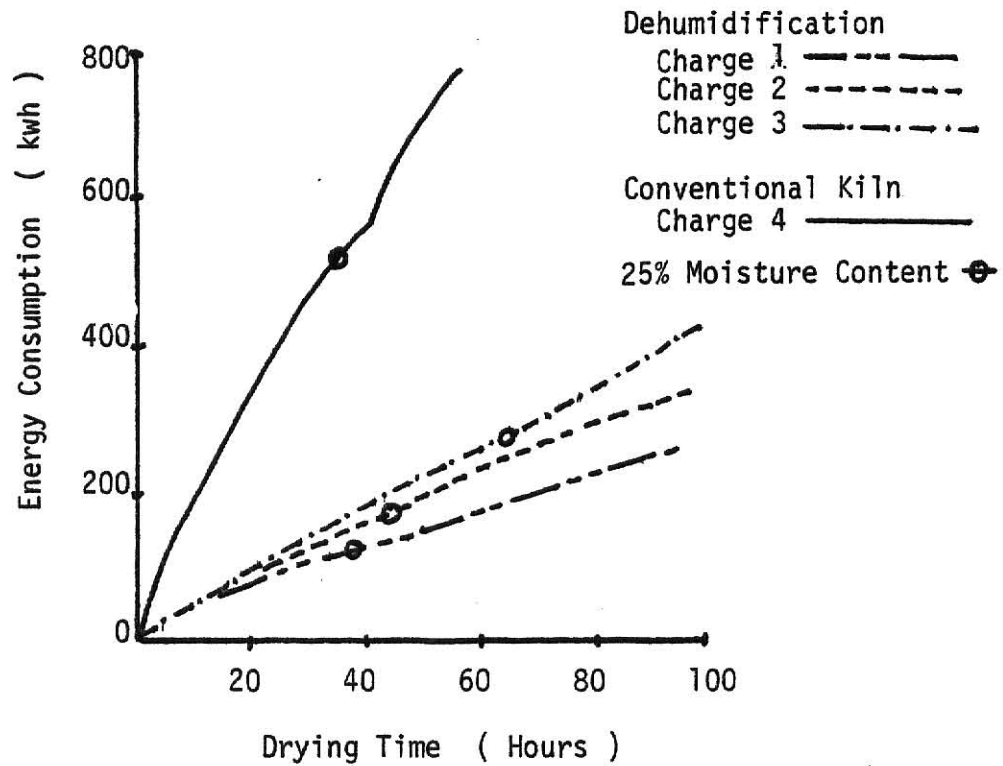
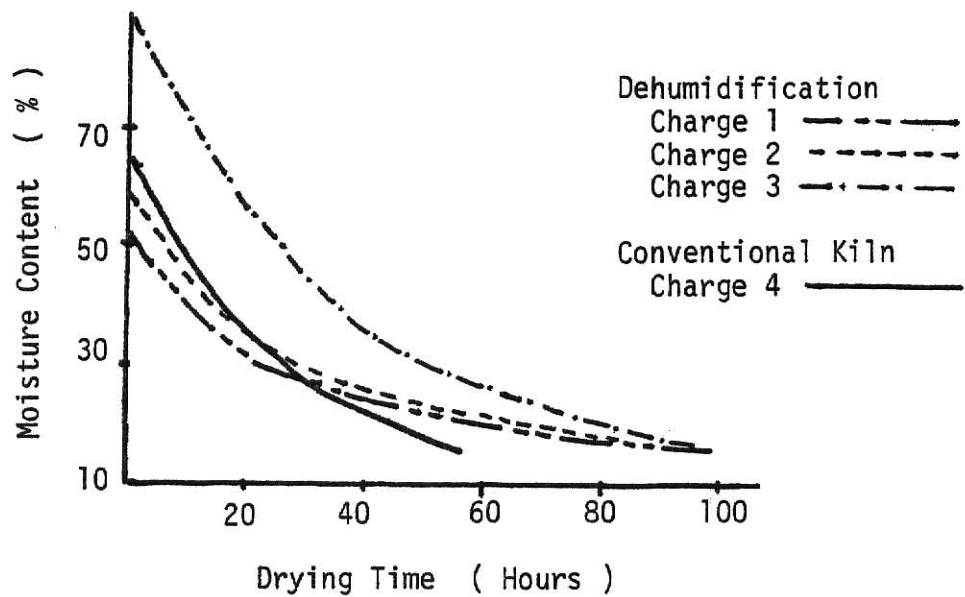
ENERGY CONSUMPTION FOR DRYING SPRUCE STUDS

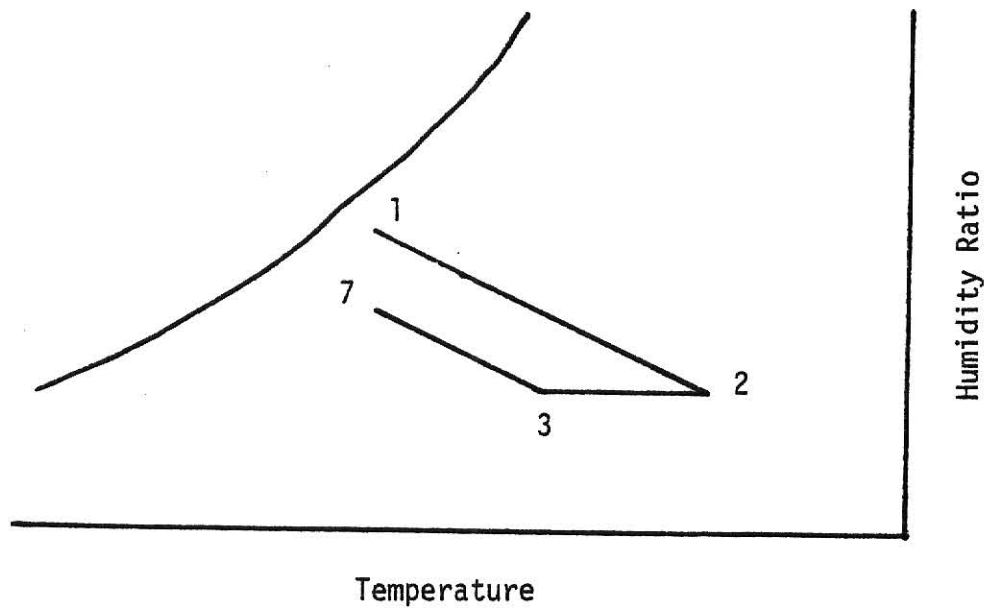
FIGURE 19

DRYING CURVES FOR SPRUCE STUDS

using asbestoes wheels for the rotating desiccant bed and heat exchanger, and by Dunkle (16) using a solar powered sorption system. The addition of a humidifier on the sorption system takes advantage of evaporative cooling, which then delivers cool, moist air to a building. The humidifying process takes place from 3 to 7 in Figure 20 on a psychrometric chart.

FIGURE 20

PSYCHROMETRIC PLOT OF
DESICCANT AIR CONDITIONING



CHAPTER VI

RESULTS AND DISCUSSION

The purpose of this chapter is to investigate the Economic Analysis model. Choosing one application at one location, the dependency of the life cycle costs can be examined as the dehydration and economic parameters vary. A good application for this system is drying grain, principally corn, throughout the midwest United States. The TMY station at Omaha, Nebraska lies right in the heart of large corn production where harvest takes place in October.

The Weather Data program described in Chapter III was run for the month of October, to locate five consecutive clear days. Clear weather occurred October 20-24, so the entire program - the combination of the Weather Data, Subroutine System and the Economic Analysis - was run for twenty four hours a day during that interval. This provided the dehumidifier process outlet air conditions and atmospheric pressure data required by the Economic Analysis program. With these data, the dependency of life cycle costs on the various economic and dehydration parameters can be studied by running only the Economic Analysis program (Appendix D).

Since the fraction of energy supplied by solar can be written as

$$F = \frac{Q_s}{L}$$

the third term for the life cycle cost of the fuel in equation 4.2 can be simplified to:

$$\begin{aligned} P_{1s} C_f L (1 - F) &= P_{1s} C_f L \left(1 - \frac{Q_s}{L}\right) \\ &= P_{1s} C_f (L - Q_s) \end{aligned}$$

Equations 4.1 and 4.2 are then recognizable as linear equations of the form $y = b + ax$, where:

| | <u>CONVENTIONAL</u> | <u>SOLAR</u> |
|-----|---------------------|---|
| y = | LCC_c | LCC_s |
| x = | L | L |
| a = | $P_{1c}C_f$ | $P_{1s}C_f$ |
| b = | $P_{2c}C_{Ec}$ | $P_{2s}(C_A A + C_{Es}) + P_{2c}C_{Ec} - P_{1s}C_f Q_s$ |

The dehydration parameters vary the load which the system can handle, while the economic analysis parameters vary the P_1 's and P_2 's. All other parameters (C_{Ec} , A , Q_s , . . .) remain constant for this study. Of the noteworthy dehumidification parameters, the inlet dryer wet bulb temperature and initial moisture content of the grain can be varied. The economic analysis parameters that most affect P_1 and P_2 are the Federal Tax Credit, Mortgage Interest Rate, Downpayment, Depreciation, Fuel Inflation Rate, Term of the Economic Analysis and the Market Discount Rate.

DEHYDRATION PARAMETERS

DRYER INLET WET BULB TEMPERATURE: As the set dryer inlet wet bulb temperatures of the air more nearly equal the air temperatures exiting the dehumidifier, the additional heat required decreases, so that the fraction of energy supplied by solar increases. For the solar energy system, Figure 21 shows how the life cycle cost varies with the fraction of solar energy as the inlet wet bulb temperature varies from 15 to 25°C. Note that when the origins for each curve are aligned, the curves lie along the same line. The wet bulb temperature has no effect on the life cycle cost, except indirectly through the fraction of energy supplied by solar as can be seen from Figure 22. This plot of life cycle cost versus the load has the same intercept and slope for each inlet wet bulb temperature, hence, the same line. A higher inlet wet bulb

FIGURE 21

EFFECT OF THE DRYER INLET WET BULB
TEMPERATURE ON THE SOLAR FRACTION

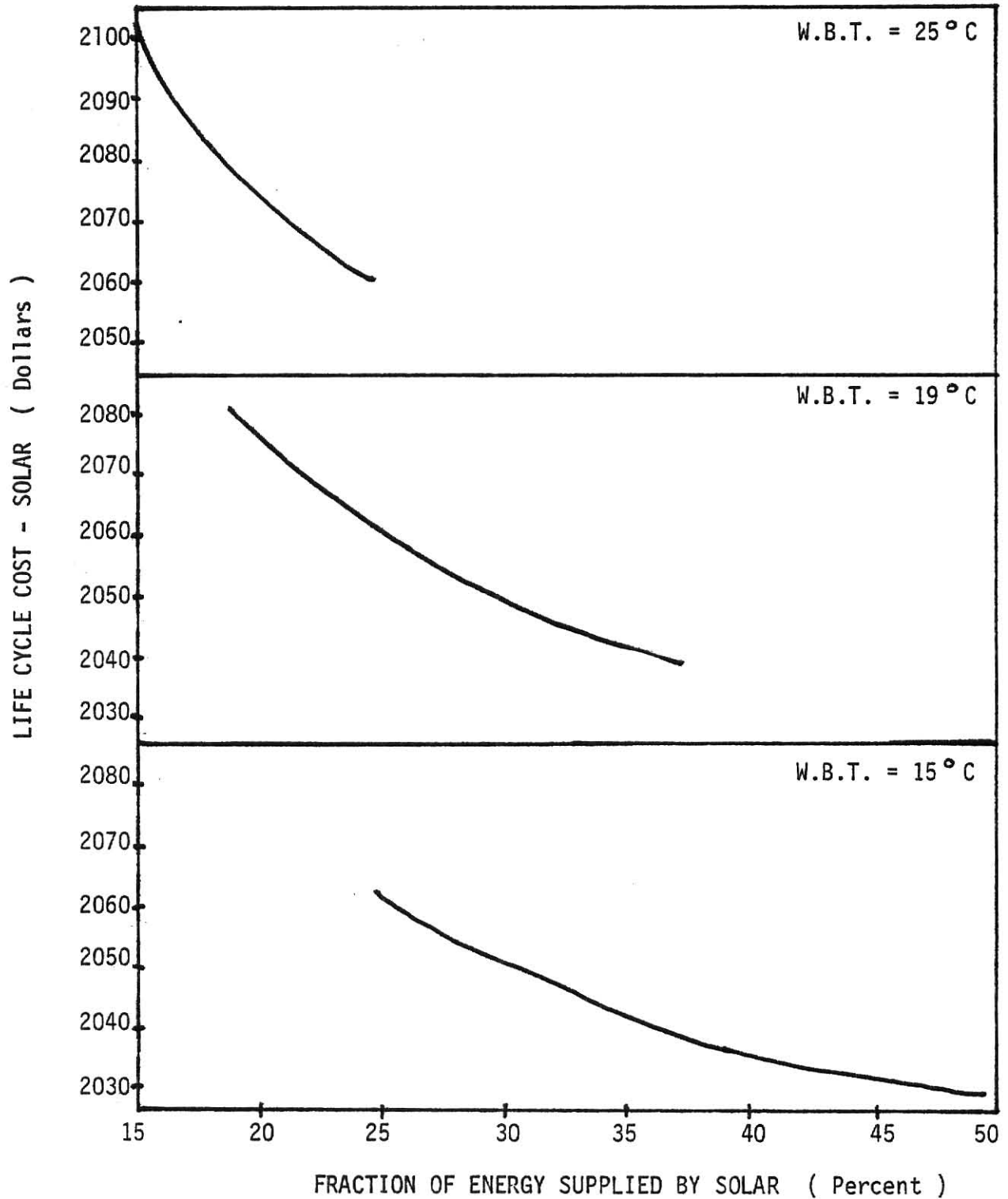
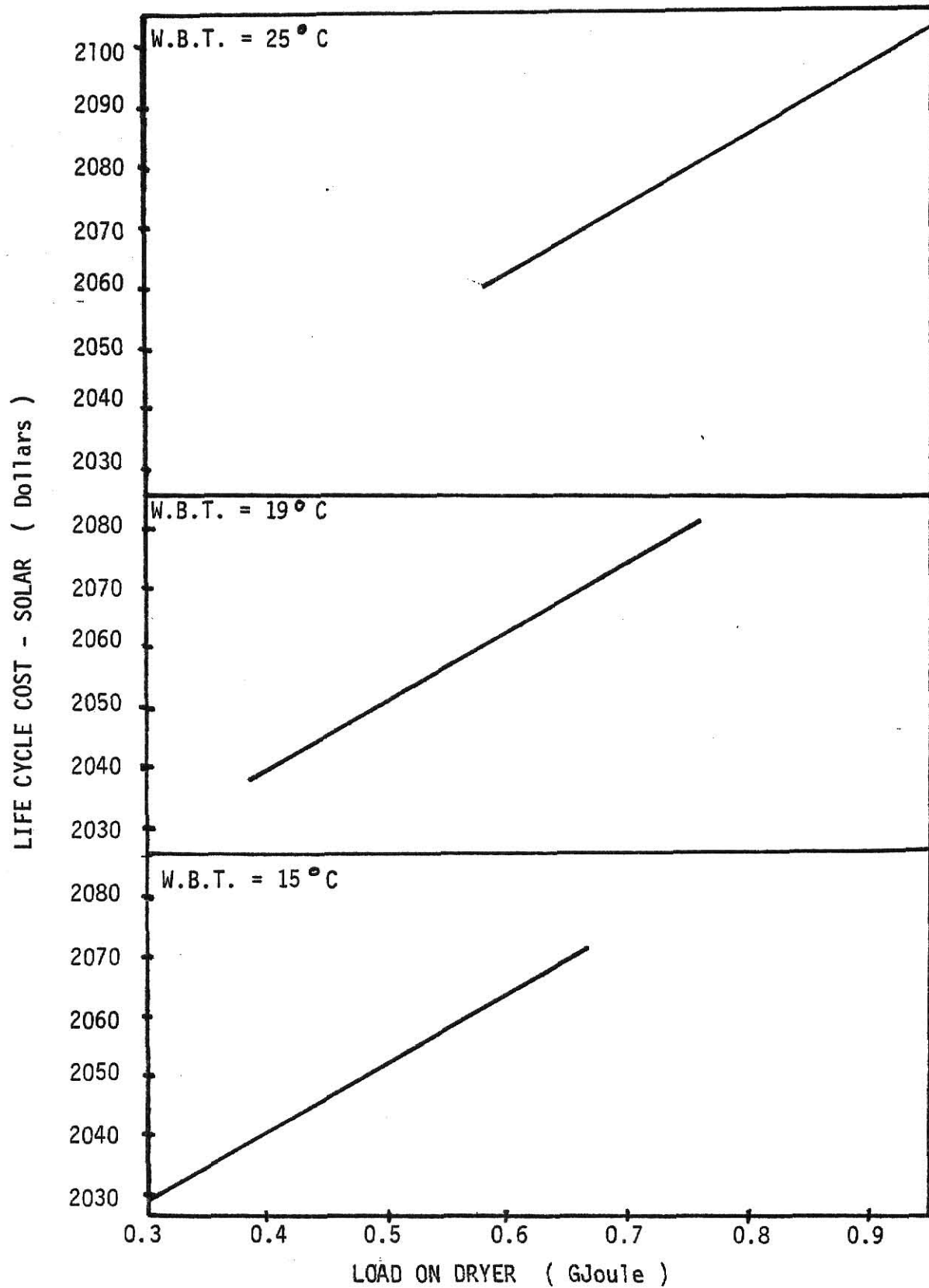


FIGURE 22

EFFECT OF THE DRYER INLET WET BULB
TEMPERATURE ON THE LOAD



temperature evaporates more moisture, thereby increasing the load on the dryer. Since the solar heat gain remains constant, the intercept, b , remains constant and only the load varies. The life cycle cost decreases as the fraction of energy supplied by solar increases.

INITIAL MOISTURE CONTENT: This also affects the life cycle cost only indirectly by varying the load, hence, the fraction of energy supplied by solar. Figure 23 is the plot of life cycle cost versus the load for moisture contents of 33.33 and 25 percent dry basis. The higher moisture content of the commodity imposes a higher load on the dryer, but does not directly influence the life cycle cost.

With these results, any reasonable dryer inlet wet bulb temperature and initial moisture content could be chosen as operating conditions, in order to vary the economic analysis parameters. An initial moisture content of 25% (corresponding to 20% wet basis), and an intermediate wet bulb temperature equal to 19 °C were chosen.

ECONOMIC PARAMETERS

From equations 4.4 and 4.6, the economic parameters affect P_1 and P_2 differently. The Federal Tax Credit (FTC), Mortgage Interest Rate (m), Down-payment (D) and Depreciation (DEP) vary only P_2 , which in turn affects the intercept of the life cycle cost equation. The Fuel Inflation Rate (i_f) varies only P_1 , thereby changing the slope of the life cycle cost equation. The Term of the Economic Analysis (N_e) and Market Discount Rate (d) appear in both terms for P_1 and P_2 . These parameters were varied individually, while the others remained at base values listed on page 122 in Appendix D.

