

GRADUATING THESIS.

TESTS ON THE EFFICIENCY OF THE  
POWER PLANT AT THE KANSAS STATE AGRICULTURAL COLLEGE.

BY

GLICK FOCKELE,

MURRAY S. COLE.

## TESTS ON THE EFFICIENCY OF THE

### POWER PLANT AT THE KANSAS AGRICULTURAL COLLEGE.

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The plant is designed to furnish heat and power for the various departments at the Kansas State Agricultural College of Manhattan, Kansas. It was originally intended by the writers to include in their work, tests on the heating power of the plant, but the arrangement of their other college work during the year, made it impossible for them to conduct the experiments at a time when the heating plant was in operation. Another drawback was the absence of suitable apparatus for the determination of the calorific power of the coal.

The heating plant consists of six externally fired, horizontal, multitubular boilers, designated as No. 1, 2, 3, 4, 5 and 6. Five are of 60 nominal horsepower and one, No. 6, is of 100 nominal horsepower. The tests were conducted upon No. 1, for the reason that it was the only high pressure boiler which was piped for individual feed water, and with this boiler is our interest concerned. None of the connections with other boilers could possibly affect results of these tests except in the test on April 28th, when No. 3 was turned in on account of the heavy load on the engine. In this case neither boiler made more steam than the engine could use, so the pressure in neither one was affected in the least and the conditions of the test were kept uniform. The boiler itself is 54" diameter, with 34 flues  $3\frac{3}{4}$  inches in diameter and 16 feet long. The grates are  $4\frac{1}{2}$ ' by  $4\frac{1}{2}$ '. The boiler was built by Joseph Bromich of Topeka, Kansas. It was installed in 1898. It was in very good condition for all tests.

The fuel used was the coal furnished by the state for use in the institution, and was of very poor quality. It comes from the state mines near Leavenworth, and is mined by the prisoners from the state penitentiary.

The engine tested was the one in general use as a source of power at the plant and was manufactured by A. L. Ide & Sons, of Springfield, Illinois. It is generally called the "Ideal," and is of 100 horsepower, and is direct connected with a 60 kilowatt dynamo. All apparatus used, such as thermometers and gauges, were standard and were carefully tested as to correctness. By comparing the tachometer with an ordinary speed counter we found an error of nine revolutions per minute less than actual speed. The tabulated results are the corrected results.

The first boiler test was run on February 14, 1902. The steam generated was used in the heating system of the college. This test was run under the direction of W. M. Sawdon, Assistant in Engineering. The per cent of water was determined as recommended in "Smart's Laboratory Practice," the weight before exposure to heat being 14 lbs., 10-1/4 oz. and afterwards 13 lbs., 15-1/8 oz., showing a loss of 11-1/8 oz., or 4-3/4 per cent water.

Following are the data and results of the boiler tests. On the general forms:

- Item 5 = 3 — (4 x 3) + 6
- " 7 = 6 ÷ 3
- " 9 =  $\frac{\text{amount of water in steam} + \text{steam condensed}}{\text{steam condensed.}}$
- " 10 = 8 x  $\frac{\text{total heat (supposing all to be vaporized)}}{\text{heat of vaporization.}}$
- " 11 =  $\frac{8 \times \text{heat of liquid} + 8 \times 9 \times \text{heat of vaporization}}{\text{heat of vaporization.}}$
- " 14 = 8 ÷ 2
- " 15 = 11 ÷ 2
- " 16 = 15 ÷ area of surface
- " 23 = 15 ÷ 34-1/2 lbs.
- " 25 = 23 ÷ 24

Item 26 =  $8 \div 3$

" 27 =  $11 \div 3$

" 28 =  $27 \div (100 \% - 4)$

" 29 =  $11 \div 5$

We will here take the opportunity of thanking Messrs.

D. V. Corbin and Alexis J. Reed, of the Junior class, for the service rendered us in the taking of data on the boiler tests and also Messrs. C. E. Peterson and J. O. Baylor for their efficient services as firemen during the progress of the tests.

## DATA AND RESULTS OF EVAPORATIVE TESTS.

Arranged in accordance with the short form advised by the boiler test committee of the American Society of Mechanical Engineers.