FEDERAL TAX CREDIT : The effect of the federal tax credit can be seen in Figure 24. Since the tax credit applies only to a solar system, P_{2c} remains

FIGURE 23
EFFECT OF EQUILIBRIUM MOISTURE CONTENT
ON THE LOAD

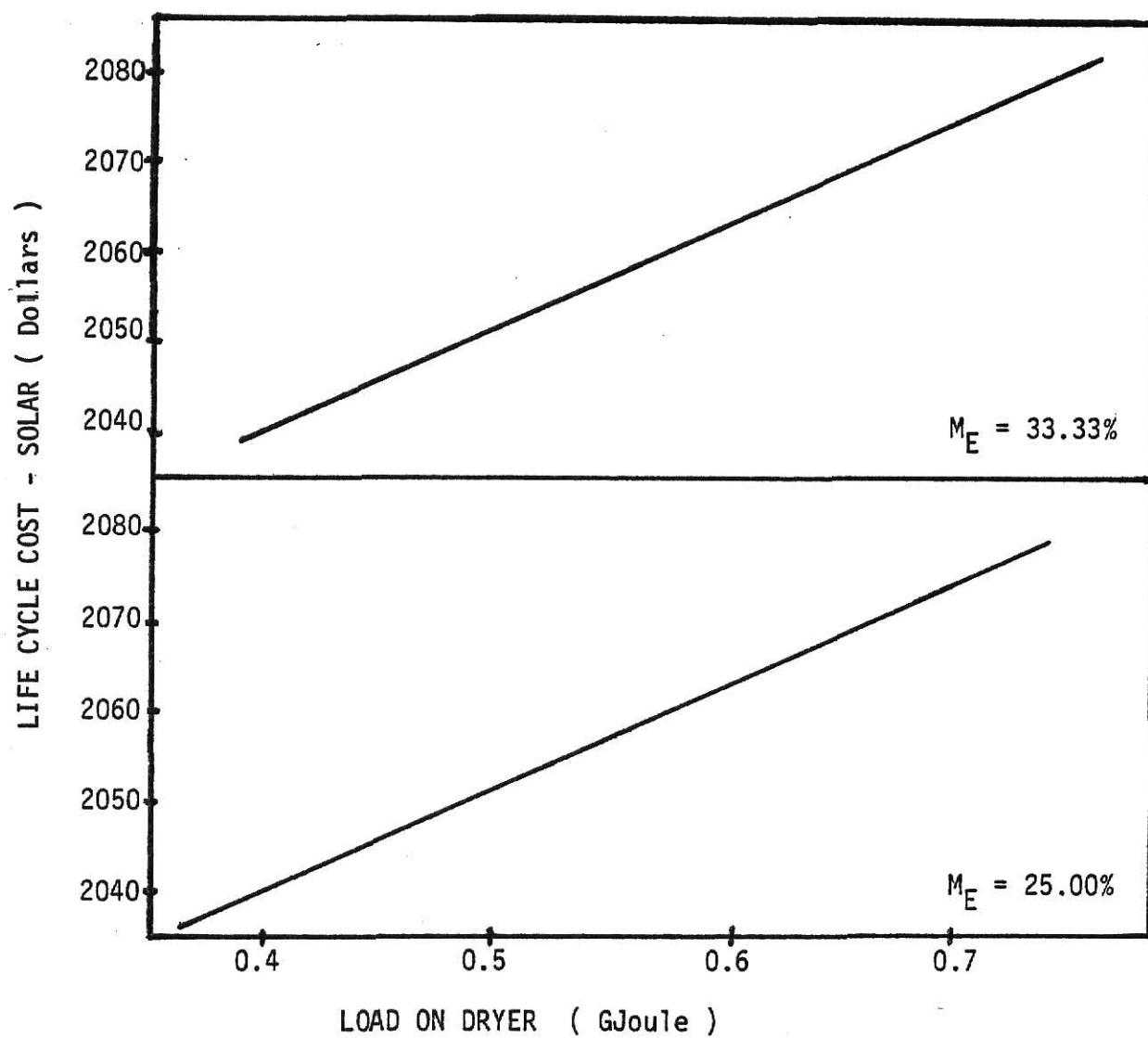
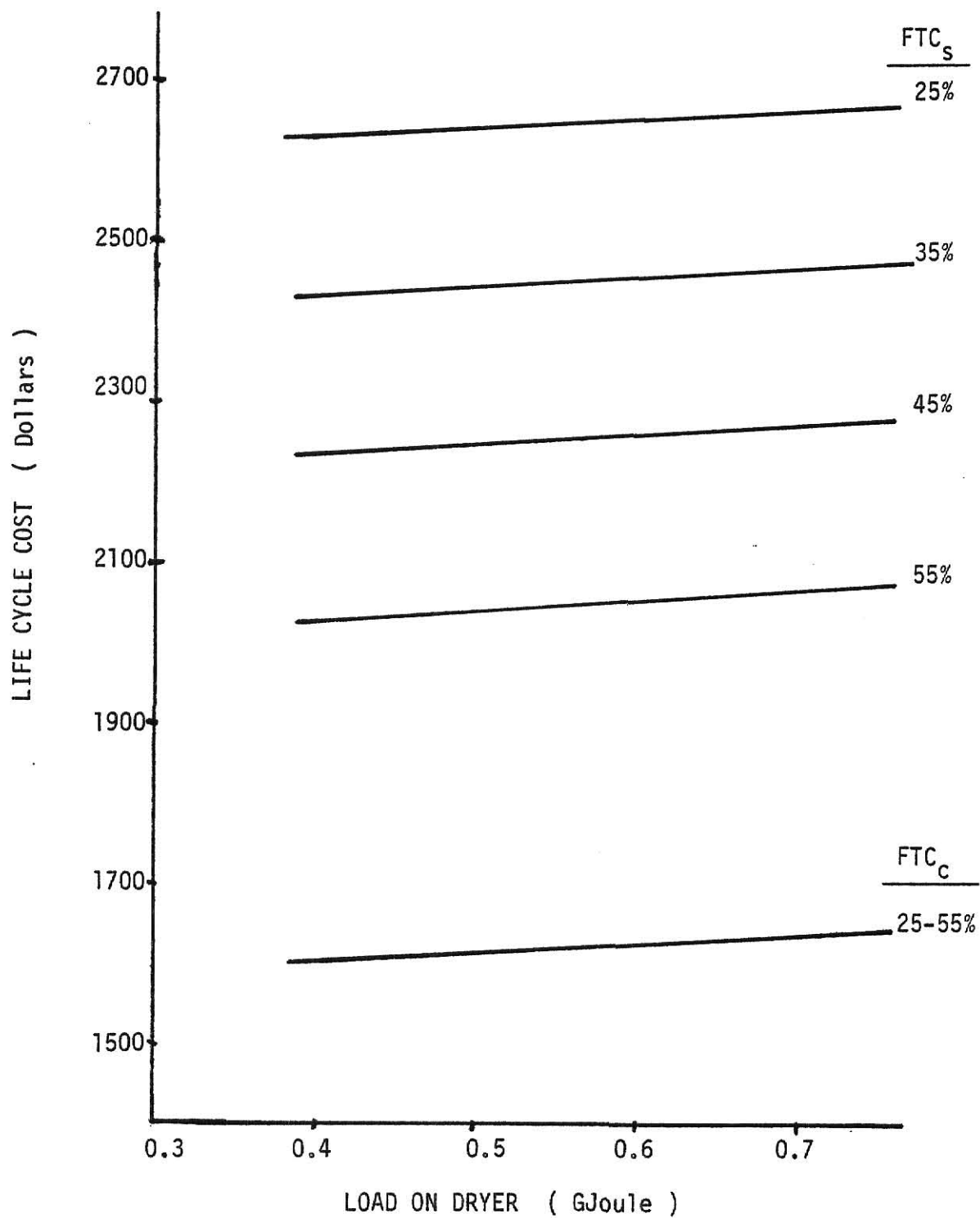


FIGURE 24
EFFECT OF FEDERAL TAX CREDIT
ON LIFE CYCLE COST



constant while P_{2s} changes. The life cycle cost decreases as the federal tax credit increases.

MARKET INTEREST RATE: The relationship between the Market Interest Rate and the life cycle cost is shown in Figure 25. As the mortgage interest rates decline, the life cycle costs decrease, as one would expect. The life cycle cost for the solar energy system at 14 and 16 percent differ by a larger amount than for the conventional system at 14 and 16 percent. This is due to a larger equipment cost for the solar energy system which includes both solar and conventional equipment. For the 2% intervals between 8-10-12-14-16 percent, P_{2s} differs by the same amount as P_{2c} .

DOWNPAYMENT: The downpayment varies linearly with P_2 as shown in Figure 26. Because of this, Figure 27 shows that as the downpayment increases, the life cycle cost increases. Again, the larger equipment cost for the solar energy system causes a proportionally larger change in the solar life cycle cost.

DEPRECIATION: The most favorable method of depreciation is accelerated cost recovery. Since the depreciation affects only P_2 , the slope of the life cycle cost versus the load is the same for the different methods. Figure 28 illustrates the advantage of the accelerated cost recovery depreciation over no depreciation for the systems.

FUEL INFLATION RATE: This parameter affects only P_1 and varies the slope of the life cycle cost equation. The slope increases as the fuel inflation rate increases (Figure 29). Note also that at one value of load, the life cycle cost for the conventional system varies with the fuel inflation rate more than the life cycle cost for the solar energy system. This can be attributed to the fraction of energy supplied by solar which reduces the amount of fuel required at a particular load.