Grate surface.....	20 $\frac{1}{4}$	Sq. Ft.
Water heating surface.....	606	" "
Ratio of grate to water heating surface.....	1:30	
Kind of fuel.....	LEAVENWORTH	

### TOTAL QUANTITIES.

1. Date of trial.....	FEB. 24, 1902	
2. Duration of trial in hours.....	6	
3. Weight of coal fired.....	1871	LBS.
4. Per cent of water in coal.....	4 $\frac{3}{4}$	
5. Total weight of coal consumed, in pounds.....	1550	
6. Total ash and refuse.....	232	LBS.
7. Percentage ash and refuse in coal.....	12.5	
8. Total weight of water fed to boiler.....	9930 $\frac{1}{2}$	LBS.
9. Condition of steam.....	90.8	%
10. Equivalent water apparently evaporated from and at 212° F.....	11,059	LBS.
11. Equivalent water actually evaporated from and at 212° (corrected for moisture in steam).....	11,041.7	"

### HOURLY QUANTITIES.

12. Pounds coal consumed per hour.....	258 $\frac{1}{2}$	
13. Coal per hour per square foot of grate surface.....	12.7	LBS.
14. Pounds feed water per hour.....	1655	"
15. Equivalent water evaporated per hour from and at 212° F. (corrected for moisture in steam).....	1840	"
16. Equivalent water evaporated per square foot of heating surface per hour.....	2.76	"

### AVERAGE BOILER PRESSURES, TEMPERATURES, ETC.

17. Average boiler pressure, pounds per square inch, gage.....	59.8	
18. Average boiler pressure, pounds per square inch, absolute.....	74.5	
19. Average temperature of feed water, ° Fahrenheit.....	49.8	
20. Average temperature of boiler room, ° Fahrenheit.....	65.16	
21. Average temperature of outside air, ° Fahrenheit.....		
22. Barometer.....		

### HORSE-POWER.

23. Horse-power developed.....	53.3	
24. Builders' rated horse-power.....	60	
25. Percentage of builders' rated horse-power.....	89 $\frac{1}{2}$	

### ECONOMIC RESULTS.

26. Water apparently evaporated per pound of coal under actual conditions.....	5.3	LBS.
27. Equivalent water actually evaporated from and at 212° F., per pound of coal fired.....	5.9	"
28. Equivalent evaporation from and at 212° F., per pound of dry coal.....	6.2	"
29. Equivalent evaporation from and at 212° F., per pound of combustible.....	7.1	"





## DATA AND RESULTS OF EVAPORATIVE TESTS.

Arranged in accordance with the short form advised by the boiler test committee of the American Society of Mechanical Engineers.

Grate surface.....	20 $\frac{1}{4}$ SQ. FT.
Water heating surface .....	606 .....
Ratio of grate to water heating surface .....	1:30
Kind of fuel .....	LEAVENWORTH

### TOTAL QUANTITIES.

1. Date of trial.....	APR. 19, 1902
2. Duration of trial in hours .....	8
3. Weight of coal fired.....	398.5 LBS
4. Per cent of water in coal .....	4 $\frac{3}{4}$
5. Total weight of coal consumed, in pounds .....	2940.21
6. Total ash and refuse.....	855 $\frac{1}{2}$ LBS.
7. Percentage ash and refuse in coal .....	21 $\frac{1}{2}$
8. Total weight of water fed to boiler .....	1820.6 LBS.
9. Condition of steam .....	87%
10. Equivalent water apparently evaporated from and at 212° F....	2157.1 LBS.
11. Equivalent water actually evaporated from and at 212°F (corrected for moisture in steam)...	1641.27 "

### HOURLY QUANTITIES.

12. Pounds coal consumed per hour.....	367 $\frac{1}{2}$
13. Coal per hour per square foot of grate surface .....	18.14 LBS.
14. Pounds feed water per hour .....	2275 $\frac{3}{4}$
15. Equivalent water evaporated per hour from and at 212° F. (corrected for moisture in steam)...	2051.6 LBS.
16. Equivalent water evaporated per square foot of heating surface per hour.....	3.87 "

### AVERAGE BOILER PRESSURES, TEMPERATURES, ETC.

17. Average boiler pressure, pounds per square inch, gage .....	65 $\frac{3}{11}$
18. Average boiler pressure, pounds per square inch, absolute .....	79.97
19. Average temperature of feed water, ° Fahrenheit .....	55.8
20. Average temperature of boiler room, ° Fahrenheit.....	69.2
21. Average temperature of outside air, ° Fahrenheit.....	51.58
22. Barometer.....	