TERM OF ECONOMIC ANALYSIS: The program was run with terms of 5 and 15 years

FIGURE 25
EFFECT OF MORTGAGE INTEREST RATE
ON LIFE CYCLE COST

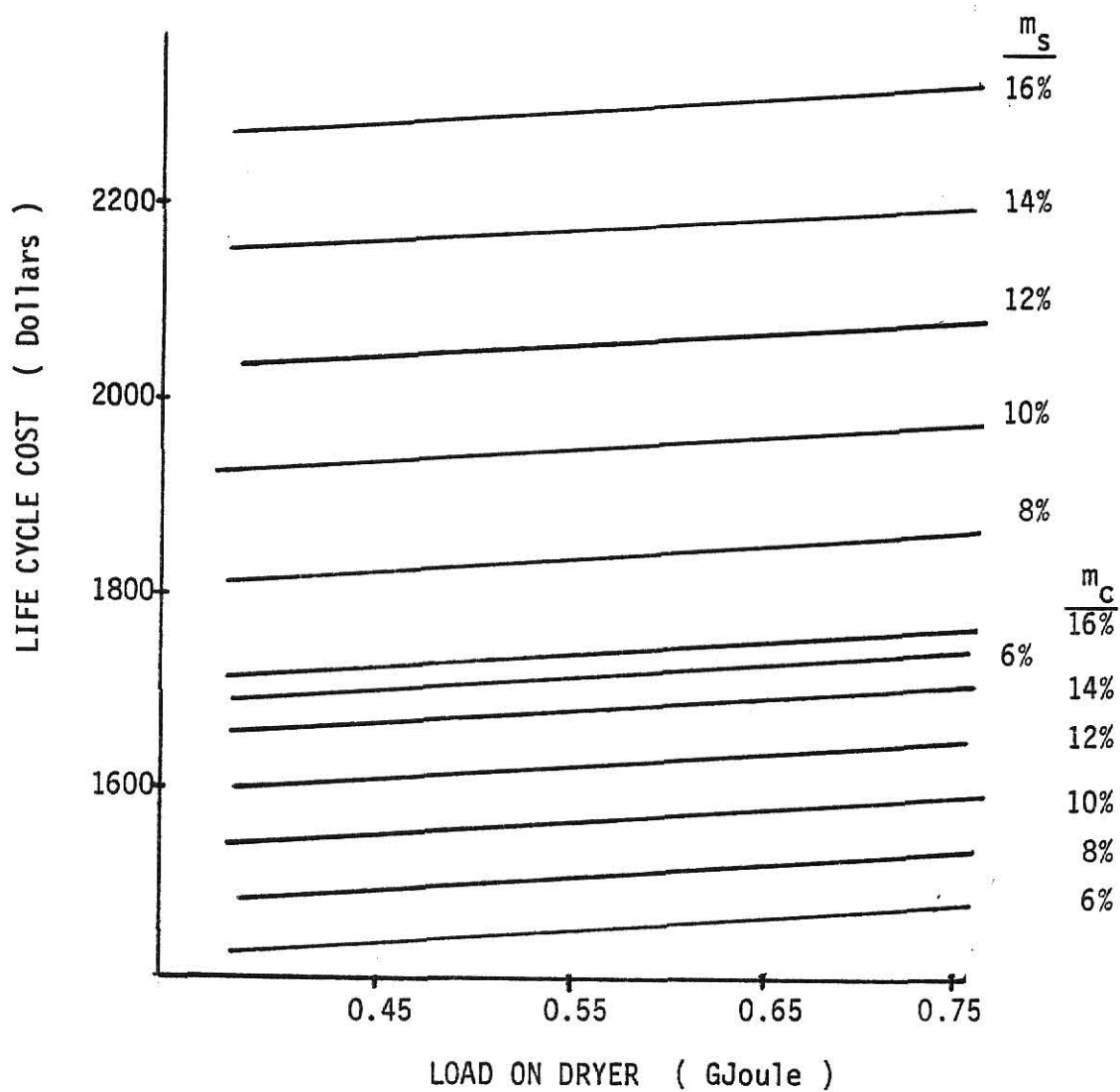


FIGURE 26
EFFECT OF DOWNPAYMENT OF P_2

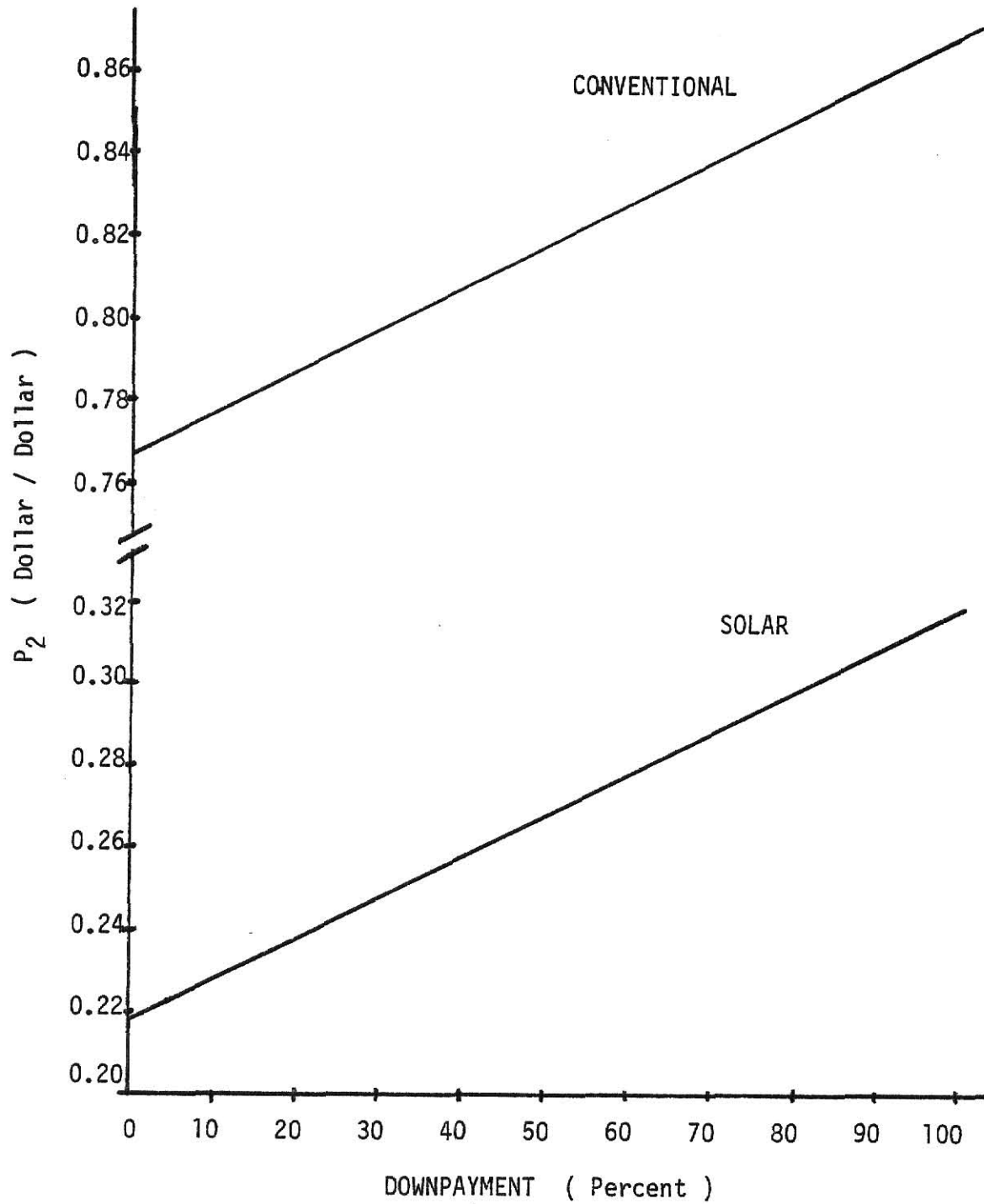


FIGURE 27
EFFECT ON DOWNPAYMENT
ON LIFE CYCLE COST

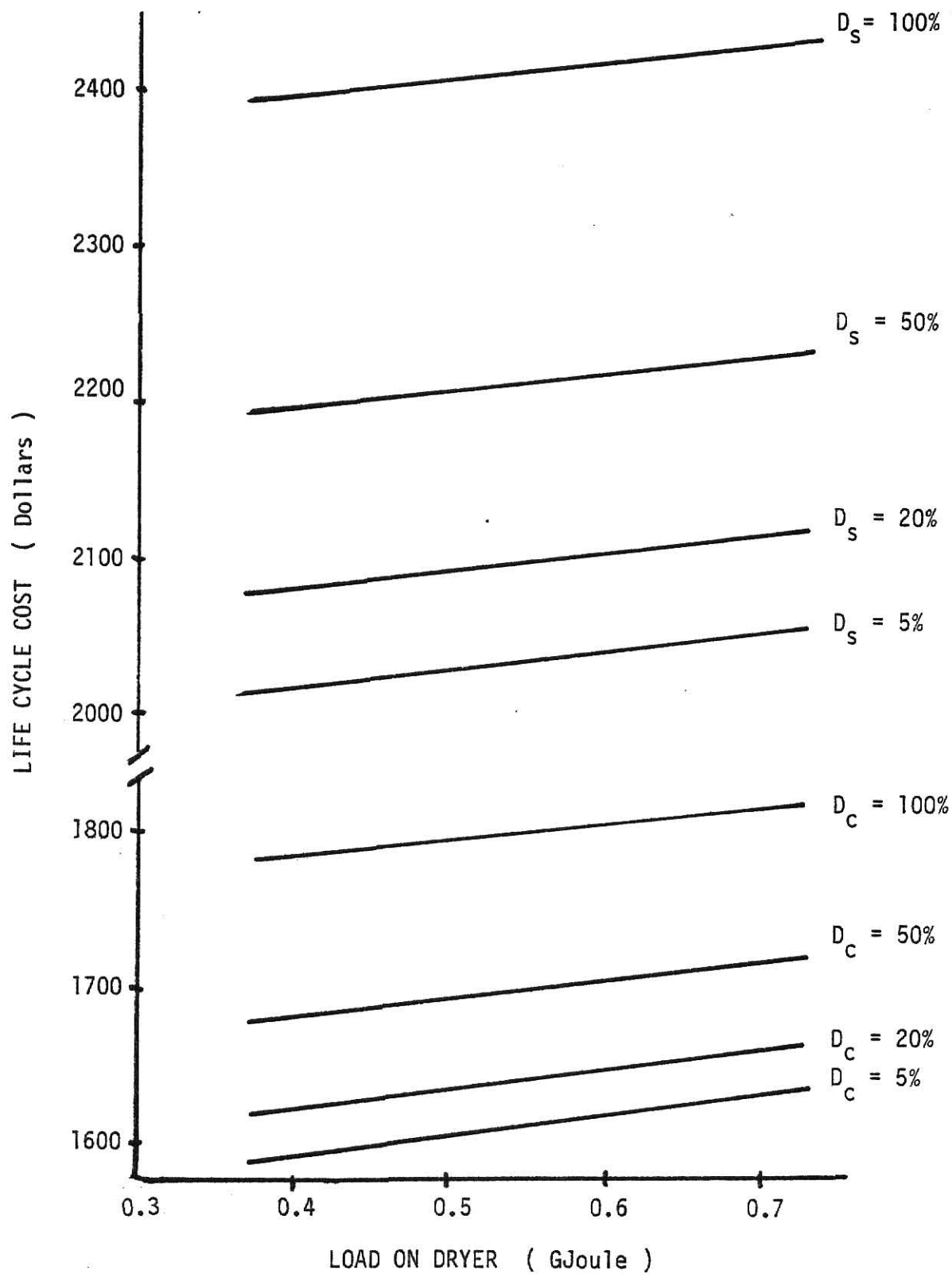


FIGURE 28
EFFECT OF DEPRECIATION
ON LIFE CYCLE COST

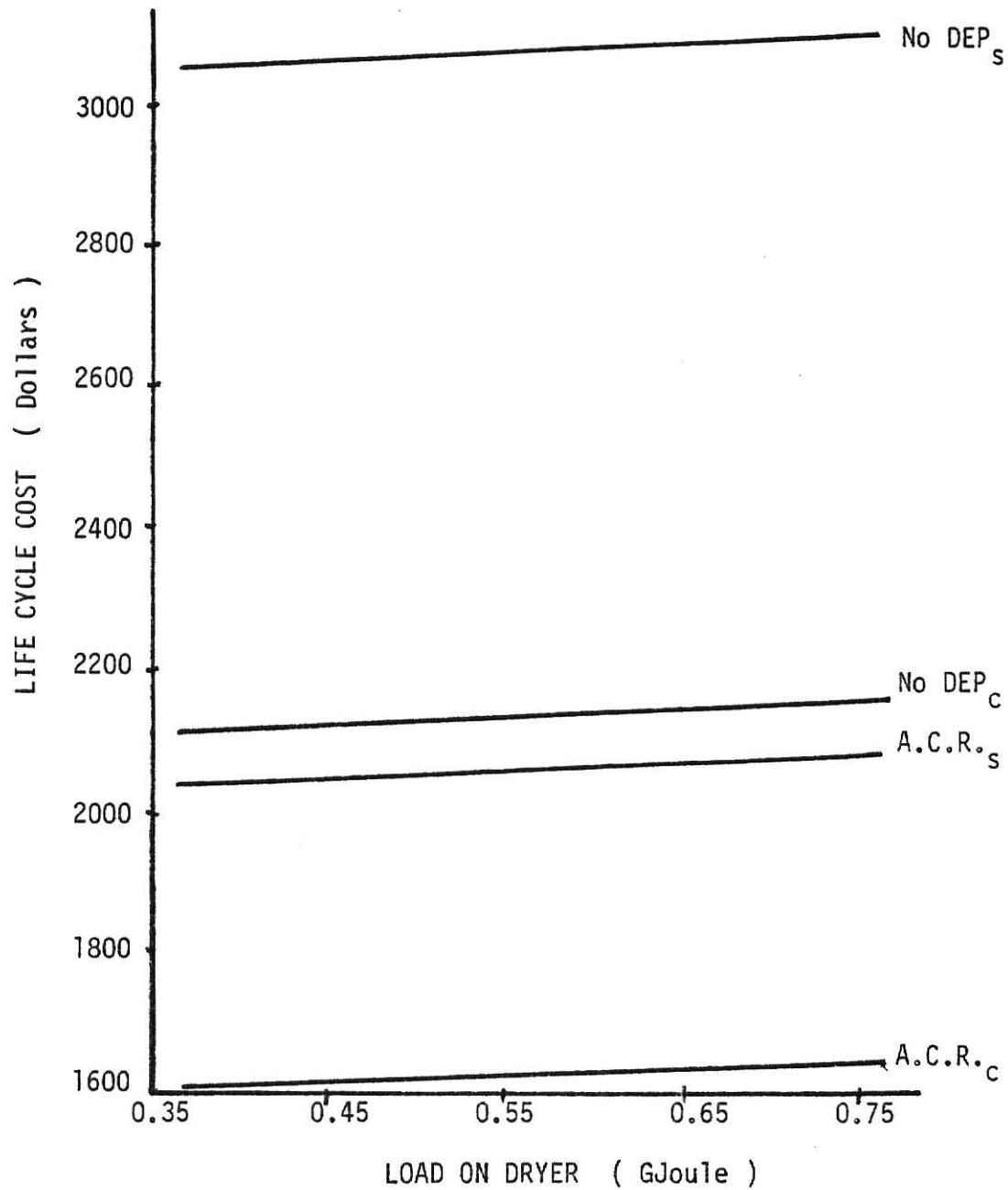
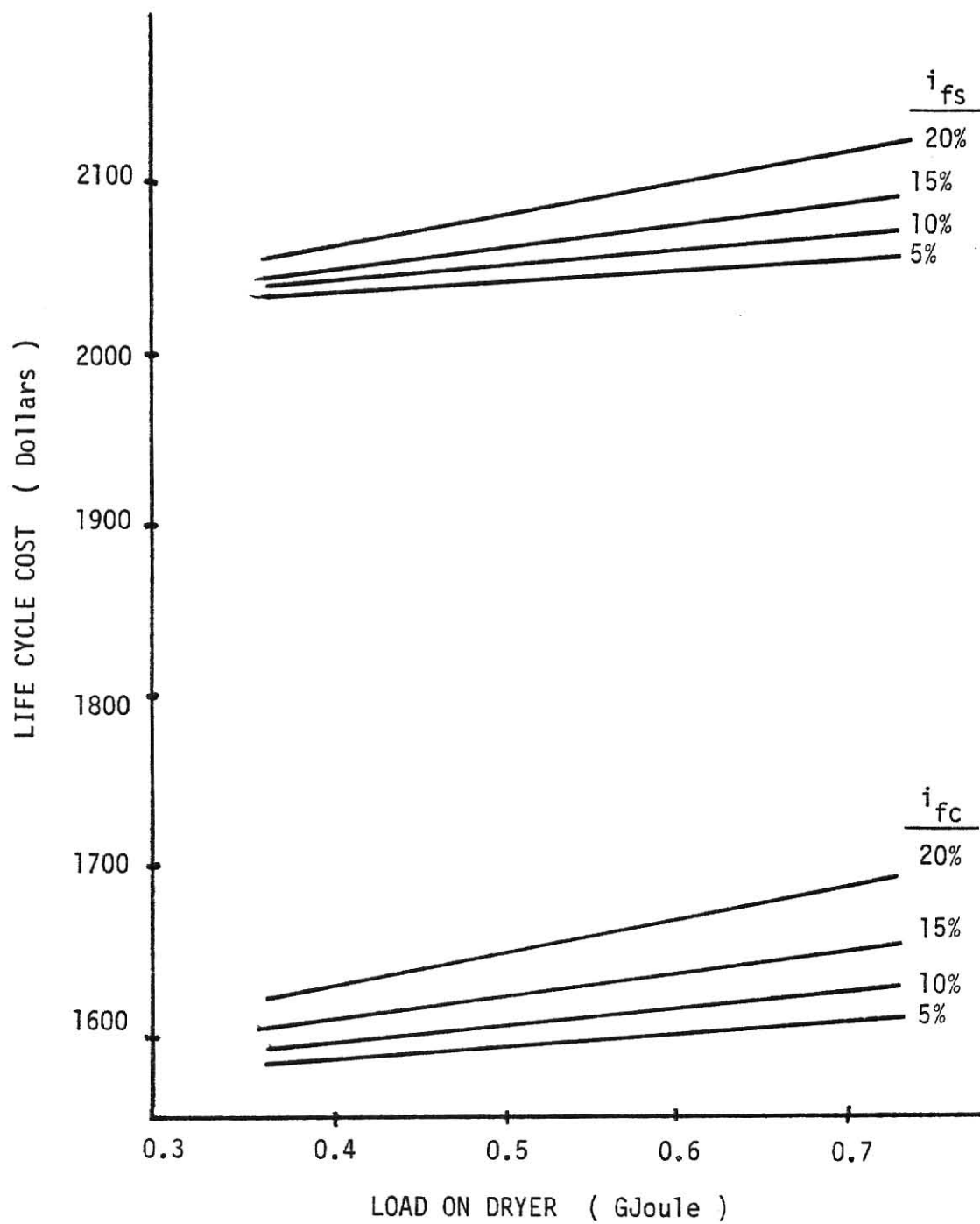


FIGURE 29
EFFECT OF FUEL INFLATION RATE
ON LIFE CYCLE COST



to illustrate the basic effect of N_e on the life cycle cost. The life cycle cost increased both in value and in rate of increase with the change in load, when the length of the economic analysis increased as shown in Figure 30.

This makes sense, since the present day (life cycle) cost of operating the system would increase if operated for a longer period of time. However, the annual cost of the system decreases, as the system lifetime increases as shown in Figure 31.

MARKET DISCOUNT RATE: Lower investment rates of return cause both P_1 and P_2 to increase so that the life cycle cost increases, and the rate of change in life cycle cost, increase with the load as shown in Figure 32.

FIGURE 30
EFFECT OF ECONOMIC ANALYSIS TERM
ON LIFE CYCLE COST

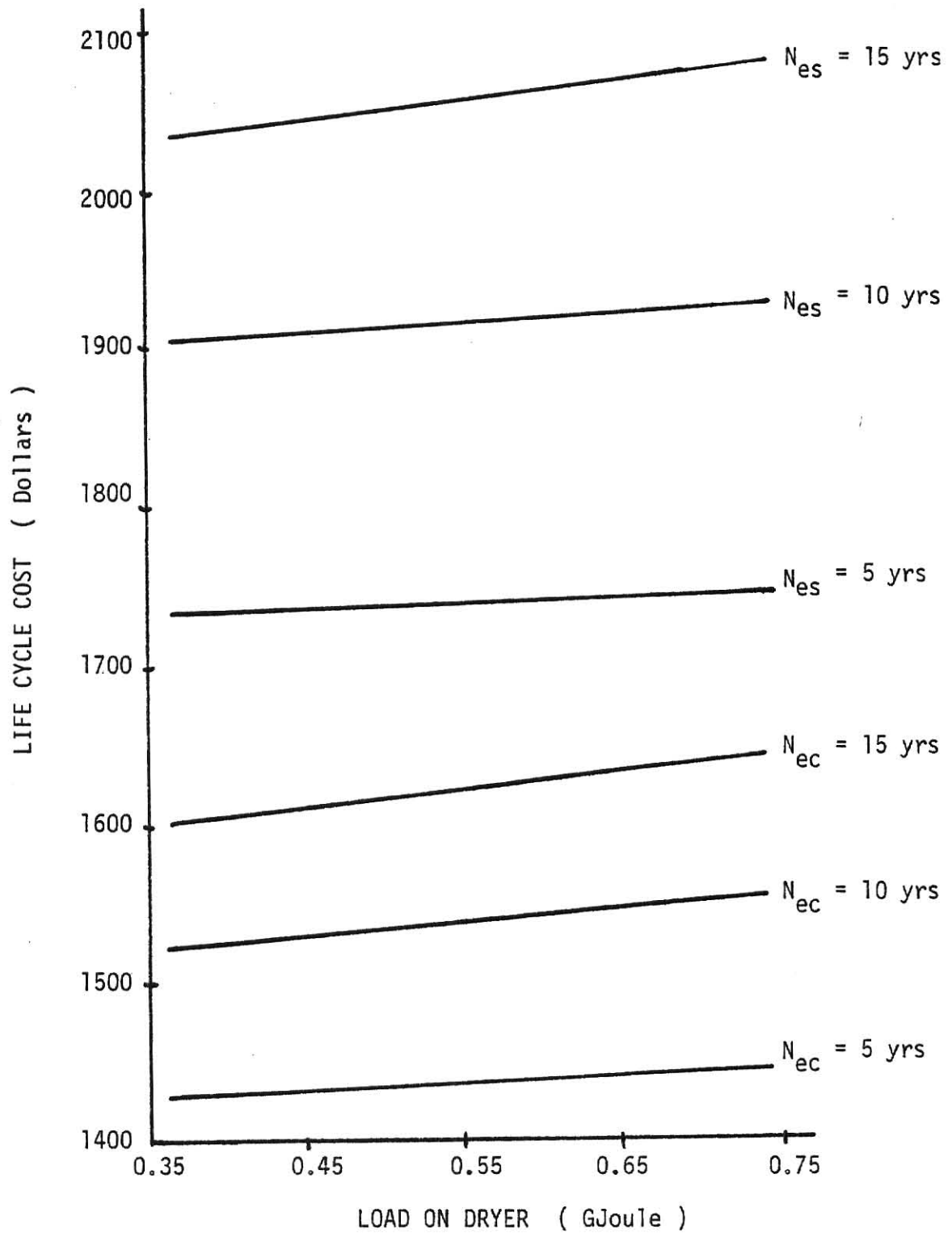


FIGURE 31
ANNUAL COST AFFECTED BY
ECONOMIC ANALYSIS TERM

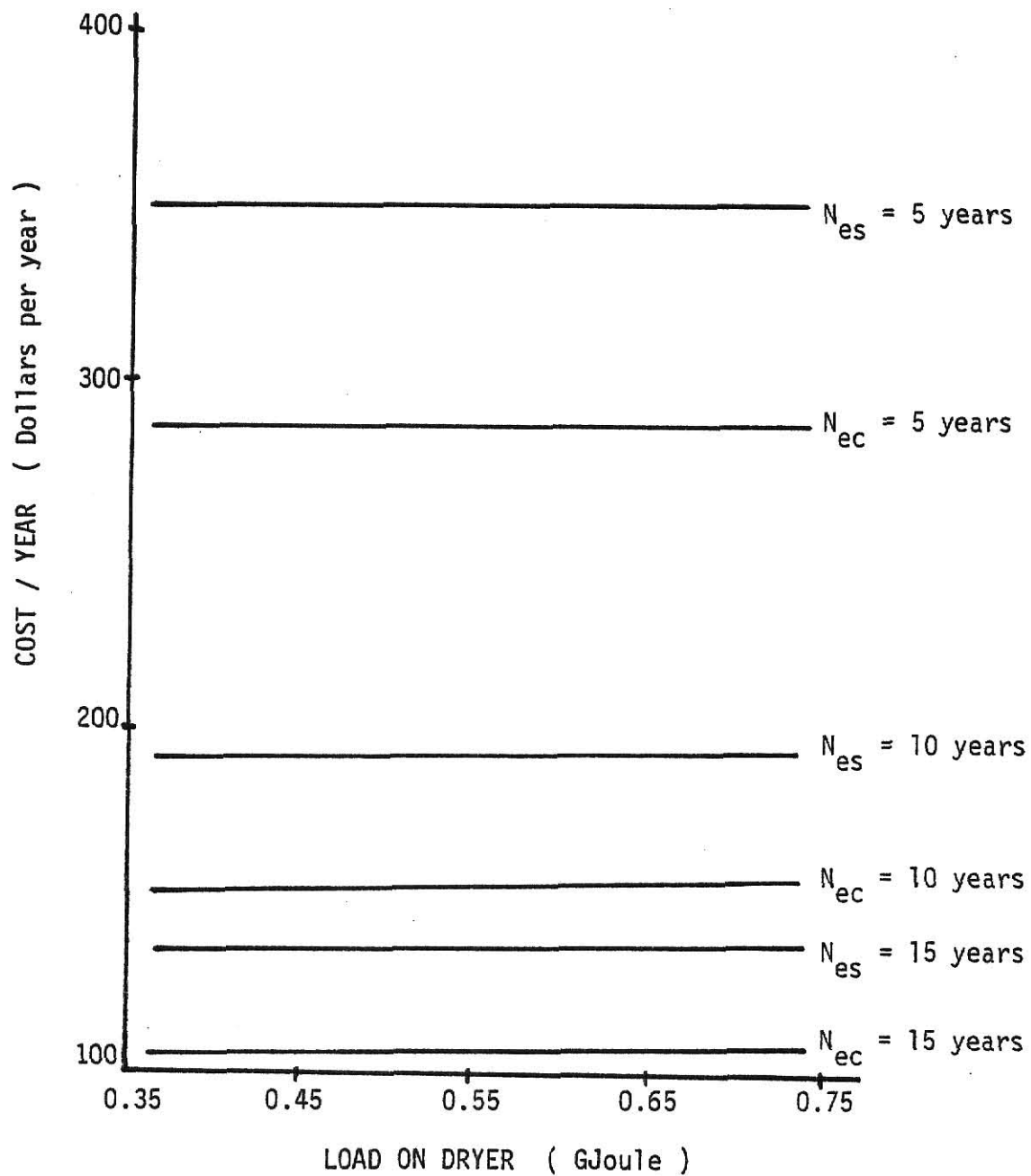
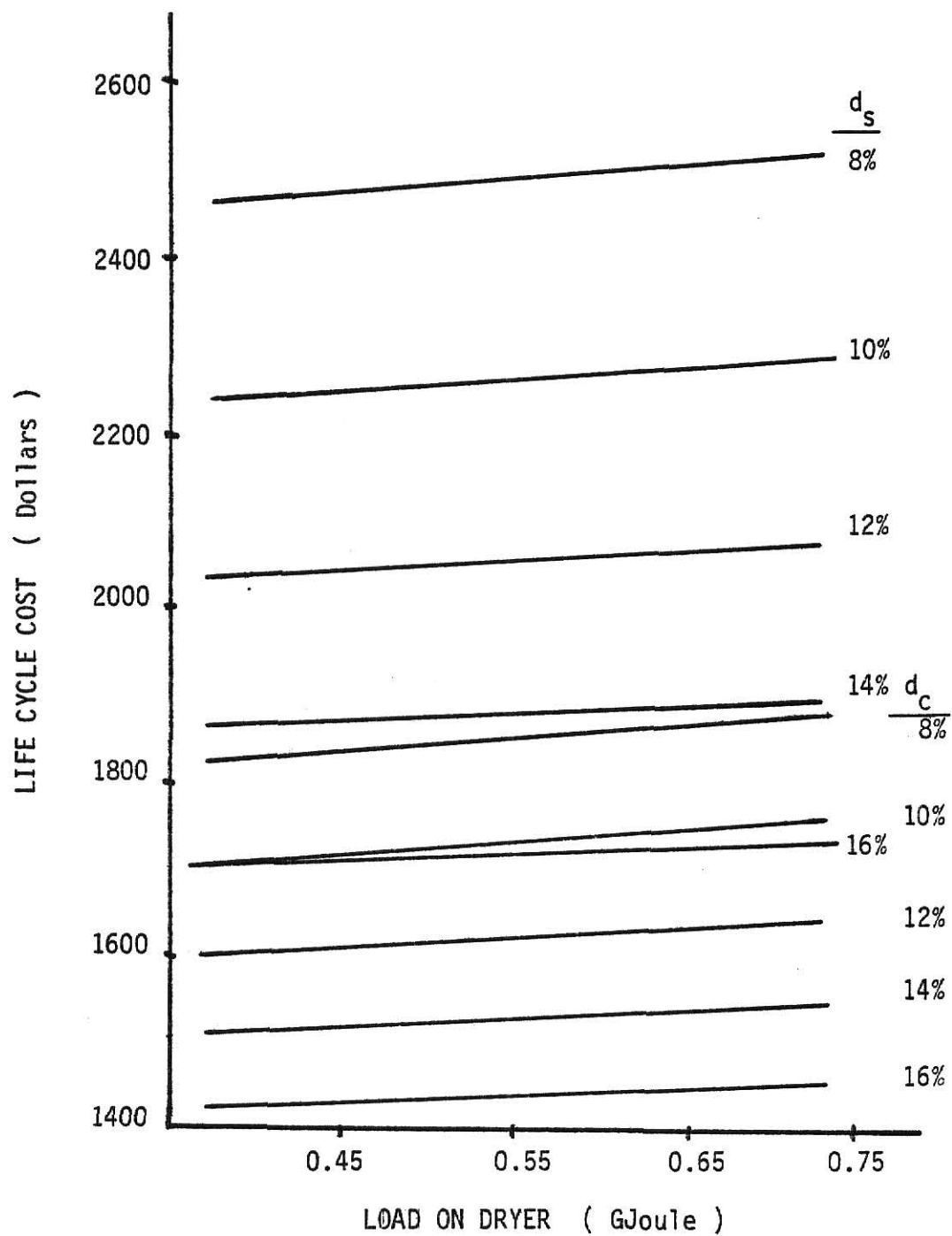


FIGURE 32

EFFECT OF MARKET DISCOUNT RATE
ON LIFE CYCLE COST



CHAPTER VII

SUMMARY AND CONCLUSIONS

The computer model developed incorporates:

1. Hourly solar radiation and meteorological data from the Typical Meteorological Year computer tape.
2. A computer model developed previously by other graduate students which predicts the hourly performance of a solar energy powered sorption dehumidification system.
3. An economic analysis for the operation of the system for several days.

Since the performance of the system depends on the location and time of year, and the economic analysis depends on the particular use of the system, a survey of several potential applications is given.

The Economic Analysis computer model predicts that favorable economic conditions for the Solar Energy Powered Sorption Dehumidification System include a healthy federal tax credit, low market interest rate, low mortgage downpayment, depreciation with the accelerated cost recovery method, longer term for the economic analysis and a high market discount rate.

CHAPTER VIII

RECOMMENDATIONS FOR FURTHER STUDY

There are enough recommendations for future research to keep many a graduate student busy. Among them are:

1. Incorporate a desiccant storage system as an option into the present economic analysis model, as described on page 42.
2. Study the effect of various dehumidifier operating conditions on the life cycle cost of the solar energy system, i.e. air mass flow rate, solar collector size, etc.
3. Search out the potentially feasible life cycle costs for different applications in different locations with typical economic analysis parameters.
4. Develop an accurate method to predict the amount of moisture removed from a commodity in the dryer, as described on page 38.
5. Build a pilot plant combining the dehumidification and drying systems to obtain experimental verification of the model.
6. Develop a design computer model to size the Solar Energy Powered Sorption Dehumidification System.