### HORSE-POWER.

23. Horse-power developed .....	59 $\frac{1}{2}$
24. Builders' rated horse-power.....	60
25. Percentage of builders' rated horse-power .....	99 $\frac{1}{8}$

### ECONOMIC RESULTS.

26. Water apparently evaporated per pound of coal under actual conditions .....	4.5 LBS.
27. Equivalent water actually evaporated from and at 212° F., per pound of coal fired.....	4.1 "
28. Equivalent evaporation from and at 212° F., per pound of dry coal.....	4.3 "
29. Equivalent evaporation from and at 212° F., per pound of combustible.....	5.58 "





# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

MADE AT THE COLLEGE POWER PLANT  
 MANHATTAN, KANSAS  
 ON BOILER No. 1,  
 DATE APRIL 14, 1902

## PRIMING LOG. (SEPARATING CALORIMETER.)

OBSERVERS:

GLICK FOCKELE,  
 MURRAY S. COLE

No.	Time.		Scale reading.		Amount of water in steam.	Weight on scales.				Condensed steam.	Quality of steam.	Remarks.	
	Start.	Stop.	Start.	Stop.		Start.	Stop.	Start.	Stop.				
			oz.	oz.	oz.	lbs.	oz.	lbs.	oz.	lbs.	oz.		
1	9:4	9:8	3	5	2	15	14	16	10½	0	12½	86%	
2	9:35	9:38½	3	4	1	20	11	21	6¼	0	12¼	91½%	
3	10:6½	10:10½	2.5	5	2.5	19	1¾	20	0¼	0	19½	85%	
4	11:4½	11:8½	2	4.5	2.5	19	2¼	19	15	0	12¾	83¾%	
5	12:6	12:11	3.3	7.5	4.2	18	5½	19	7¼	1	1¾	80.86%	
6	1:34	1:38	3.3	4.2	0.9	18	12	19	9¼	0	13¼	93.7%	
7	2:3	2:7	2	4.9	2.9	21	2¾	22	0	0	13¼	82%	
8	2:52	2:56	2.4	5.9	3.4	21	0½	21	15	0	14½	80½%	
9	3:33	3:36	2.6	3.2	0.6	18	10	19	3¾	0	9¾	94½%	
10	4:21	4:25	3.3	5	1.7	21	13½	22	11	0	13½	88¾%	
												AVERAGE	87%

## DATA AND RESULTS OF EVAPORATIVE TESTS.

Arranged in accordance with the short form advised by the boiler test committee of the American Society of Mechanical Engineers.

Grate surface.....	20 <sup>1</sup> / <sub>7</sub> SQ. FT.
Water heating surface.....	606 " "
Ratio of grate to water heating surface.....	1:30
Kind of fuel.....	LEAVENWORTH

### TOTAL QUANTITIES.

1. Date of trial.....	APRIL 28, 1902
2. Duration of trial in hours.....	7
3. Weight of coal fired.....	2445 LBS
4. Per cent of water in coal.....	4 <sup>3</sup> / <sub>4</sub> %
5. Total weight of coal consumed, in pounds.....	1934.36
6. Total ash and refuse.....	349.5 "
7. Percentage ash and refuse in coal.....	14 <sup>5</sup> / <sub>8</sub> %
8. Total weight of water fed to boiler.....	13,218 "
9. Condition of steam.....	84%
10. Equivalent water apparently evaporated from and at 212° F.....	15,882.4
11. Equivalent water actually evaporated from and at 212° (corrected for moisture in steam).....	11,548

### HOURLY QUANTITIES.

12. Pounds coal consumed per hour.....	276.3
13. Coal per hour per square foot of grate surface.....	13.64
14. Pounds feed water per hour.....	1888.28
15. Equivalent water evaporated per hour from and at 212° F. (corrected for moisture in steam).....	1649.7 LBS.
16. Equivalent water evaporated per square foot of heating surface per hour.....	2.7 "

### AVERAGE BOILER PRESSURES, TEMPERATURES, ETC.

17. Average boiler pressure, pounds per square inch, gage.....	65.34
18. Average boiler pressure, pounds per square inch, absolute.....	80.04
19. Average temperature of feed water, Fahrenheit.....	56.3
20. Average temperature of boiler room, Fahrenheit.....	77
21. Average temperature of outside air, Fahrenheit.....	67.55
22. Barometer.....	