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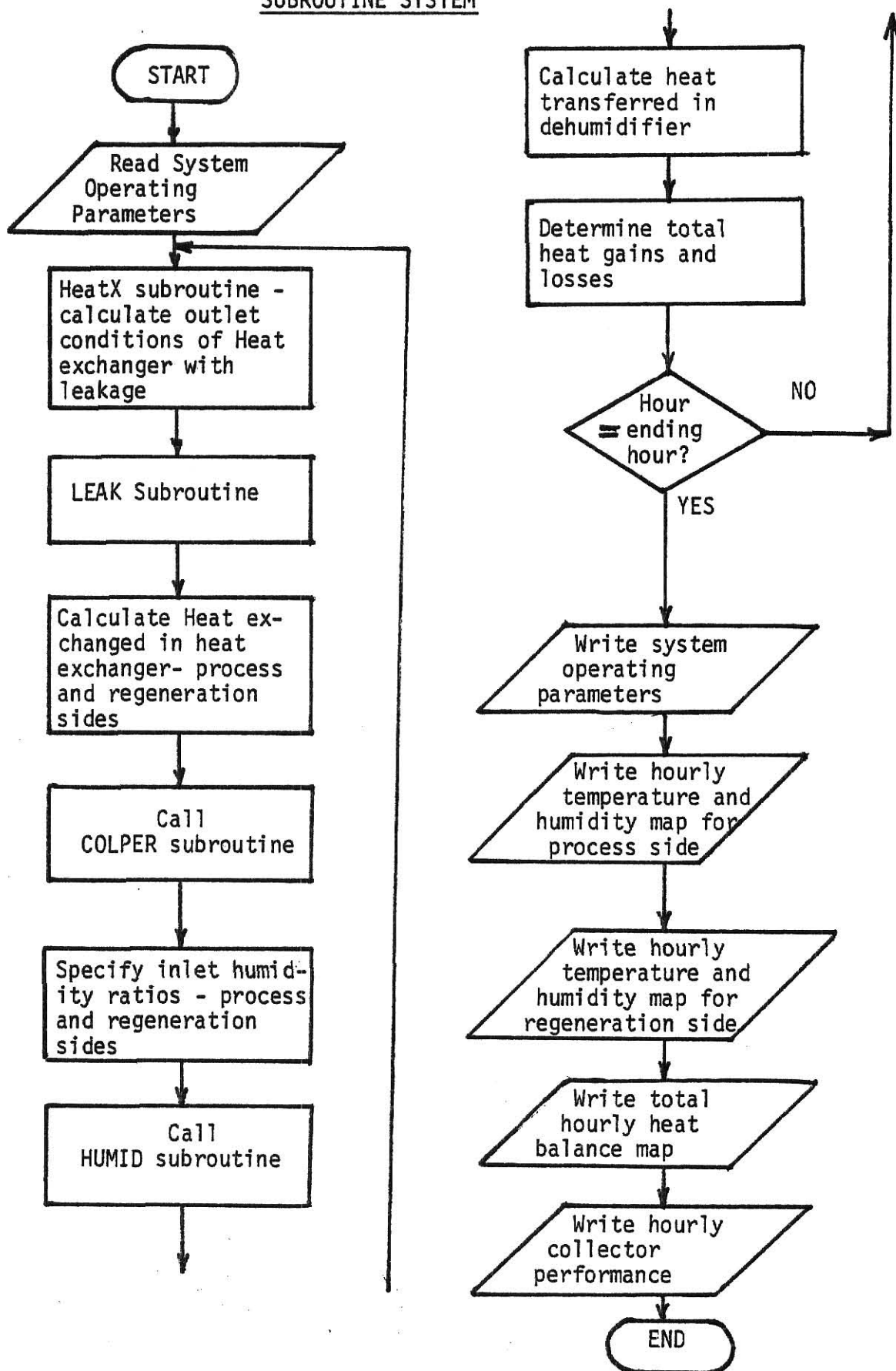
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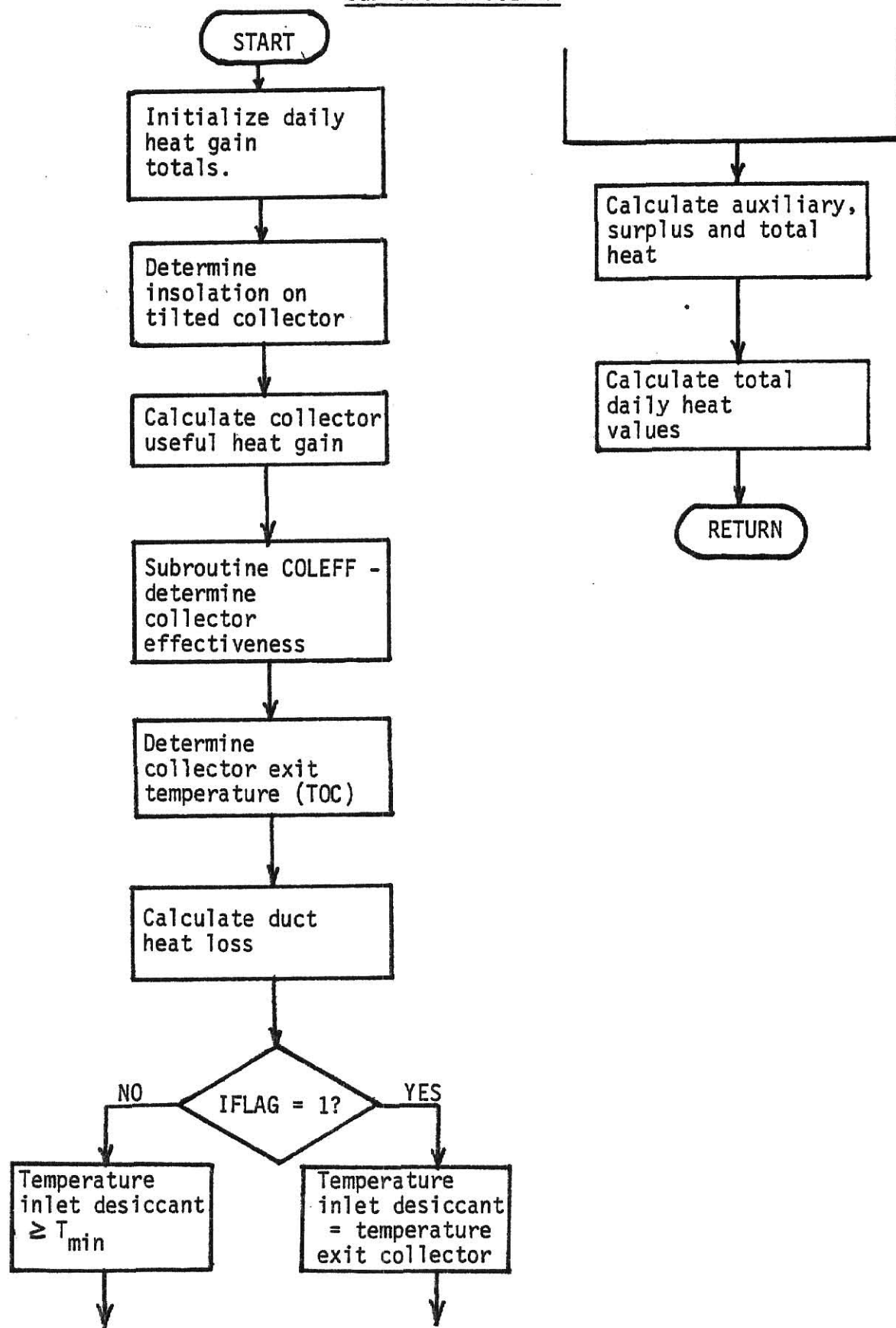
APPENDIX A

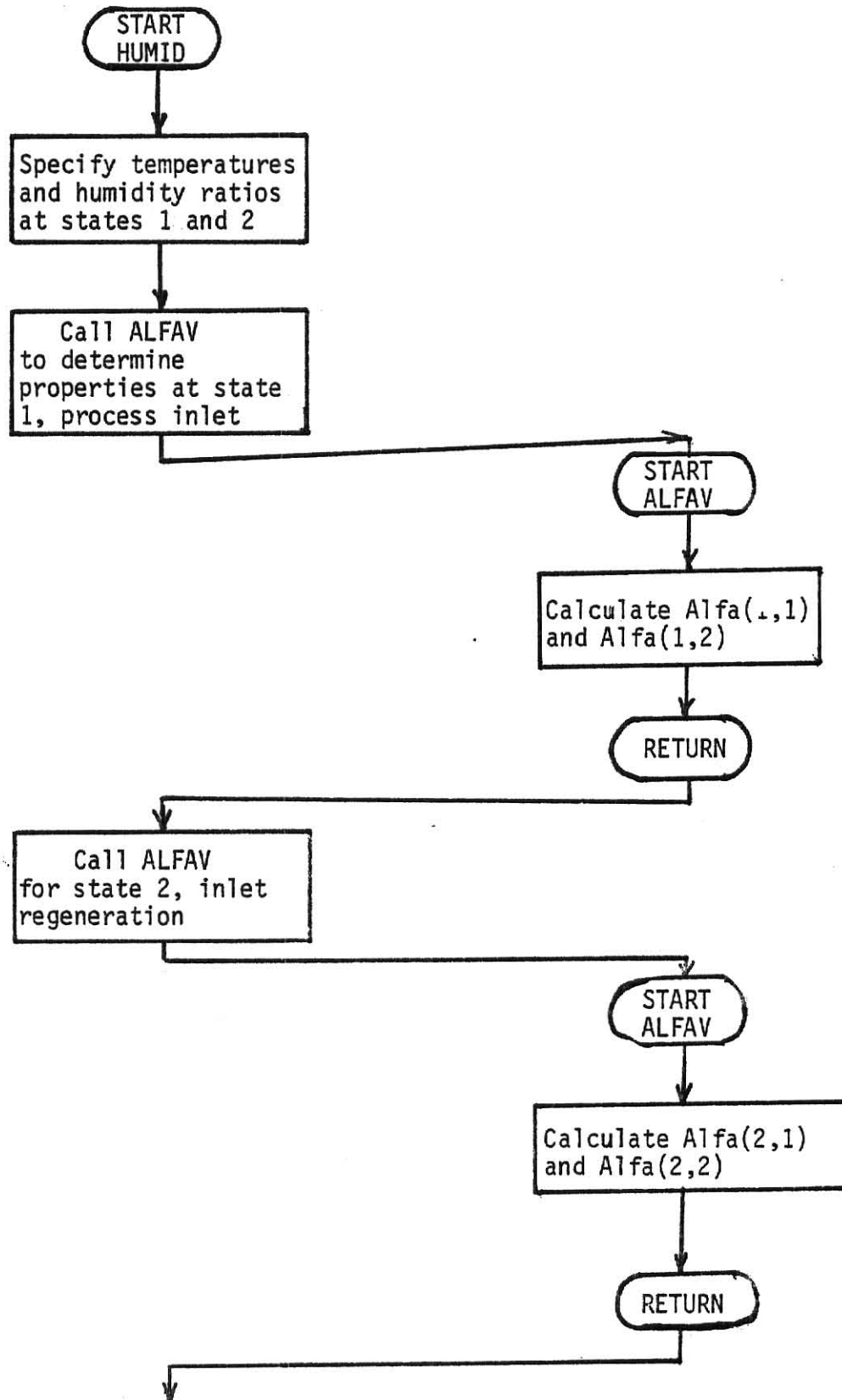
SUBROUTINE SYSTEM FLOWCHART

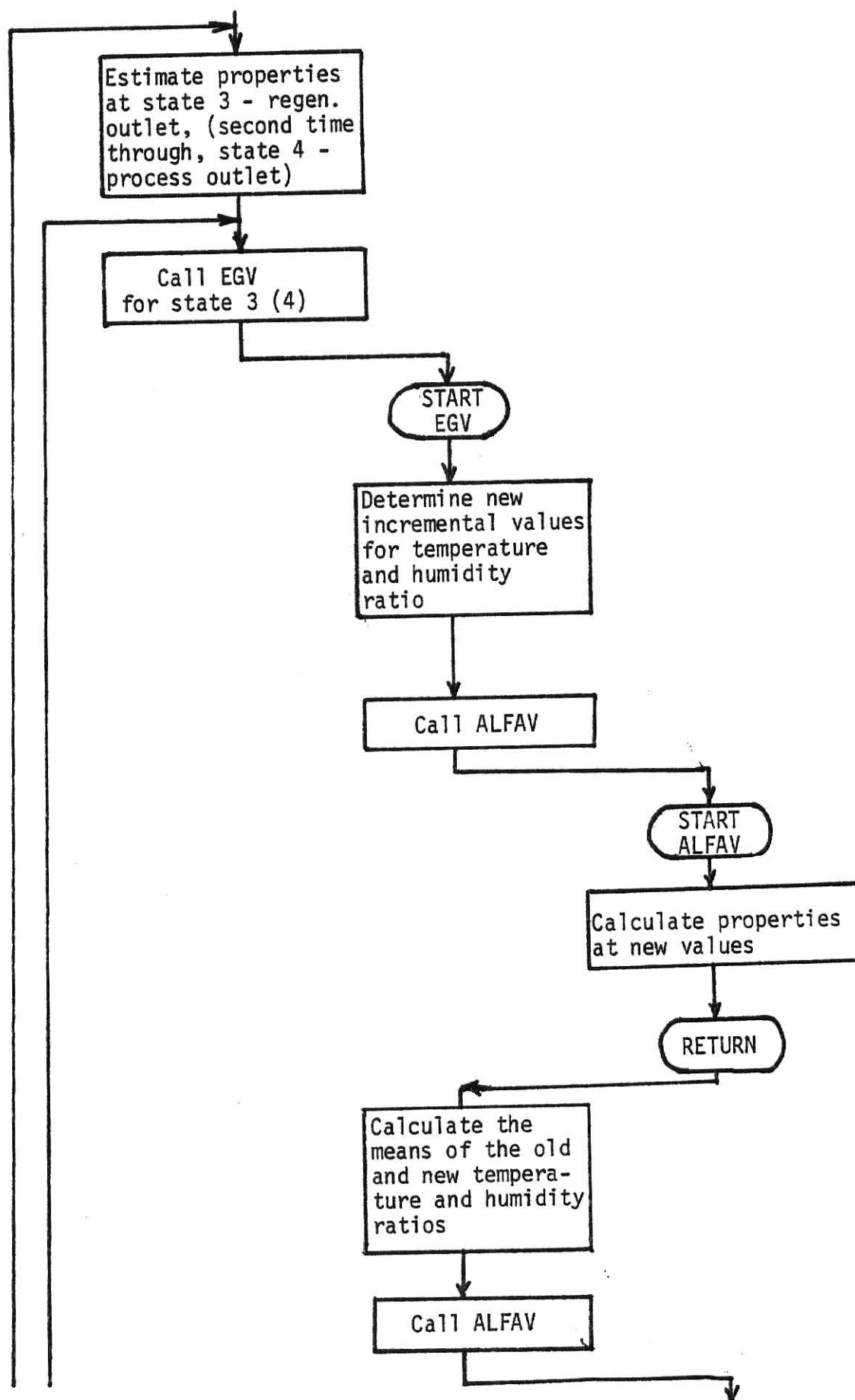
This appendix contains the flowchart for the SYSTEM subroutine, and supporting subroutines COLPER, HUMID, EGV and ALFAV. This is the modified version of the computer model developed by Singer and enhanced by Ananth.

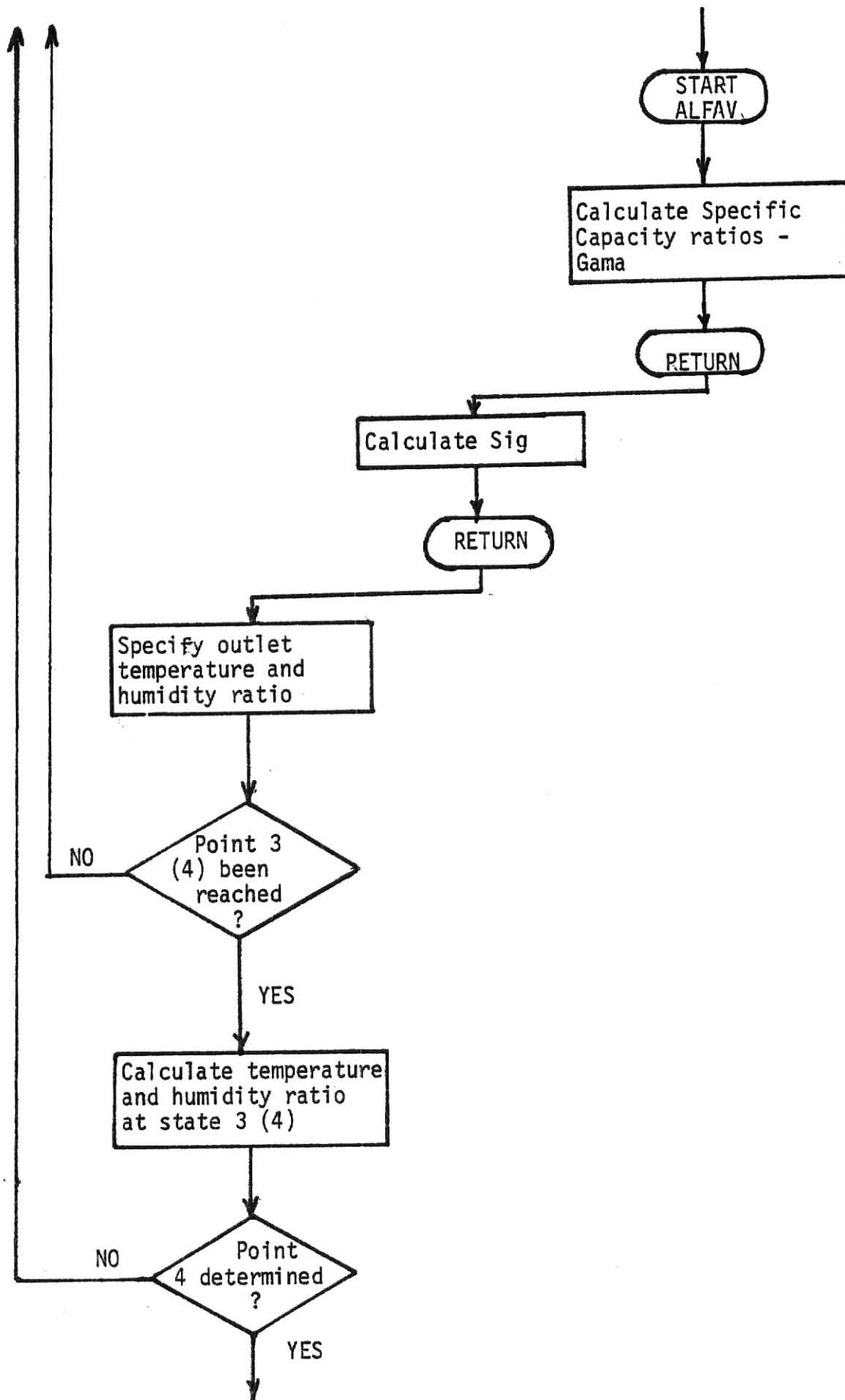
SUBROUTINE SYSTEM

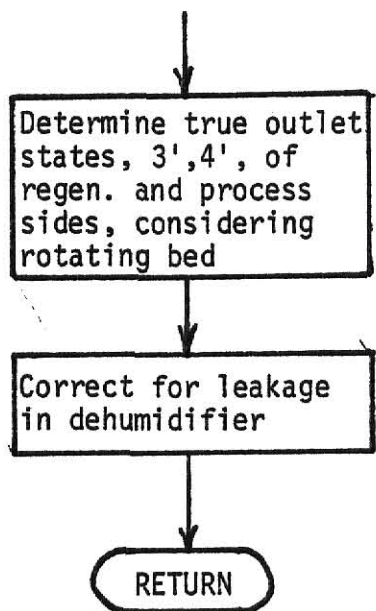


Subroutine COLPER

Subroutines HUMID, EGV and ALFAV



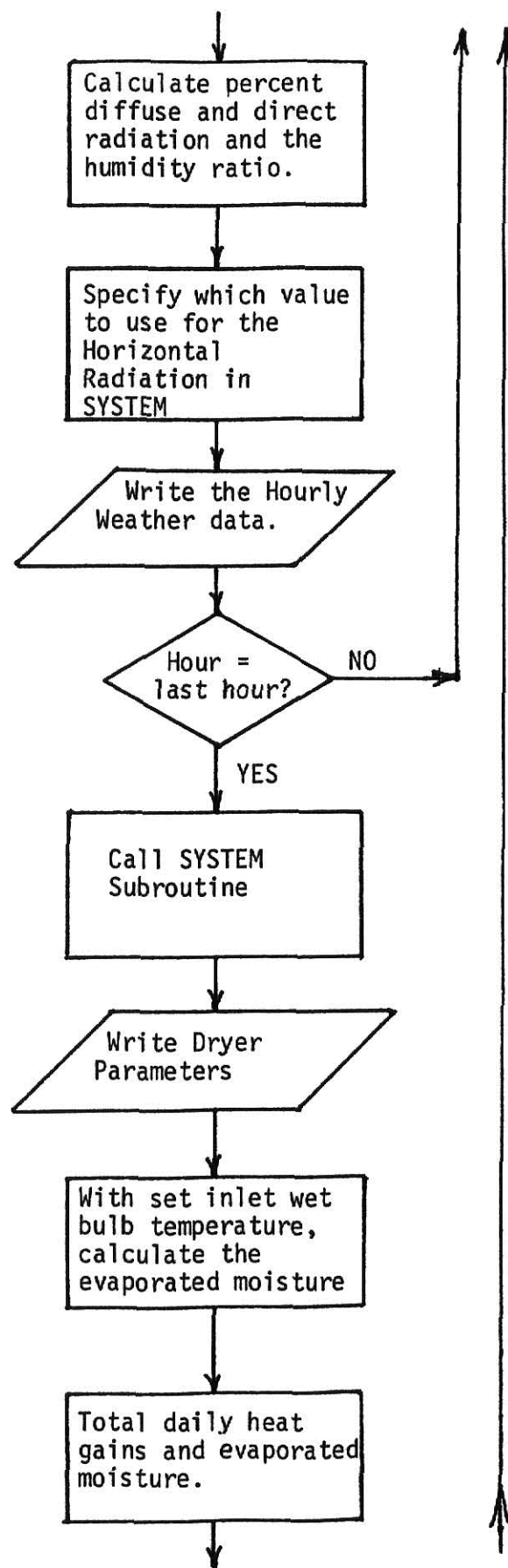
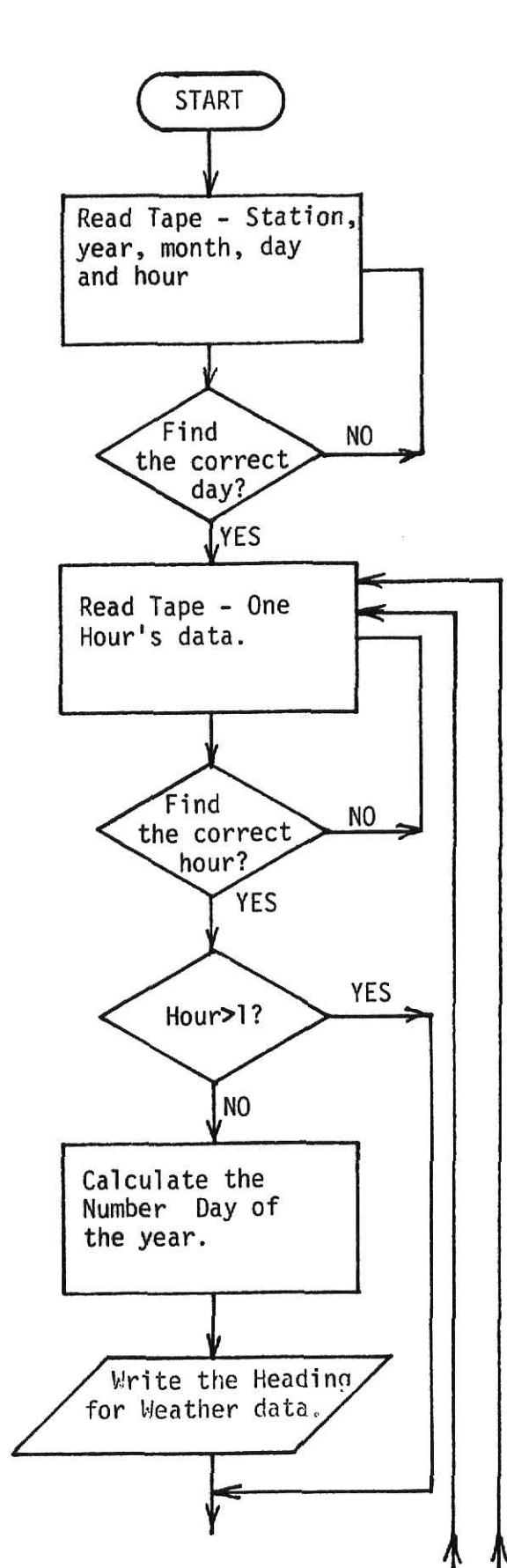


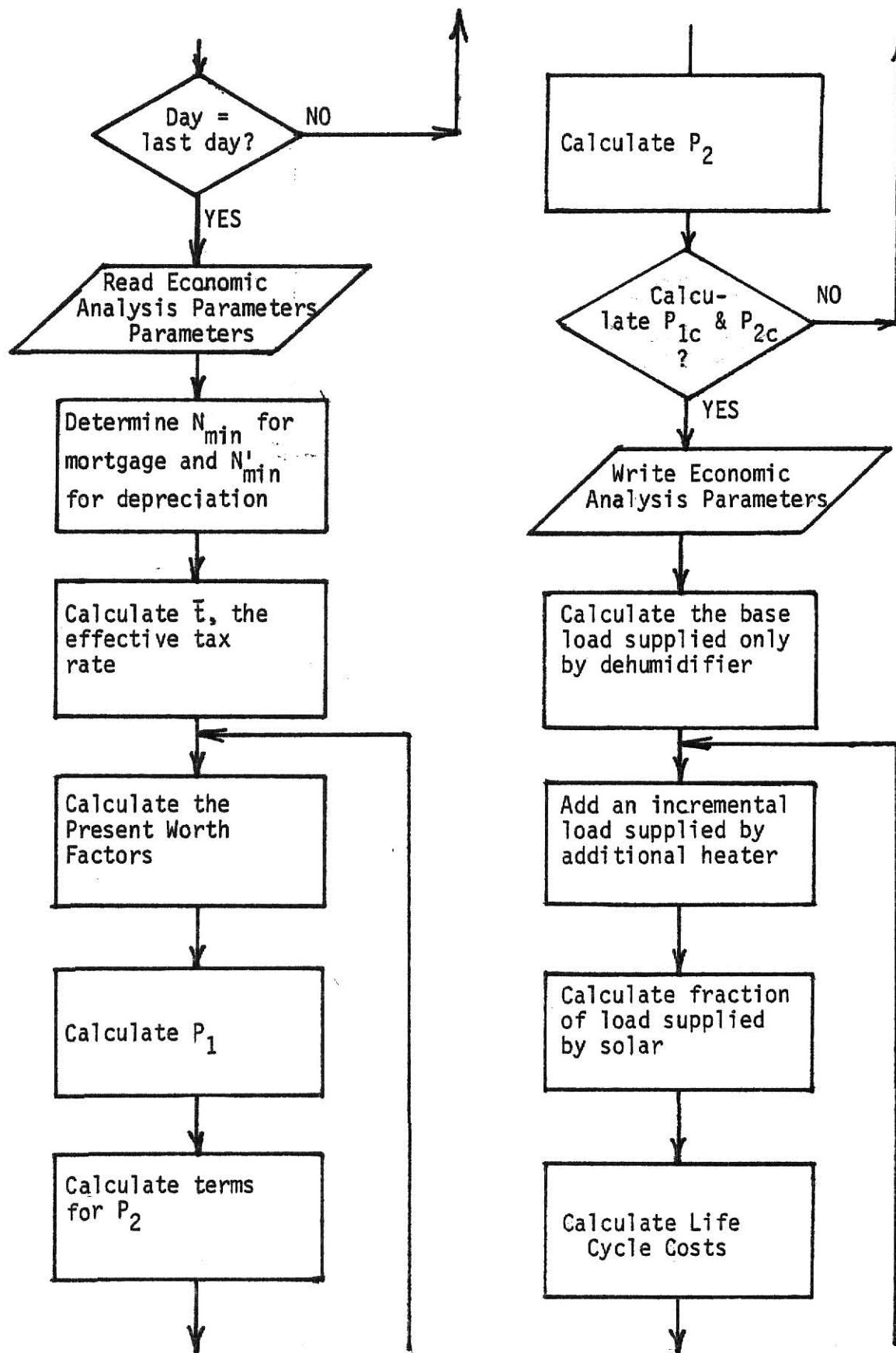


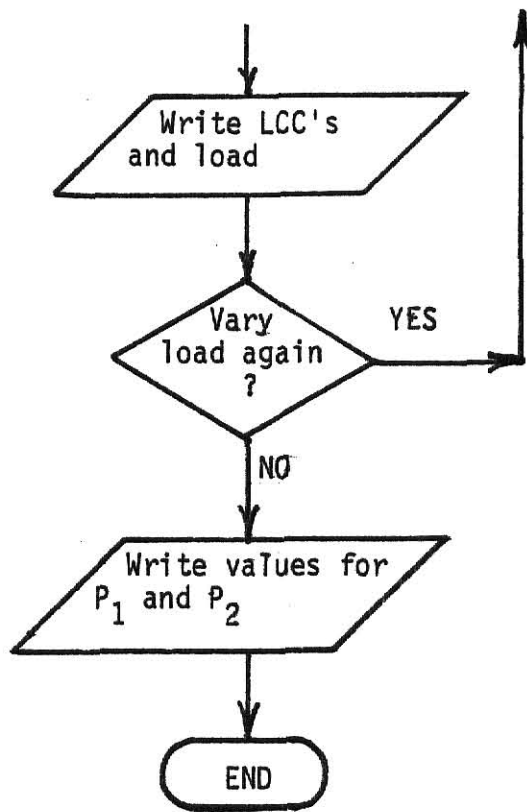
APPENDIX B

ENTIRE COMPUTER MODEL FLOWCHART

The flowchart listed here is the integrated version of the Weather Data and Economic Analysis computer models, which calls subroutine SYSTEM. To conserve space, two lines are placed side by side on each page of this appendix.







APPENDIX C

WEATHER DATA COMPUTER PROGRAM AND TYPICAL RESULTS

This appendix lists the program which reads the TMY weather data tape and calculates the ambient humidity ratio for Omaha, Nebraska during the month of October. The arrays allow storage for thirty days with 24 hours per day. The JCL required for execution in Fortran is given. Two days during that time period of typical output are listed.

```

/*ROUTE PRINT VM_---
/*TAPE9
// EXEC FORTQCLG,PARM=NOMAP
//FORT.SYSIN DD *
C      NOMENCLATURE FOR THE WEATHER PORTION OF PROGRAM FOLLOWS
C      CLK=CLOCK HOUR UNDER CONSIDERATION
C      DAY=DAY UNDER CONSIDERATION
C      DAYB=BEGINNING DESIRED DAY OF STUDY
C      DAYP=PROGRAM AID TO CONTROL READING THE TAPE. AFTER ONE DAY IS
C      READ FOR THE DESIRED NUMBER OF HOURS, THE NEXT READING IS FOR
C      THE NEXT DAY.
C      DBT=DRY BULB TEMPERATURE
C      DEND=DESIRED ENDING DAY OF STUDY
C      DIFR=DIFFUSE RADIATION
C      DIRR=DIRECT RADIATION
C      DPT=DEW POINT TEMPERATURE
C      ENCR=ENGINEERING CORRECTED RADIATION
C      EXTR=EXTRATERRESTRIAL RADIATION
C      HRB=DESIRED BEGINNING HOUR OF STUDY
C      HREND=DESIRED ENDING HOUR OF STUDY
C      I=DAY UNDER CALCULATION
C      IDATE=NUMBER DAY OF THE YEAR
C      IHR=SOLAR HOUR UNDER CONSIDERATION
C      ILEAP=PROGRAM AID TO DETERMINE THE NUMBER DAY OF THE YEAR
C      J=HOUR UNDER CALCULATION
C      KT=HOURLY CLEARNESS INDEX
C      MEND=DESIRED ENDING MGNTH OF STUDY
C      MINS=MINUTES OF SUNSHINE
C      MO=MONTH UNDER CONSIDERATION
C      MOB=DESIRED BEGINNING MGNTH OF STUDY
C      MOTEST=PROGRAM AID TO DETERMINE THE NUMBER DAY OF THE YEAR
C      ND=NUMBER OF DAYS INCLUDED IN STUDY
C      NDAY=NUMBER DAY OF THE YEAR
C      NH=NUMBER OF HOURS INCLUDED IN STUDY
C      OBSR=OBSERVED RADIATION
C      PERB=PERCENTAGE OF BEAM RADIATION BASED ON EXTRATERRESTRIAL
C      RADIATION
WEA00010
WEA00020
WEA00030
WEA00040
WEA00050
WEA00060
WEA00070
WEA00080
WEA00090
WEA00100
WEA00110
WEA00120
WEA00130
WEA00140
WEA00150
WEA00160
WEA00170
WEA00180
WEA00190
WEA00200
WEA00210
WEA00220
WEA00230
WEA00240
WEA00250
WEA00260
WEA00270
WEA00280
WEA00290
WEA00300
WEA00310
WEA00320
WEA00330
WEA00340
WEA00350
WEA00360
WEA00370

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C PERD=PERCENTAGE OF DIFFUSE RADIATION BASED ON EXTRATERRESTRIAL
C RADIATION
C PSEA=ATMOSPHERIC PRESSURE AT SEA LEVEL
C PSTN=ATMOSPHERIC PRESSURE AT THE STATION
C SKY=SKY CONDITIONS, TYPE AND EXTENT OF CLOUD COVER
C SNOW=INDICATOR OF SNOW GROUND COVERAGE
C STDYR=STANDARD YEAR RADIATION
C STN=STATION NUMBER UNDER CONSIDERATION
C WEAT=WEATHER CONDITIONS, SNCH, RAIN, MCNSCCN, ETC.
C WDIR=WIND DIRECTION
C WVEL=WIND VELOCITY
C YR=YEAR UNDER CONSIDERATION
C
C MAIN PROGRAM STARTS HERE
C DIMENSION DBT(30,24),DPT(30,24),IHR(30,24),H(30,24),IDATE(30),
C XPERB(30,24),PERD(30,24),REFLEC(30,24),WDIR(30,24),WVEL(30,24)
C X,WAM(30,24),PSTN(30,24),PSEA(30,24),OBSR(30,24),EXTR(30,24),
C XSTDYR(30,24),BETA(30,24),BETAF(30,24),PHV(30,24),ENCR(30,24)
C X,PSTNC(30,24)
C
C THE FOLLOWING 4 STATEMENTS DECLARE VARIABLES FOR THE WEATHER
C DATA PORTION OF THE PROGRAM
C
C INTEGER DAYS,DEND,HRB,HREND,DAYF
C INTEGER STN,YR,DAY,CLK(30,24),DIRR(30,24),WDIR,WVEL,DIFR(30,24),
C X WEAT(30,24),
C REAL KT(30,24)
C
C 10 FORMAT('1',//,T40,'***** TMY WEATHER DATA FOR OMAHA, NEBRASKA *****')
C
C 11 FORMAT('1',T6,I2,'/',1X,I2,'/',I2,T99,'STATION NO.',I5)
C 12 FORMAT('0',T6,'NUMBER DAY OF THE YEAR =',I3//)
C 13 FORMAT('0',T6,'TIME',T24,'RADIATION',T49,'SKY',T69,'PRESSURE',T85)
C X,'TEMPERATURE',T111,'WIND',T120,'SNOW')
C 14 FORMAT('1',SOLAR CLOCK',3X,EXT-T',T23,'ENG-C',T31,'BEAM',T37,'DWEAOC680
C XIFFUSE',T49,'COND',T56,'WEATHER',T67,'SEA-L',T77,'STN',T85,'D.B.'
C X,T93,'DEW',T99,'HUMIDITY',T109,'DIR',T115,'VEL',T120,'COVER')
C 15 FORMAT('1',I2,'00',2X,I4,4X,F5.0,2X,F5.0,2X,2(F6.4,2X),2X,I5,2X,
C X I8,3X,2(F7.2,1X),2X,2(F5.1,2X),2X,F7.5,2X,I3,1X,I4,5X,I1)
C 17 FORMAT('1',11('1',3X,2(5('1',2X),6('1',1X,7('1',5X,4('1',
C X3X,7('1',4X,14('1',4X,11('1',3X,8('1',2X,9('1',2X,5('1',
C 20 FORMAT(6I5)

```

```

WEA00380
WEA00390
WEA00400
WEA00410
WEA00420
WEA00430
WEA00440
WEA00450
WEA00460
WEA00470
WEA00480
WEA00490
WEA00500
WEA00510
WEA00520
WEA00530
WEA00540
WEA00550
WEA00560
WEA00570
WEA00580
WEA00590
WEA00600
WEA00610
WEA00620
WEA00630
WEA00640
WEA00650
WEA00660
WEA00670
WEA00680
WEA00690
WEA00700
WEA00710
WEA00720
WEA00730
WEA00740
WEA00750

```

```

1 FORMAT(I5,I2,I2,I2,I2,I2)
2 FORMAT(I5,I2,I2,I2,I2,2X,I4,F4.0,1X,I4,I5,10X,F5.0,F5.0,
XF5.0,10X,I2,6X,I5,4X,I8,2F5.2,2F4.1,I3,I4,4X,I1)
C SPECIFY THE DAYS, MCNTH AND HOURS DESIRED BY USER
READ (5,20) MOB, DAYB, MEND, DEND, HRB, HREND
ND=DEND-DAYB+1
NH=(HREND-HRB)+1
DAYP=DAYB
DAYF=DAYB-1
IZ=24
C TAPE IS READ BEGINNING JAN 1 UNTIL THE DESIRED DATES ARE FOUND.
50 READ (3,1)STN,YR,MO,DAY,MHR
IF(MO.LT.MOB) GO TO 50
IF(DAY.LT.DAYF) GO TO 50
IF(MHR.LT.IZ) GO TO 50
DO 80 I=1,ND
C THE SET OF DATA IS READ FOR EACH HOUR AT ONE WHOOSH, HOUR
C CONTROL IS REQUIRED.
DO 81 J=1,NH
40 READ (3,2)STN,YR,MO,DAY,IHR(I,J),CLK(I,J),EXTR(I,J),DIRR(I,J),DIRFWEA00950
X(I,J),OBSR(I,J),ENCR(I,J),STDYR(I,J),MINS(I,J),SKY(I,J),WEAT(I,J),WEA00960
XPSEA(I,J),PSTN(I,J),DBT(I,J),DPT(I,J),WDIR(I,J),WVEL(I,J),
XSNOW(I,J)
70 IF (IHR(I,J).LT.HRB) GO TO 40
IF (DAY.LT.DAYP) GO TO 40
IF (J.GT.1) GO TO 61
C CALCULATE THE NUMBER DAY OF THE YEAR, ALLOWING FOR LEAP YEARS
C ILEAP AND MOTEST ARE JUST TO AID IN THE PROGRAM MECHANISM
ILEAP=YR-48
IF (ILEAP/4*.NE.ILEAP) GO TO 913
MOTEST=MO-2
IF (MOTEST) 913,913,930
930 NCAY=DAY+1
GO TO 914
913 NDAY=DAY
914 GO TO (901,902,903,904,905,906,907,908,909,910,911,912),MC
901 IDATE(I)=NDAY
GO TO 960
WEA00760
WEA00770
WEA00780
WEA00790
WEA00800
WEA00810
WEA00820
WEA00830
WEA00840
WEA00850
WEA00860
WEA00870
WEA00880
WEA00890
WEA00900
WEA00910
WEA00920
WEA00930
WEA00940
WEA00950
WEA00960
WEA00970
WEA00980
WEA00990
WEA01000
WEA01010
WEA01020
WEA01030
WEA01040
WEA01050
WEA01060
WEA01070
WEA01080
WEA01090
WEA01100
WEA01110
WEA01120
WEA01130