### HORSE-POWER.

23. Horse-power developed.....	47.8
24. Builders' rated horse-power.....	60
25. Percentage of builders' rated horse-power.....	80

### ECONOMIC RESULTS.

26. Water apparently evaporated per pound of coal under actual conditions.....	5.4 LBS.
27. Equivalent water actually evaporated from and at 212° F., per pound of coal fired.....	4.72 "
28. Equivalent evaporation from and at 212° F., per pound of dry coal.....	4.9 "
29. Equivalent evaporation from and at 212° F., per pound of combustible.....	5.9 "

# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

TEST MADE AT THE COLLEGE POWER PLANT.

OBSERVERS:

MANHATTAN, KANSAS

## LOG OF BOILER TRIAL.

ON BOILER No. 1.

GLICK FOCKELE,  
MURRAY S. COLE.

DATE APRIL 28, 1902

Time.	Pressures.		Temperatures, °F.				Weights.									Remarks.
	Barom-eter.	Steam Gage.	External Air.	Boiler Room.	Feed Water.	Flue Gases.	Coal.			Cinders.			Water.			
							Gross.	Tare.	Net.	Gross.	Tare.	Net.	Gross.	Tare.	Net.	
8:15 A.M.		72½	69.8	72.5	57.2		176	23	153				458	78	380	
8:30 "		72	63.5	73.			182	23	159				470	78	392	
8:45 "		77½	65.3	73.	56.3		179	23	156				471½	99	372½	
9:00 "		72	65.75	75.5	55.85		194	23	181				472½	95½	377	
9:15 "		67½	65.75	75.			210	23	187				476	80	396	
9:30 "		62½	65.3	76			227½	23	204½				473½	79	394½	
9:45 "		75	65.3	74			243	23	220				471	92	379	
10:00 "		70	66.2	75	55.85		215	23	192				472½	94	378½	
10:15 "		67	68.	76			267	23	244	113	23	90	473	85	388	TURNED HIGH PRES-
10:30 "		57	67.1	76			240	23	217	141½	23	118½	471½	85½	386	SURE TO DAIRY.
10:45 "		65	66.2	76			258	23	235				471½	78	393½	CLEANED FIRE.
11:00 "		67½	66.2	75.5			259	23	236				476	78	398	
11:15 "		70	66.2	75.5	56.3		210	23	187				478	78	400	
11:30 "		72½	68.	75.5	57.2		176	23	153	164	23	141	459½	78	381½	
11:45 "		66	69.8	77.	57.2		203	23	180				470	84	386	CLEANED FIRE.
12:00 M		55	71.6	78	56.3		155	23	132				473	95	378	
12:15 P.M.		52	68.9	78.5			200	23	167				467	78½	388½	
12:30 "		65	68.9	77.			265	23	242				472	76	396	
12:45 "		65	69.8	78									471	78	393	
1:00 "		65	72.5	78	56.3								477	74	403	
1:15 "		65	71.6	79.5									474	74	400	CLEANED FIRE.
1:30 "		58	71.6	80.									480	73	407	
1:45 "		57	71.6	80.									473	88	385	
2:00 "		50	70.7	79.5	57.2								484	85	399	TURNED IN NO. 3
2:15 "		57	71.6	79.									474½	74½	400	AT SAME PRES-
2:30 "		62½	72.5	80.									475	74½	400½	SURE.
2:45 "		72½	79.	71.6	57.2								475	88½	386½	
3:00 "		72	79½	71.6									475	79½	395½	
3:15 "		65	79.5	72.5									468	86	382	
				56.2									473	87	386	
													473½	80½	393	
													472	80½	391½	
				56.2									478	78½	399½	
													476	136½	339½	
TOTAL WEIGHED OUT																
WEIGHED BACK																
Total, FIRED		1895.	1958.95	2233	731.9				3445½							
Average.		65.34	67.55	77	56.3				1000½							
									2445½	418½	69	349½			13218	

# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

MADE AT THE COLLEGE POWER PLANT  
MANHATTAN, KANSAS.  
 ON BOILER No. 1.  
 DATE APRIL 28, 1902.