```

```

902 IDATE(I)=NDAY+31
GO TO 960
903 IDATE(I)=NDAY+59
GO TO 960
904 IDATE(I)=NDAY+90
GO TO 960
905 IDATE(I)=NDAY+120
GO TO 960
906 IDATE(I)=NDAY+151
GO TO 960
907 IDATE(I)=NDAY+181
GO TO 960
908 IDATE(I)=NDAY+212
GO TO 960
909 IDATE(I)=NDAY+243
GO TO 960
910 IDATE(I)=NDAY+273
GO TO 960
911 IDATE(I)=NDAY+304
GO TO 960
912 IDATE(I)=NDAY+334
960 CONTINUE
WRITE (6,10)
WRITE (6,11) MO, DAY, YR, STN
WRITE (6,12) IDATE(I)
WRITE (6,13)
WRITE (6,14)
WRITE (6,17)
61 CONTINUE
C THE TAPE STORES A '9' IF THE DATA WAS NOT RECORDED OR IF THE
C VALUE IS ZERO. THE FOLLOWING 'IF' STATEMENTS MAKES PRETTY OUTPUT
IF (EXTR(I,J).EQ.9999) EXTR(I,J)=0
IF (DIRR(I,J).EQ.9999) DIRR(I,J)=0
IF (DIFR(I,J).EQ.9999) DIFR(I,J)=0
IF (OBSR(I,J).EQ.9999) OBSR(I,J)=0
IF (ENCR(I,J).EQ.9999) ENCR(I,J)=0
IF (STDYR(I,J).EQ.9999) STDYR(I,J)=0
IF (MINS(I,J).EQ.99) MINS(I,J)=0
WEA01140
WEA01150
WEA01160
WEA01170
WEA01180
WEA01190
WEA01200
WEA01210
WEA01220
WEA01230
WEA01240
WEA01250
WEA01260
WEA01270
WEA01280
WEA01290
WEA01300
WEA01310
WEA01320
WEA01330
WEA01340
WEA01350
WEA01360
WEA01370
WEA01380
WEA01390
WEA01400
WEA01410
WEA01420
WEA01430
WEA01440
WEA01450
WEA01460
WEA01470
WEA01480
WEA01490
WEA01500
WEA01510

```

```

IF (SKY(I,J).EQ.99999) SKY(I,J)=0
IF (WEAT(I,J).EQ.99999999) WEAT(I,J)=0
IF (SNOW(I,J).EQ.1) REFLEC(I,J)=0.7
IF (SNOW(I,J).EQ.0) REFLEC(I,J)=0.2
IF (SNOW(I,J).NE.9) GO TO 62
SNOW(I,J)=0
REFLEC(I,J)=0.0
62 CONTINUE
IF (EXTR(I,J).EQ.0) GO TO 44
KT(I,J)=ENCR(I,J)/EXTR(I,J)
IF (KT(I,J).LE.0.35) PERD(I,J)=1.0-0.249*KT(I,J)
IF (KT(I,J).GE.0.75) PERD(I,J)=0.117
IF ((KT(I,J).GT.0.35).AND.(KT(I,J).LT.0.75)) PERD(I,J)=1.557-1.84*WEA01640
XKT(I,J)
PERB(I,J)=1.0-C-PERD(I,J)
GO TO 45
44 PERB(I,J)=0.0
PERD(I,J)=0.0
45 CONTINUE
C      CALCULATE AMBIENT HUMIDITY FROM DRY BULB AND STATIC
C      ATMOSPHERIC PRESSURE DATA USING ASHRAE METHOD
BETA(I,J)=374.12-DBT(I,J)
RETAF(I,J)=BETA(I,J)/100.
PWV(I,J)=218.167*10**(-BETA(I,J)/(DPT(I,J)+273.15))*(3.2437814+.586WEA01750
X826*BETAF(I,J)+0.011702379*BETAF(I,J)**3)/(1.+2187846*BETAF(I,J))WEA01760
X)
PSTNC(I,J)=PSTN(I,J)*.0098692
WAM(I,J)={.62198*PWV(I,J)}/{PSTNC(I,J)-PWV(I,J)}
WRITE (6,15) IHR(I,J),CLK(I,J),EXTR(I,J),ENCR(I,J),PERB(I,J),PERD
X(I,J),SKY(I,J),WEAT(I,J),PSEA(I,J),PSTN(I,J),CBT(I,J),DPT(I,J),
XWAM(I,J),WDIR(I,J),WVEL(I,J),SNOW(I,J)
81 CONTINUE
C      DAYP IS JUST A CONTROL FOR READING THE TAPE. AFTER ONE DAY ISWEA01830
C      READ FOR THE DESIRED NUMBER OF HOURS, THE NEXT READING IS FOR WEA01840
C      THE NEXT DAY. WEA01850
C      DAYP=DAY+1 WEA01860
80 CONTINUE WEA01870
STOP WEA01880
WEA01890

```


WEA01900
WEA01910
WEA01920
WEA01930
WEA01940

END
//GO.FT03F001 DD DSN=TMY,UNIT=TAPE1600,VOL=SER=9TMY03,LABEL=(26,,,IN),
// DCB=(RECFM=FB,LRECL=132,BLKSIZE=3168)
//GO.SYSIN DD *
10 1 10 30 7 17

***** TMY WEATHER DATA FOR GMAHA, NEBRASKA *****

STATION NC.94918

10/ 5/68

NUMBER DAY OF THE YEAR =279

| TIME SOLAR CLOCK | EXT-T | RADIATION | | SKY COND | WEATHER | PRESSURE | | TEMPERATURE | | HUMIDITY | WIND | | SNOW COVER |
|---------------------|-------|-----------|--------|-------------|---------|----------|-------|-------------|------|----------|------|-----|---------------|
| | | ENG-C | BEAM | | | SEA-L | STN | D.B. | DEW | | DIR | VEL | |
| 700 | 712 | 8. | 0.0076 | 0 | 0 | 101.35 | 97.81 | 10.9 | 7.0 | 0.00732 | 160 | 57 | 0 |
| 800 | 812 | 56. | 0.0119 | 0 | 0 | 101.37 | 97.82 | 10.2 | 8.0 | 0.00741 | 160 | 51 | 0 |
| 900 | 912 | 169. | 0.0210 | 4600 | 1000010 | 101.38 | 97.83 | 9.4 | 8.9 | 0.00745 | 160 | 46 | 0 |
| 1000 | 1012 | 2689. | 0.0209 | 0 | 0 | 101.34 | 97.80 | 9.8 | 9.3 | 0.00766 | 163 | 44 | 0 |
| 1100 | 1112 | 3172. | 0.0193 | 0 | 0 | 101.31 | 97.76 | 10.2 | 9.6 | 0.00785 | 167 | 43 | 0 |
| 1200 | 1212 | 3423. | 0.0167 | 6000 | 1000010 | 101.27 | 97.73 | 10.6 | 10.0 | 0.00807 | 170 | 41 | 0 |
| 1300 | 1312 | 3423. | 0.0145 | 0 | 0 | 101.20 | 97.66 | 10.4 | 9.8 | 0.00796 | 170 | 46 | 0 |
| 1400 | 1412 | 3172. | 0.0147 | 0 | 0 | 101.12 | 97.60 | 10.2 | 9.6 | 0.00786 | 170 | 52 | 0 |
| 1500 | 1512 | 2689. | 0.0132 | 6000 | 1000010 | 101.05 | 97.53 | 10.0 | 9.4 | 0.00776 | 170 | 57 | 0 |
| 1600 | 1612 | 2005. | 0.0139 | 0 | 0 | 101.04 | 97.52 | 9.8 | 9.2 | 0.00766 | 167 | 48 | 0 |
| 1700 | 1712 | 1168. | 0.0064 | 0 | 0 | 101.02 | 97.50 | 9.6 | 9.1 | 0.00758 | 163 | 40 | 0 |

***** TMY WEATHER DATA FOR OMAHA, NEBRASKA *****

STATION NO.94918

10/ 6/68

NUMBER DAY OF THE YEAR =280

| TIME SOLAR CLOCK | EXT-T | RADIATION | | SKY COND | WEATHER | PRESSURE | | TEMPERATURE | | HUMIDITY | WIND | | SNOW COVER | |
|---------------------|-------|-----------|-------|-------------|---------|----------|--------|-------------|------|----------|---------|-----|---------------|-----|
| | | ENG-C | BEAM | | | DIFUSE | SEA-L | STN | D.B. | | DEW | DIR | | VEL |
| 700 | 711 | 247. | 139. | 0.4785 | 0.5215 | 0 | 101.12 | 97.57 | 8.7 | 7.4 | C.00692 | 323 | 31 | 0 |
| 800 | 811 | 1145. | 754. | 0.6547 | 0.3453 | 0 | 101.16 | 97.62 | 10.2 | 7.6 | 0.00732 | 327 | 36 | 0 |
| 900 | 911 | 1983. | 1492. | 0.8830 | 0.1170 | 0 | 101.21 | 97.66 | 11.7 | 7.8 | 0.00775 | 330 | 41 | 0 |
| 1000 | 1011 | 2667. | 2124. | 0.8830 | 0.1170 | 0 | 101.20 | 97.65 | 13.7 | 7.2 | 0.00810 | 323 | 48 | 0 |
| 1100 | 1111 | 3150. | 2518. | 0.8830 | 0.1170 | 0 | 101.18 | 97.64 | 15.8 | 6.7 | C.00853 | 317 | 55 | 0 |
| 1200 | 1211 | 3401. | 2697. | 0.8830 | 0.1170 | 0 | 101.17 | 97.63 | 17.8 | 6.1 | C.00892 | 310 | 62 | 0 |
| 1300 | 1311 | 3401. | 2664. | 0.8830 | 0.1170 | 0 | 101.11 | 97.58 | 18.3 | 5.5 | C.00888 | 307 | 50 | 0 |
| 1400 | 1411 | 3150. | 2448. | 0.8830 | 0.1170 | 0 | 101.06 | 97.54 | 18.9 | 5.0 | C.00890 | 303 | 38 | 0 |
| 1500 | 1511 | 2667. | 2014. | 0.8830 | 0.1170 | 0 | 101.00 | 97.49 | 19.4 | 4.4 | C.00886 | 300 | 26 | 0 |
| 1600 | 1611 | 1983. | 1370. | 0.7142 | 0.2858 | 0 | 100.98 | 97.47 | 17.4 | 5.7 | C.00869 | 247 | 24 | 0 |
| 1700 | 1711 | 1145. | 571. | 0.3606 | 0.6394 | 0 | 100.96 | 97.45 | 15.3 | 7.0 | 0.00849 | 193 | 25 | 0 |

APPENDIX D

ECONOMIC ANALYSIS COMPUTER PROGRAM AND TYPICAL RESULTS

The computer program listed here determines the life cycle costs for a conventional dryer and a dryer utilizing the Solar Energy Powered Sorption Dehumidification System. The equipment costs were estimated at \$2000 for a conventional dryer, \$1000 for the Sorption system and \$300 per square meter for the collector. The price of fuel estimated propane at \$0.85 per gallon with an efficiency of 85%, which results in a delivered price of approximately \$12.00 per GJoule. One day of hourly station atmospheric pressures and dehumidifier outlet humidity ratio data are included with this example. Typical output include the commodity dryer parameters, one day of hourly dryer inlet and outlet conditions, the economic analysis parameters and the life cycle costs as they vary with the load.

| | | |
|---|--|----------|
| C | NGMENCLATURE LIST BEGINS HERE | EA 00010 |
| C | AMASSP=AIR MASS FLOW RATE THROUGH DEHUMIDIFIER - PROCESS SIDE | EA 00020 |
| C | CA=UNIT COST VARYING WITH SIZE OF SOLAR ENERGY SYSTEM | EA 00030 |
| C | CEC=EQUIPMENT COST - CONVENTIONAL | EA 00040 |
| C | CES=EQUIPMENT COST - SCLAR | EA 00050 |
| C | CF=COST OF FUEL, DOLLARS | EA 00060 |
| C | CFLAG=PROGRAMING FLAG TO DESIGNATE THE DIFFERENCE BETWEEN INCOME | EA 00070 |
| C | PRODUCING AND NON-INCOME PRODUCING SOLAR SYSTEMS | EA 00080 |
| C | CMC=MISCELLANEOUS COST - CONVENTIONAL | EA 00090 |
| C | CMIR= ANNUAL MORTGAGE INTEREST RATE - CONVENTIONAL | EA 00100 |
| C | DC=DOWNPAYMENT - CONVENTIONAL | EA 00110 |
| C | DEPC=DEPRECIATION TAX DEDUCTION - CONVENTICNAL | EA 00120 |
| C | DEPS=DEPRECIATION TAX DEDUCTION - SOLAR | EA 00130 |
| C | DIC=DEPRECIATION INFLATION RATE- USED FOR DECLINING BALANCE | EA 00140 |
| C | - CONVENTIONAL | EA 00150 |
| C | DIS=DEPRECIATION INFLATION RATE- USED FOR DECLINING BALANCE | EA 00160 |
| C | -SCLAR | EA 00170 |
| C | DMCC=DISCOUNTED VALUE OF MISC. COSTS - SOLAR (INSURANCE,ETC.) | EA 00180 |
| C | -CCNVENTICNAL | EA 00190 |
| C | DMCS=DISCOUNTED VALUE OF MISC. COSTS - SCLAR (INSURANCE,ETC.) | EA 00200 |
| C | -SCLAR | EA 00210 |
| C | DOPC=FLAG INDICATING WHICH DEPRECIATION SCHEDULE IS UTILIZED - | EA 00220 |
| C | CCNVENTICNAL | EA 00230 |
| C | DCPS=FLAG INDICATING WHICH DEPRECIATION SCHEDULE IS UTILIZED - | EA 00240 |
| C | SCLAR | EA 00250 |
| C | DRC=D, MARKET DISCOUNT RATE - CONVENTIONAL | EA 00260 |
| C | DRS=D, MARKET DISCOUNT RATE - SCLAR | EA 00270 |
| C | DRVC=DISCOUNTED RESALE VALUE - CONVENTICNAL | EA 00280 |
| C | DRVS=DISCOUNTED RESALE VALUE - SOLAR | EA 00290 |
| C | DS= DOWNPAYMENT - SOLAR | EA 00300 |
| C | EMCD= EQUILIBRIUM MOISTURE CONTENT OF WET GRAIN, DRY BASIS | EA 00310 |
| C | F=FRACTION OF HEAT SUPPLIED BY SOLAR, WITH RESPECT TO THE TOTAL | EA 00320 |
| C | HEAT SUPPLIED (1-F IS THE PERCENT AUXILIARY AND ADDITIONAL | EA 00330 |
| C | HEAT SUPPLIED) | EA 00340 |
| C | FI=FUEL INFLATION RATE | EA 00350 |
| C | FTR=FEDERAL TAX RATE | EA 00360 |
| C | GC= GRAIN CCNSTANT C | EA 00370 |
| C | GEMT=GRAND TCTAL EVAPORATED MOISTURE | EA 00380 |

| | | |
|---|--|----------|
| C | GI=GENERAL INFLATION RATE | EA 00390 |
| C | GN=GRAIN CONSTANT N | EA 00400 |
| C | GTQAU=GRAND TOTAL AUXILIARY HEAT SUPPLIED TO THE DEHUMIDIFIER | EA 00410 |
| C | GTQU=GRAND TOTAL OF SOLAR HEAT GAIN | EA 00420 |
| C | HID=SET ENTHALPY RATIO OF INLET AIR TO DRIER | EA 00430 |
| C | LADD=ADDITIONAL LOAD VARIED TO DETERMINE OPTIMUM LIFE CYCLE COST | EA 00440 |
| C | LCCC=LIFE CYCLE COSTS - CONVENTIONAL | EA 00450 |
| C | LCCS=LIFE CYCLE COSTS - SOLAR | EA 00460 |
| C | LP=LOAD DRIED BY DEHUMIDIFIER | EA 00470 |
| C | LPP=LOAD DRIED BY DEHUMIDIFIER AND SUPPLEMENTAL SOURCE | EA 00480 |
| C | MPIC=DISCOUNTING MORTGAGE, PRINCIPAL AND INTEREST - CONVENTIONAL | EA 00490 |
| C | MPIS=DISCOUNTING MORTGAGE, PRINCIPAL AND INTEREST - SOLAR | EA 00500 |
| C | ND=NUMBER OF DAYS TO BE ANALYZED | EA 00510 |
| C | NDEPC=TERM OF DEPRECIATION - CONVENTIONAL | EA 00520 |
| C | NDEPS=TERM OF DEPRECIATION - SOLAR | EA 00530 |
| C | NDEPPC=TERM OF DEPRECIATION, USED FOR D.B. AND S.O.D.-CONVENTION | EA 00540 |
| C | NDEPPS=TERM OF DEPRECIATION, USED FOR D.B. AND S.O.D.-SOLAR | EA 00550 |
| C | NEA=TERM OF THE ECONOMIC ANALYSIS | EA 00560 |
| C | NH=NUMBER OF HOURS TO BE ANALYZED | EA 00570 |
| C | NLS=TERM OF THE LOAN - SOLAR | EA 00580 |
| C | NLC=TERM OF THE LOAN - CONVENTIONAL | EA 00590 |
| C | NMINC=YEARS THE MORTGAGE PAYMENTS CONTRIBUTE TO THE ANALYSIS - | EA 00600 |
| C | CONVENTIONAL | EA 00610 |
| C | NMINPC= YEARS THE MORTGAGE PAYMENTS CONTRIBUTE TO THE ANALYSIS | EA 00620 |
| C | FOR THE EQUIPMENT - CONVENTIONAL | EA 00630 |
| C | NMINS=YEARS THE MORTGAGE PAYMENTS CONTRIBUTE TO THE ANALYSIS - | EA 00640 |
| C | SOLAR | EA 00650 |
| C | NMINPS= YEARS THE MORTGAGE PAYMENTS CONTRIBUTE TO THE ANALYSIS | EA 00660 |
| C | FOR THE EQUIPMENT - SOLAR | EA 00670 |
| C | POD=PARTIAL WATER VAPOR PRESSURE EXITING DRIER | EA 00680 |
| C | PONES=RATIO OF LIFE CYCLE FUEL COSTS TO FIRST YEAR FUEL COSTS | EA 00690 |
| C | - SOLAR | EA 00700 |
| C | PSTN=STATION ATMOSPHERIC PRESSURE - KILOPASCALS | EA 00710 |
| C | PSTNC=STATION ATMOSPHERIC PRESSURE - ATMOSPHERES | EA 00720 |
| C | PTCC=DISCOUNTED PROPERTY TAX COSTS - CONVENTIONAL | EA 00730 |
| C | PTCS=DISCOUNTED PROPERTY TAX COSTS - SOLAR | EA 00740 |
| C | PTWOS=LIFE CYCLE COSTS OF ADDITIONAL CAPITOL INVESTMENT TO | EA 00750 |
| C | INITIAL INVESTMENT - SOLAR | EA 00760 |

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C PW_C=PRESENT WORTH FACTOR FOR CALCULATING TERMS IN PCNE OR PTWO EA 00770
C ~ CONVENTIONAL EA 00780
C PW_S=PRESENT WORTH FACTOR FOR CALCULATING TERMS IN PCNE OR PTWO EA 00790
C ~ SCLAR EA 00800
C PWF=PRESENT WORTH FACTOR IF I NOT = D EA 00810
C PWF=PRESENT WORTH FACTOR IF I=D EA 00820
C RH=RELATIVE HUMIDITY OF AIR EXITING DRIER EA 00830
C RVS=RATIO OF RESALE VALUE AT END OF PERIOD OF ANALYSIS TO INITIAL EA 00840
C INVESTMENT ~ SOLAR EA 00850
C RVC=RATIO OF RESALE VALUE AT END OF PERIOD OF ANALYSIS TO INITIAL EA 00860
C INVESTMENT ~ CCNVENTIONAL EA 00870
C SMC=SOLAR RATIO OF MISC. COSTS TO INITIAL INVESTMENT EA 00880
C SMIR=ANNUAL MORTGAGE INTEREST RATE ~ SOLAR EA 00890
C STR=STATE TAX RATE EA 00900
C TAC=TOTAL COLLECTOR AREA EA 00910
C TAVS=PROPERTY TAX RATE BASED ON ASSESSED VALUE EA 00920
C TBAR=EFFECTIVE TAX RATE EA 00930
C TCR=TAX CREDIT RATE EA 00940
C TDIC=DISCOUNTED VALUE OF INCOME TAX DEDUCTIONS ON THE INTEREST EA 00950
C ~ CONVENTIONAL EA 00960
C TDIS=DISCOUNTED VALUE OF INCOME TAX DEDUCTIONS ON THE INTEREST EA 00970
C ~ SOLAR EA 00980
C TIDD-INLET TEMPERATURE TO DRYER, DRY BULB EA 00990
C TIDW-INLET TEMPERATURE TO DRYER, WET BULB EA 01000
C VC=RATIO OF ASSESSED VALUATION OF THE SYSTEM IN FIRST YEAR TO EA 01010
C THE INITIAL INVESTMENT OF THE SYSTEM ~ CONVENTIONA EA 01020
C VS=RATIO OF ASSESSED VALUATION OF THE SYSTEM IN FIRST YEAR TO EA 01030
C THE INITIAL INVESTMENT OF THE SYSTEM ~ SOLAR EA 01040
C WOD=HUMIDITY RATIO EXITING DRIER EA 01050
C WOX=HUMIDITY RATIO EXITING DEHUMIDIFIER ~ PROCESS AIR EA 01060
C REAL MPIS,LP,MPIC,LCCS,LCCC,LADD,LPP EA 01070
C INTEGER DOPS,DOPC,CFLAG EA 01080
C DIMENSION EM(10,24),EMT(10),POD(10,24),PSTN(10,24),PSTNC(10,24), EA 01090
C XRH(10,24),TIDD(10,24),TIDDR(10,24),WOXP(10,24),WOD(10,24) EA 01100
C X,IHR(10,24) EA 01110
C PWF(KK,Y,Z)=(1.-((1.+Y)/(1.+Z))**KK)/(Z-Y) EA 01120
C PWF(KKP,YP,ZP)=KKP/(1.+YP) EA 01130
C THE ECCNMC ANALYSIS BEGINS HERE EA 01140

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C      READ INFLATION AND TAX RATES
      READ (5,90) FI,GI,FTR,STR
C      READ TERM CF ECONOMIC ANALYSIS, DISCOUNT RATES AND FUEL COSTS
      READ (5,91) NEA,DRS,DRC,CF,CFLAG
C      READ SOLAR MORTGAGE DATA
      READ (5,92) NLS,DS,SMIR,TCR
C      READ SOLAR DEPRECIATION DATA
      READ (5,93) DCPS,NDEPS
C      READ SOLAR COST AND VALUE DATA
      READ (5,94) CA,CES,TAVS,VS,RVS,SMC
C      READ CONVENTICNAL MORTGAGE DATA
      READ (5,95) NLC,DC,CMIR
C      READ CONVENTICNAL DEPRECIATION DATA
      READ (5,96) DCPC,NDEPC
C      READ CCNVENTICNAL CCST AND VALUE DATA
      READ (5,97) CEC,TAVC,VC,RVC,CMC
C      READ SYSTEM OPERATING DATA
      READ (5,98) AMASSP,GTQU,ND,NH,TAC
C      READ PROCESS COMMODITY DATA
      READ (5,18) EMCD,GC,GN,TIDW,HID
      BETAD=374.12-TIDW
      BETAFD=BETAD/100.
      PSAT=218.167*10**(-BETAD/(TIDW+273.15))*(3.2437814+
X.586826*BETAFD+0.011702379*BETAFD**3)/(1.+2187846*BETAFD))
C      WRITE DRYER PARAMETERS
      WRITE(6,170)
      WRITE(6,171) AMASSP
      WRITE(6,172) ND
      WRITE(6,173) NH
      WRITE(6,174) GTQU
      WRITE(6,175) TAC
      WRITE(6,176) EMCD,GC,GN
      WRITE(6,177) TIDW
      WRITE(6,178) FID
170 FORMAT ('1',//,T6,'***** COMMODITY DRYER PARAMETERS *****',)
171 FORMAT ('-',//T8,'AIR FLOW RATE',T40,'= ',F8.5,' KG/S',)
172 FORMAT ('0',T8,'NUMBER OF DAYS ANALYZED',T40,'= ',I3)
173 FORMAT ('0',T8,'NUMBER OF HOURS PER DAY',T40,'= ',I3)
EA 01150
EA 01160
EA 01170
EA 01180
EA 01190
EA 01200
EA 01210
EA 01220
EA 01230
EA 01240
EA 01250
EA 01260
EA 01270
EA 01280
EA 01290
EA 01300
EA 01310
EA 01320
EA 01330
EA 01340
EA 01350
EA 01360
EA 01370
EA 01380
EA 01390
EA 01400
EA 01410
EA 01420
EA 01430
EA 01440
EA 01450
EA 01460
EA 01470
EA 01480
EA 01490
EA 01500
EA 01510
EA 01520

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174 FORMAT('0',T8,'TOTAL HEAT SUPPLIED BY SOLAR',T40,'= ',F10.6,' GJUEA 01530
    XLE')
    EA 01540
175 FORMAT('0',T8,'TOTAL COLLECTOR AREA',T40,'= ',F6.3,' M**2')
    EA 01550
176 FORMAT('0',T8,'CCMMODITY INITIAL MOISTURE',T40,'= ',F6.2,' PERCENEA 01560
    XT DRY BASIS',T19,'CONTENT',T19,'CONSTANT "C",T40,'= ',F10.6/T19,' EA 01570
    X'CONSTANT "N",T40,'= ',F5.2)
    EA 01580
177 FORMAT('0',T8,'SET INLET WET BULB',T40,'= ',F6.2/T8,'TEMPERATURE')EA 01590
178 FORMAT('0',T8,'SET INLET ENTHALPY',T40,'= ',F6.2,' KJ/KG DRY AIR')EA 01600
    DO 10 I=1,ND
    EA 01610
    DO 11 J=1,NH
    EA 01620
    C READ HCURLY DATA FROM SYSTEM
    EA 01630
    C READ (5,99) IHR(I,J),WOXP(I,J),PSTN(I,J)
    EA 01640
    C CALCULATE THE EVAPORATED MOISTURE, IN KILOGRAMS MOISTURE PER
    EA 01650
    C HOUR
    EA 01660
    C HEAT INLET DRYER AIR TO TIDW
    EA 01670
    TIDD(I,J)=(HID-2501.*WOXP(I,J))/(1.+1.805*WOXP(I,J))
    EA 01680
    C TIDDR=TIDD IN DEGREES RANKINE
    EA 01690
    TIDDR(I,J)=1.8*TIDD(I,J)+492.
    EA 01700
    RH(I,J)=1.-EXP(-GC*TIDDR(I,J)*(EMCD**GN))
    EA 01710
    POD(I,J)=PSAT*RH(I,J)
    EA 01720
    PSTNC(I,J)=PSTN(I,J)*0.0098692
    EA 01730
    WOD(I,J)=.62158*POD(I,J)/(PSTNC(I,J)-POD(I,J))
    EA 01740
    EM(I,J)=AMASSP*(WOD(I,J)-WOXP(I,J))*3600.
    EA 01750
    C NEED TO INITIALIZE DAILY TOTAL EVAPORATED MOISTURE
    EA 01760
    IF (J.EQ.1) EMT(I)=0.0
    EA 01770
    EMT(I)=EMT(I)+EM(I,J)
    EA 01780
    EA 01790
    11 CONTINUE
    EA 01800
    C WRITE HOURLY TEMPERATURE AND HUMIDITY MAP FOR THE DRYER
    WRITE (6,160) I
    EA 01810
    WRITE (6,163)
    EA 01820
    WRITE (6,164)
    EA 01830
    WRITE (6,165)
    EA 01840
    DC 167 J=1,NH
    EA 01850
    RHP=RH(I,J)*100.
    EA 01860
    WRITE(6,166) IHR(I,J),TIDD(I,J),WOXP(I,J),RHP,WOD(I,J),EM(I,J)
    EA 01870
    167 CONTINUE
    EA 01880
    WRITE (6,168) EMT(I)
    EA 01890
    160 FORMAT('1',/T6,'***** HOURLY TEMPERATURE AND HUMIDITY MAP *****'EA 01900

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X,T100,'**** DRYER ****'//T102,'DAY ',I3)
163 FORMAT('O',T25,'INLET',T50,'OUTLET'//T8,'HOUR',T44,'RELATIVE',T70,EA 01910
X'MOISTURE')
164 FORMAT('O',T7,'ENDING',T20,'TEMP',T29,'HUMIDITY',T44,'HUMIDITY',T5EA 01920
X4,'HUMIDITY',T69,'EVAPCRATED')
165 FORMAT('O',T6,8(' '),T19,18(' '),T43,18(' '),T68,12(' '),EA 01930
166 FORMAT(' ',T6,I5,T20,F5.1,T30,F6.4,T45,F5.1,T55,F6.4,T70,F5.2) EA 01940
168 FORMAT(' ',//T12,'DAILY TOTAL MOISTURE EVAPCRATED = ',F6.2) EA 01950
IF (I.EQ.1) GEMT=0.0
GEMT=GEMT+EMT(I)
10 CONTINUE
90 FORMAT (4F5.2)
91 FORMAT (15,2F5.3,F5.2,I5)
92 FORMAT (15,3F5.2)
93 FORMAT (2I5)
94 FORMAT (F5.0,F10.0,3F10.2,F5.2)
95 FORMAT (15,2F5.2)
96 FORMAT (2I5)
97 FORMAT (F10.0,3F10.2,F5.2)
98 FORMAT (F10.5,F10.6,I3,I2,F6.3)
99 FORMAT (I6,F6.4,F6.2)
18 FCRMAT (F6.2,F10.8,F5.3,F5.1,F10.4)
ZERO=0.0
C NMINPS IS EQUAL TO THE MINIMUM OF NLS AND NEA
IF (NEA.GT.NLS) GO TO 82
NMIN=NEA
GO TO 83
82 NMIN=NLS
83 CONTINUE
C NDEPS IS EQUAL TO THE MINIMUM OF NEA AND NDEPS
IF (NEA.GT.NDEPS) GO TO 84
NMINPS=NEA
GO TO 85
84 NMINPS=NDEPS
85 CONTINUE
NDEPPS=NDEPS-1.
NDEPPC=NDEPC-1.
DIS=-2./NDEPS
EA 01960
EA 01970
EA 01980
EA 01990
EA 02000
EA 02010
EA 02020
EA 02030
EA 02040
EA 02050
EA 02060
EA 02070
EA 02080
EA 02090
EA 02100
EA 02110
EA 02120
EA 02130
EA 02140
EA 02150
EA 02160
EA 02170
EA 02180
EA 02190
EA 02200
EA 02210
EA 02220
EA 02230
EA 02240
EA 02250
EA 02260
EA 02270
EA 02280

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CIC=-2./NDEPC
TBAR=FTR+STR-FTR*STR
  DEFINE THE PRESENT WORTH FACTORS FOR THE SOLAR ENERGY SYSTEM
  IF (FI.NE.DRS) PWAS=PWFF(NEA,FI,DRS)
  IF (FI.EQ.DRS) PWAS=PWFF(NEA,FI,DRS)
  IF (ZERO.NE.DRS) PWBS=PWFF(NMINS,ZERO,DRS)
  IF (ZERO.EQ.DRS) PWBS=PWFF(NMINS,ZERO,DRS)
  IF (ZERO.NE.SMIR) PWCS=PWFF(NLS,ZERO,SMIR)
  IF (ZERO.EQ.SMIR) PWCS=PWFF(NLS,ZERO,SMIR)
  IF (SMIR.NE.DRS) PWDS=PWFF(NMINS,SMIR,DRS)
  IF (SMIR.EQ.DRS) PWDS=PWFF(NMINS,SMIR,DRS)
  IF (GI.NE.DRS) PWES=PWFF(NEA,GI,DRS)
  IF (GI.EQ.DRS) PWES=PWFF(NEA,GI,DRS)
  IF (DIS.NE.DRS) PWFS=PWFF(NMINPS,ZERO,DRS)
  IF (DIS.EQ.DRS) PWFS=PWFF(NMINPS,ZERO,DRS)
  IF (DIS.NE.DRS) PWGS=PWFF(NDEPPS,DIS,DRS)
  IF (DIS.EQ.DRS) PWGS=PWFF(NDEPPS,DIS,DRS)
  IF (DIS.NE.ZERO) PWHS=PWFF(NDEPPS,DIS,ZERO)
  IF (DIS.EQ.ZERO) PWHS=PWFF(NDEPPS,DIS,ZERO)
  IF (ZERO.NE.DRS) PWIS=PWFF(NDEPS,ZERO,DRS)
  IF (ZERO.EQ.DRS) PWIS=PWFF(NDEPS,ZERO,DRS)
  IF (ZERO.NE.DRS) PWJS=PWFF(NDEPPS,ZERO,DRS)
  IF (ZERO.EQ.DRS) PWJS=PWFF(NDEPPS,ZERO,DRS)
  PWKS=1./(1.+DRS)
  PWLS=PWKS*PWKS
  PONES=(1.-CFLAG*TBAR)*PWAS
  MPIS=(1.-DS)*PWBS/PWCS
  TDIS=(1.-DS)*TBAR*(PWCS*(SMIR-1./PWCS)+PWBS/PWCS)
  DMCS=(1.-CFLAG*TBAR)*SMC*PWES
  PTCS=TAVS*(1.-TBAR)*VS*PWES
  STRAIGHT LINE DEPRECIATION IF DOPS=1, DECLINING BALANCE IF
  DOPS=2, SUM OF DIGITS IF DOPS=3, ACCELERATED CCST RECOVERY
  IF DOPS=4
    GO TO (801,802,803,804),DOPS
  801 DEPS=CFLAG*TBAR*PWFS/NDEPS
    GO TO 800
  802 DEPS=TBAR*CFLAG*(1.+2./NDEPS*(PWGS-PWHS/((1.+DRS)**NDEPS)))
    GO TO 800

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EA 02290
EA 02300
EA 02310
EA 02320
EA 02330
EA 02340
EA 02350
EA 02360
EA 02370
EA 02380
EA 02390
EA 02400
EA 02410
EA 02420
EA 02430
EA 02440
EA 02450
EA 02460
EA 02470
EA 02480
EA 02490
EA 02500
EA 02510
EA 02520
EA 02530
EA 02540
EA 02550
EA 02560
EA 02570
EA 02580
EA 02590
EA 02600
EA 02610
EA 02620
EA 02630
EA 02640
EA 02650
EA 02660

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803 DEPS=CFLAG*2.*TBAR/(NDEPS*(NDEPS+1.))*((PWIS+(NDEPS-1.-PWJS)/DRS) EA 02670
GC TO 800 EA 02680
804 DEPS=(0.2*PWKS+0.32*PWLS+0.24*PWKS*PWLS+0.16*PWLS*PWLS+0.08*PWKS* EA 02690
X*PWLS*PWLS)*TEAR EA 02700
800 CONTINUE EA 02710
C THE TERM FOR DECLINING BALANCE DEPRECIATION INCLUDES THE RESALE EA 02720
C VALUE EA 02730
EA 02740
IF (DOPS.EQ.2) RVS=0. EA 02750
DRVS=RVS/((1.+DRS)**NEA) EA 02760
PTWOS=DS+MPIIS-TDIS+DMCS+PTCS-DEPS-DRVS-TCR EA 02770
NMINC IS EQUAL TO THE MINIMUM OF NLC AND NEA EA 02780
IF (NEA.GT.NLC) GO TO 86 EA 02790
NMINC=NEA EA 02800
GO TO 87 EA 02810
86 NMINC=NLC EA 02820
87 CONTINUE EA 02830
C NMINPC IS EQUAL TO THE MINIMUM OF NEA AND NDEPC EA 02840
IF (NEA.GT.NDEPC) GO TO 88 EA 02850
NMINPC=NEA EA 02860
GO TO 89 EA 02870
88 NMINPC=NDEPC EA 02880
89 CONTINUE EA 02890
C DEFINE THE PRESENT WORTH FACTORS FOR THE CONVENTIONAL SYSTEM EA 02900
EA 02910
IF (FI.NE.DRC) PWAC=PWFP(NEA,FI,DRC) EA 02920
IF (FI.EQ.DRC) PWAC=PWFP(NEA,FI,DRC) EA 02930
IF (ZERO.NE.DRC) PWBC=PWFP(NMINC,ZERO,DRC) EA 02940
IF (ZERO.EQ.DRC) PWBC=PWFP(NMINC,ZERO,DRC) EA 02950
IF (ZERO.NE.CMIR) PWCC=PWFP(NLC,ZERO,CMIR) EA 02960
IF (ZERO.EQ.CMIR) PWCC=PWFP(NLC,ZERO,CMIR) EA 02970
IF (CMIR.NE.DRC) PWDC=PWFP(NMINC,CMIR,DRC) EA 02980
IF (CMIR.EQ.DRC) PWDC=PWFP(NMINC,CMIR,DRC) EA 02990
IF (GI.NE.DRC) PWEC=PWFP(NEA,GI,DRC) EA 03000
IF (GI.EQ.DRC) PWEC=PWFP(NEA,GI,DRC) EA 03010
IF (DIC.NE.DRC) PWFC=PWFP(NMINPC,ZERO,DRC) EA 03020
IF (DIC.EQ.DRC) PWFC=PWFP(NMINPC,ZERO,DRC) EA 03030
IF (DIC.NE.DRC) PWGC=PWFP(NDEPPC,DIC,DRC) EA 03040
IF (DIC.EQ.DRC) PWGC=PWFP(NDEPPC,DIC,DRC)
IF (DIC.NE.ZERO) PWHC=PWFP(NDEPPC,DIC,ZERO)

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IF (CIC.EQ.ZERO) PWHC=PWFP(NDEPPC,DIC,ZERO)
IF (ZERO.NE.DRC) PWIC=PWFP(NDEPC,ZERO,DRC)
IF (ZERO.EQ.DRC) PWIC=PWFP(NDEPC,ZERO,DRC)
IF (ZERO.NE.DRC) PWJC=PWFP(NDEPPC,ZERO,DRC)
IF (ZERO.EQ.DRC) PWJC=PWFP(NDEPPC,ZERO,DRC)
PWKC=1./{(1.+CRC)
PWLC=PWKC*PWKC
PONEC={1.-CFLAG*TBAR}*PWAC
MPIC={1.-DC}*PWBC/PWCC
TDIC={1.-DC}*TBAR*(PWDC*(CMIR-1./PWCC)+PWBC/PWCC)
DMCC={1.-CFLAG*TEAR}*CMC*PWEC
PTCC=TAVC*(1.-TBAR)*VC*PWEC
      STRAIGHT LINE DEPRECIATION IF DOPC=1, DECLINING BALANCE IF
      DOPC=2, SUM OF DIGITS IF DOPC=3, ACCELERATED COST RECOVERY
      IF DOPC=4
701 GO TO (701,702,703,704),DOPC
      DEPC=TBAR*PWFC/NDEPC
      GO TO 700
702 DEPC=TBAR*(1.+2.*CFLAG/NDEPC*(PWGC-PWHC/((1.+DRC)**NDEPC)))
      GO TO 700
703 DEPC=2.*TBAR/(NDEPC*(NDEPC+1.))*{(PWIC+(NDEPC-1.-PWJC)/DRC)
      GO TO 700
704 DEPC={0.2*PWKC+0.32*PWLC+0.24*PWKC*PWLC+0.16*PWLC*PWLC+0.08*PWKC*
      X*PWLC*PWLC)*TBAR
700 CONTINUE
      THE TERM FOR DECLINING BALANCE DEPRECIATION INCLUDES THE RESALE
      VALUE
      IF (DOPC.EQ.2) RVC=0.
      DRVC=RVC/((1.+DRC)**NEA)
      PTWOC=DC+MPIC-TDIC+DMCC+PTCC-DEPC-DRVC
      OUTPUT FOR ECONOMIC ANALYSIS PARAMETERS
      DRSP=DRS*100.
      DRCP=DRC*100.
      DSP=DS*100.
      DCP=DC*100.
      SMIRP=SMIR*100.
      CMIRP=CMIR*100.
      SMCN=SMC*100.
EA 03050
EA 03060
EA 03070
EA 03080
EA 03090
EA 03100
EA 03110
EA 03120
EA 03130
EA 03140
EA 03150
EA 03160
EA 03170
EA 03180
EA 03190
EA 03200
EA 03210
EA 03220
EA 03230
EA 03240
EA 03250
EA 03260
EA 03270
EA 03280
EA 03290
EA 03300
EA 03310
EA 03320
EA 03330
EA 03340
EA 03350
EA 03360
EA 03370
EA 03380
EA 03390
EA 03400
EA 03410
EA 03420

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CMCP=CMC*100.
TCRP=TCR*100.
FIP=FI*100.
GIP=GI*100.
FTR=FTR*100.
STR=STR*100.
WRITE (6,100)
WRITE (6,101)
WRITE (6,102) DRSP,DRCP
WRITE (6,103) NLS,NLC
WRITE (6,104) DSF,DCP
WRITE (6,105) SMIRP,CMIRP
WRITE (6,106) NDEPS,NDEPC
WRITE (6,107) TAVS,TAVC
WRITE (6,108) VS,VC
WRITE (6,109) RVS,RVC
WRITE (6,110) SMCP,CMCP
WRITE (6,111) CES,CEC
WRITE (6,112) CA
WRITE (6,113) TCRP
WRITE (6,114) FIP
WRITE (6,115) GIP
WRITE (6,116) FTRP
WRITE (6,117) STRP
WRITE (6,118) CF
WRITE (6,119) NEA
GO TO (120,121,122,127),DOPS
120 WRITE (6,124)
GO TO 123
121 WRITE (6,125)
GO TO 123
122 WRITE (6,126)
GO TO 123
127 WRITE (6,128)
123 CONTINUE
GO TO (130,131,132,137),DOPC
130 WRITE (6,134)
GO TO 133
EA 03430
EA 03440
EA 03450
EA 03460
EA 03470
EA 03480
EA 03490
EA 03500
EA 03510
EA 03520
EA 03530
EA 03540
EA 03550
EA 03560
EA 03570
EA 03580
EA 03590
EA 03600
EA 03610
EA 03620
EA 03630
EA 03640
EA 03650
EA 03660
EA 03670
EA 03680
EA 03690
EA 03700
EA 03710
EA 03720
EA 03730
EA 03740
EA 03750
EA 03760
EA 03770
EA 03780
EA 03790
EA 03800

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131 WRITE (6,135)
    GO TO 132
132 WRITE (6,136)
    GO TO 133
137 WRITE (6,138)
133 CONTINUE
    ICFLAG=CFLAG+1
    GO TO (142,143), ICFLAG
142 WRITE (6,140)
    GO TO 144
143 WRITE (6,141)
144 CONTINUE
100 FORMAT ('1',/T6,'***** ECONOMIC ANALYSIS PARAMETERS *****')
101 FORMAT ('-',T45,'SOLAR ENERGY',T65,'CCONVENTIONAL',T45,'DEHUMIDIFIER',
    X',T65,'DEHUMIDIFIER',T44,14(' '),T64,14(' '))
102 FORMAT ('-',T5,'MARKET DISCOUNT RATE - PERCENT',T48,F5.1,T68,F5.1)
103 FORMAT ('0',T5,'TERM OF MORTGAGE - YEARS',T46,I5,T66,I5)
104 FORMAT ('0',T5,'DOWNPAYMENT - PERCENT',T48,F5.1,T68,F5.1)
105 FORMAT ('0',T5,'MORTGAGE INTEREST RATE - PERCENT',T48,F5.1,T68,F5.1,
    X)
106 FORMAT ('0',T5,'TERM OF DEPRECIATION - YEARS',T46,I5,T66,I5)
107 FORMAT ('0',T5,'PROPERTY TAX - DOLLARS',T43,F10.2,T63,F10.2)
108 FORMAT ('0',T5,'ASSESSED VALUATION - DOLLARS',T43,F10.2,T63,F10.2)
109 FORMAT ('0',T5,'RESALE VALUE - DOLLARS',T43,F10.2,T63,F10.2)
110 FORMAT ('0',T5,'MISCELLANEOUS COSTS - PERCENT',T48,F5.1,T68,F5.1)
111 FORMAT ('0',T5,'FIXED COST OF EQUIPMENT',T44,F10.0,T64,F10.0)
112 FORMAT ('0',T5,'COST PER COLLECTOR AREA',T48,F5.0,T69,5(' '))
113 FORMAT ('0',T5,'FEDERAL TAX CREDIT - PERCENT',T48,F5.1,T69,5(' '))
114 FORMAT ('-',T25,'FUEL INFLATION RATE',T54,'=',F5.1,' PERCENT')
115 FORMAT ('0',T25,'GENERAL INFLATION RATE',T54,'=',F5.1,' PERCENT')
116 FORMAT ('0',T25,'FEDERAL TAX RATE',T54,'=',F5.1,' PERCENT')
117 FORMAT ('0',T25,'STATE TAX RATE',T54,'=',F5.1,' PERCENT')
118 FORMAT ('0',T25,'FUEL PRICE',T54,'=',F5.2,' DOLLAR/GJoule')
119 FORMAT ('0',T25,'TERM OF ECONOMIC ANALYSIS',T54,'=',I5,' YEARS')
124 FORMAT ('-',T25,'STRAIGHT LINE DEPRECIATION - SOLAR')
125 FORMAT ('-',T25,'DECLINING BALANCE DEPRECIATION - SOLAR')
126 FORMAT ('-',T25,'SUM OF DIGITS DEPRECIATION - SOLAR')
128 FORMAT ('-',T25,'ACCELERATED COST RECOVERY DEPRECIATION - SOLAR')
EA 03810
EA 03820
EA 03830
EA 03840
EA 03850
EA 03860
EA 03870
EA 03880
EA 03890
EA 03900
EA 03910
EA 03920
EA 03930
EA 03940
EA 03950
EA 03960
EA 03970
EA 03980
EA 03990
EA 04000
EA 04010
EA 04020
EA 04030
EA 04040
EA 04050
EA 04060
EA 04070
EA 04080
EA 04090
EA 04100
EA 04110
EA 04120
EA 04130
EA 04140
EA 04150
EA 04160
EA 04170
EA 04180

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134 FORMAT('0',T25,'STRAIGHT LINE DEPRECIATION - CONVENTICNAL') EA 04190
135 FORMAT('0',T25,'DECLINING BALANCE DEPRECIATION - CONVENTICNAL') EA 04200
136 FORMAT('0',T25,'SUM OF DIGITS DEPRECIATION - CONVENTICNAL') EA 04210
138 FORMAT('0',T25,'ACCELERATED COST RECOVERY DEPRECIATION - CONVENTIOEA 04220
    XNAL') EA 04230
140 FORMAT('0',T25,'NON-INCOME PRODUCING SYSTEM') EA 04240
141 FORMAT('0',T25,'INCOME PRODUCING SYSTEM') EA 04250
    WRITE (6,151) EA 04260
    WRITE (6,152) EA 04270
    WRITE (6,153) EA 04280
    LADD=0. EA 04290
    DO 150 KOUNT=1,16 EA 04300
    LP=GEMT#0.00240 EA 04310
    LPP=LP+LADD EA 04320
    F=GTQU/LPP EA 04330
    FF=F#100. EA 04340
    EF=1./{(1.-F) EA 04350
    LPP IS IN GJOULE EA 04360
    LCCS=PTWGS*(CA#TAC+CES)+PTWDC*CEC+PONES#CF*LPP*(1.-F) EA 04370
    LCCC=PTWGC*(CEC)+PONEC*CF*LPP EA 04380
    WRITE (6,154) LP,LADD,LPP,FF,LCCS,LCCC,EF EA 04390
    LADD=LADD+0.025 EA 04400
150 CONTINUE EA 04410
    WRITE (6,155) PGNES,PONEC EA 04420
    WRITE (6,156) PTWOS,PTWCC EA 04430
    WRITE (6,157) EA 04440
    WRITE (6,158) EA 04450
151 FCFORMAT ('1',T6,'***** ECONOMIC ANALYSIS *****') EA 04460
152 FCFORMAT ('-',T53,'SOLAR ENERGY',T87,'CONVENTICNAL',//T14,'LOAD SUPPLEA 04470
    XIED BY',T45,'FRACTION OF',T15,'SUPPLEMENTARY',T44,'LOAD SUPPLIED',/EA 04480
    XT2,'DEHUMIDIFIER',T19,'HEATER',T32,'TOTAL',T47,'BY SOLAR',T60,'LIFEA 04490
    XE CYCLE COST',T85,'LIFE CYCLE COST',T106,'DRYER EFFICIENCY') EA 04500
153 FCFORMAT (' ',T2,T2,12(' '),T15,13(' '),T31,7(' '),T44,13(' '),T59,17('EA 04510
    X-',T84,17(' '),T105,16(' ')) EA 04520
154 FCFORMAT ('0',T2,F10.3,T14,F10.3,T28,F10.4,T48,F5.1,T60,F10.2,T86,F1EA 04530
    X0.2,T109,F1C.4) EA 04540
155 FCFORMAT ('-',T53,'P-ONE = ',F8.4,T84,'P-ONE = ',F8.4) EA 04550
    EA 04560

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156 FORMAT ('-',T53,'P-TWO = ',F8.4,T84,'P-TWO = ',F8.4)
157 FORMAT ('-',T6,'MOISTURE EVAPORATED IN KILOGRAMS',/T6,'LOAD IN GJOUEA 04580
XLE '/T6,'SOLAR FRACTION IN PERCENT',/T6,'LIFE CYCLE COSTS IN DOLLAREA 04590
XS')
158 FORMAT ('1')
STOP
END
EA 04570
EA 04580
EA 04590
EA 04600
EA 04610
EA 04620
EA 04630

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| \$ENTRY | 6 | 30 | 5 | 1 | 0 | 0 | 0 | 2 |
|---------|----------|------|----|-----|-------|---|---|---|
| 13 | 120 | 1200 | | | | | | |
| 15 | 10 | 12 | 55 | | | | | |
| 5 | 10 | 12 | 55 | | | | | |
| 4 | 5 | | | | | | | |
| 300 | 1000 | | | 0 | 0 | 0 | 0 | 2 |
| 5 | 10 | 12 | | | | | | |
| 4 | 5 | | | | | | | |
| 2000 | | | | | 0 | 0 | 0 | 2 |
| 02836 | 143420 | | | 524 | 3272 | | | |
| 2500 | 00001100 | 19 | | 190 | 54216 | | | |
| 1 | 0052 | 9837 | | | | | | |
| 2 | 0000 | 9841 | | | | | | |
| 3 | 0002 | 9844 | | | | | | |
| 4 | 0000 | 9845 | | | | | | |
| 5 | 0000 | 9847 | | | | | | |
| 6 | 0000 | 9848 | | | | | | |
| 7 | 0000 | 9851 | | | | | | |
| 8 | 0000 | 9855 | | | | | | |
| 9 | 0005 | 9858 | | | | | | |
| 10 | 0005 | 9852 | | | | | | |
| 11 | 0008 | 9847 | | | | | | |
| 12 | 0018 | 9841 | | | | | | |
| 13 | 0024 | 9830 | | | | | | |
| 14 | 0024 | 9818 | | | | | | |
| 15 | 0029 | 9807 | | | | | | |
| 16 | 0032 | 9805 | | | | | | |
| 17 | 0024 | 9802 | | | | | | |
| 18 | 0022 | 9800 | | | | | | |

| | | |
|----|------|------|
| 19 | 0015 | 9800 |
| 20 | 0012 | 9800 |
| 21 | 0009 | 9800 |
| 22 | 0008 | 9794 |
| 23 | 0010 | 9789 |
| 24 | 0005 | 9783 |

***** COMMODITY DRYER PARAMETERS *****

| | | |
|---------------------------------------|---|-------------------------|
| AIR FLOW RATE | = | 0.02836 KG/S |
| NUMBER OF DAYS ANALYZED | = | 5 |
| NUMBER OF HOURS PER DAY | = | 24 |
| TOTAL HEAT SUPPLIED BY SOLAR | = | 0.143420 GJ/DOLE |
| TOTAL COLLECTOR AREA | = | 3.272 M**2 |
| COMMODITY INITIAL MOISTURE CONTENT | = | 25.00 PERCENT DRY BASIS |
| CONSTANT "C" | = | 0.000011 |
| CONSTANT "N" | = | 1.90 |
| SET INLET WET BULB TEMPERATURE | = | 19.00 |
| SET INLET ENTHALPY | = | 54.22 KJ/KG DRY AIR |

**** DRYER ****

DAY 1

**** HOURLY TEMPERATURE AND HUMIDITY MAP ****

| HOUR ENDING | INLET | | OUTLET | | MOISTURE EVAPORATED |
|----------------|-------|----------|----------------------|----------|------------------------|
| | TEMP | HUMIDITY | RELATIVE HUMIDITY | HUMIDITY | |
| 1 | 40.8 | 0.0052 | 94.0 | 0.0133 | 0.82 |
| 2 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 3 | 53.7 | 0.0002 | 94.7 | 0.0134 | 1.35 |
| 4 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 5 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 6 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 7 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 8 | 54.2 | 0.0 | 94.7 | 0.0134 | 1.37 |
| 9 | 52.9 | 0.0005 | 94.6 | 0.0134 | 1.32 |
| 10 | 52.9 | 0.0005 | 94.6 | 0.0134 | 1.32 |
| 11 | 52.1 | 0.0008 | 94.6 | 0.0134 | 1.29 |
| 12 | 49.6 | 0.0018 | 94.5 | 0.0134 | 1.18 |
| 13 | 48.0 | 0.0024 | 94.4 | 0.0134 | 1.12 |
| 14 | 48.0 | 0.0024 | 94.4 | 0.0134 | 1.12 |
| 15 | 46.7 | 0.0029 | 94.3 | 0.0134 | 1.07 |
| 16 | 45.9 | 0.0032 | 94.3 | 0.0134 | 1.04 |
| 17 | 48.0 | 0.0024 | 94.4 | 0.0134 | 1.13 |
| 18 | 48.5 | 0.0022 | 94.4 | 0.0134 | 1.15 |
| 19 | 50.3 | 0.0015 | 94.5 | 0.0135 | 1.22 |
| 20 | 51.1 | 0.0012 | 94.6 | 0.0135 | 1.25 |
| 21 | 51.9 | 0.0009 | 94.6 | 0.0135 | 1.28 |
| 22 | 52.1 | 0.0008 | 94.6 | 0.0135 | 1.29 |
| 23 | 51.6 | 0.0010 | 94.6 | 0.0135 | 1.27 |
| 24 | 52.9 | 0.0005 | 94.6 | 0.0135 | 1.33 |

DAILY TOTAL MOISTURE EVAPORATED = 29.79

***** ECONOMIC ANALYSIS PARAMETERS *****

| | SOLAR ENERGY DEHUMIDIFIER | CONVENTIONAL DEHUMIDIFIER |
|----------------------------------|------------------------------|------------------------------|
| MARKET DISCOUNT RATE - PERCENT | 12.0 | 12.0 |
| TERM OF MORTGAGE - YEARS | 5 | 5 |
| DOWNPAYMENT - PERCENT | 10.0 | 10.0 |
| MORTGAGE INTEREST RATE - PERCENT | 12.0 | 12.0 |
| TERM OF DEPRECIATION - YEARS | 5 | 5 |
| PROPERTY TAX - DOLLARS | 0.0 | 0.0 |
| ASSESSED VALUATION - DOLLARS | 0.0 | 0.0 |
| RESALE VALUE - DOLLARS | 0.0 | 0.0 |
| MISCELLANEOUS COSTS - PERCENT | 2.0 | 2.0 |
| FIXED COST OF EQUIPMENT | 1000. | 2000. |
| COST PER COLLECTOR AREA | 300. | ---- |
| FEDERAL TAX CREDIT - PERCENT | 55.0 | ---- |

| | | | |
|---------------------------|---|-------|----------------|
| FUEL INFLATION RATE | = | 13.0 | PERCENT |
| GENERAL INFLATION RATE | = | 6.0 | PERCENT |
| FEDERAL TAX RATE | = | 30.0 | PERCENT |
| STATE TAX RATE | = | 5.0 | PERCENT |
| FUEL PRICE | = | 12.00 | DOLLAR/GJ/DOLE |
| TERM OF ECONOMIC ANALYSIS | = | 15 | YEARS |

ACCELERATED COST RECOVERY DEPRECIATION - SOLAR
ACCELERATED COST RECOVERY DEPRECIATION - CONVENTIONAL
INCOME PRODUCING SYSTEM

***** ECONOMIC ANALYSIS *****

| LOAD SUPPLIED BY | | | SOLAR ENERGY | | CONVENTIONAL | |
|------------------|----------------------|--------|------------------------------------|-----------------|-----------------|------------------|
| DEHUMIDIFIER | SUPPLEMENTARY HEATER | TOTAL | FRACTION OF LOAD SUPPLIED BY SOLAR | LIFE CYCLE COST | LIFE CYCLE COST | DRYER EFFICIENCY |
| 0.365 | 0.0 | 0.3654 | 39.3 | 2251.22 | 1707.16 | 1.6462 |
| 0.365 | 0.025 | 0.3904 | 36.7 | 2254.07 | 1710.01 | 1.5807 |
| 0.365 | 0.050 | 0.4154 | 34.5 | 2256.91 | 1712.85 | 1.5274 |
| 0.365 | 0.075 | 0.4404 | 32.6 | 2259.76 | 1715.70 | 1.4830 |
| 0.365 | 0.100 | 0.4654 | 30.8 | 2262.60 | 1718.54 | 1.4455 |
| 0.365 | 0.125 | 0.4904 | 29.2 | 2265.45 | 1721.39 | 1.4134 |
| 0.365 | 0.150 | 0.5154 | 27.8 | 2268.29 | 1724.23 | 1.3856 |
| 0.365 | 0.175 | 0.5404 | 26.5 | 2271.14 | 1727.08 | 1.3613 |
| 0.365 | 0.200 | 0.5654 | 25.4 | 2273.98 | 1729.92 | 1.3399 |
| 0.365 | 0.225 | 0.5904 | 24.3 | 2276.83 | 1732.77 | 1.3209 |
| 0.365 | 0.250 | 0.6154 | 23.3 | 2279.67 | 1735.61 | 1.3039 |
| 0.365 | 0.275 | 0.6404 | 22.4 | 2282.52 | 1738.46 | 1.2886 |
| 0.365 | 0.300 | 0.6654 | 21.6 | 2285.36 | 1741.30 | 1.2748 |
| 0.365 | 0.325 | 0.6904 | 20.8 | 2288.21 | 1744.15 | 1.2622 |
| 0.365 | 0.350 | 0.7154 | 20.0 | 2291.05 | 1746.99 | 1.2508 |
| 0.365 | 0.375 | 0.7404 | 19.4 | 2293.90 | 1749.84 | 1.2403 |

P-ONE = 9.4832 P-TWO = 9.4832

P-ONE = 9.4832 P-TWO = 0.8328

MOISTURE EVAPORATED IN KILOGRAMS
LCAD IN GJOULE
SOLAR FRACTION IN PERCENT

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Finally, this thesis is for my father, who taught me how to build my Yellow Brick Road, to my mother, who taught me the determination and love to start building it, and to my brothers and sisters, who have never ceased to give me the motivation to dream about it.

VITA

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FEASIBILITY STUDY OF A SOLAR ENERGY POWERED
SORPTION DEHUMIDIFICATION SYSTEM

by

ANN FERN ATKINSON

B.S., Kansas State University, 1981

ABSTRACT OF A MASTER'S THESIS

submitted in partial fulfillment of the

requirements for the degree

MASTER OF SCIENCE

Department of Mechanical Engineering

KANSAS STATE UNIVERSITY
Manhattan, Kansas

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ABSTRACT

A computer model was developed that combined TMY-SOLMET weather data with a computer model of an air dehumidifier utilizing a rotating bed of silica gel regenerated by solar energy, and includes an economic analysis of the entire apparatus. The computer model describing the dehumidifier performance was previously found to acceptably predict the performance of an experimental solar dehumidifier, but required hourly insolation and corresponding meteorological data. The TMY-SOLMET weather computer tape readily supplies credible data at a variety of United States Weather stations. Once the performance of the solar dehumidifier is determined at a particular location, time of year and length of operation, the economic analysis can be done. The economic analysis calculates the life cycle cost of a conventional dehumidifier and a solar dehumidifier augmented with a conventional dehumidifier serving as a backup, and includes the cost of fuel and all costs associated with the equipment. For a fifteen year study of a small corn dryer in Omaha, Nebraska for five days during October, it was found that the life cycle cost of the solar dehumidifier system is about \$500 higher than that of the conventional system alone.