## PRIMING LOG. (SEPARATING CALORIMETER.)

OBSERVERS:

GLICK FOCKELE  
MURRAY S. COLE

No.	Time.		Scale reading.		Amount of water in steam.	Weight on scales.				Condensed steam.	Quality of steam.	Remarks.
	Start.	Stop.	Start.	Stop.		Start.		Stop.				
						oz.	oz.	lbs.	oz.			
1	9:4 A.M.	9:8 A.M.	0.6	6.5	5.9 oz.	18 - 6 $\frac{1}{2}$	19 - 8	1 - 1 $\frac{3}{4}$	75%			
2	9:51:30 "	9:54:30 "	3.5	8.6	5.1	18 - 5 $\frac{1}{2}$	19 - 8 $\frac{1}{2}$	1 - 3 $\frac{1}{4}$	79%			
3	10:37 "	10:40 "	3.0	5.25	2.25	18 - 9 $\frac{1}{2}$	19 - 10 $\frac{1}{2}$	1 - 1	83+			
4	11:36 "	11:39 "	3.9	9.9	6.0	15 - 6 $\frac{1}{2}$	16 - 8 $\frac{1}{2}$	1 - 1 $\frac{3}{4}$	74.7%			
5	12:24 P.M.	12:27 P.M.	2.0	3.3	1.1	19 - 9	20 - 8	0 - 15	93.7%			
6	1:34 "	1:37 "	2.35	6.6	4.25	17 - 12 $\frac{1}{2}$	18 - 11 $\frac{1}{2}$	0 - 15	77.4			
7	2:24:30 "	2:27:30 "	4.05	4.9	0.85	16 - 5 $\frac{1}{2}$	17 - 5 $\frac{1}{2}$	1 - 0	95%			
8	2:33:30	2:36:30	1.6	2.8	1.2	19 - 15 $\frac{1}{2}$	21 - 2 $\frac{1}{2}$	1 - 2 $\frac{3}{4}$	93.9%			
9	3:3:30	3:6:30	4.0	5.7	1.7	20 - 6 $\frac{1}{2}$	21 - 8	1 - 1 $\frac{1}{2}$	91.2%			
AVERAGE										84%		

The discrepancy between the calorimeter logs for the boiler tests and those of the engine tests is explained by the fact that during the latter, two boilers were in use and thus vaporization was less rapid in each boiler than in the one in use during the boiler tests, and for this reason there was less tendency to priming.

Reference to the tables following will show the high quality of steam in use during the engine tests. It was originally the intention of the writers to derive the efficiency of the piping from the quotient of the quality at the engine by the quality at the boiler, but this was impracticable for the reason that we had but one calorimeter.

As is shown on the log of the engine trial opposite, the brake load was determined from the switch board in volts and amperes, and the brake horsepower was computed from the equation

$$B. H. P. = \frac{\text{volts} \times \text{amperes}}{746}$$

In determining the indicated horsepower we made use of the equation:

$$I. H. P. = K \times M. E. P. \times r. p. m.$$

where K is the constant of cylinder, viz: .004022 and .003954 for the head and crank ends respectively. That is,

$$K (\text{crank end}) = \frac{\text{area of crank end piston in inches} \times \text{stroke in feet}}{33000}$$

$$\text{and } K (\text{head end}) = \frac{\text{area of head end piston in inches} \times \text{stroke in feet}}{33000}$$

The mean effective pressure was determined from the indicator cards in the usual way, by taking the quotient of the area by the length x strength of spring, 50 lbs. The efficiency, E, of the engine was computed as follows:

$$E = \frac{B. H. P.}{I. H. P.}$$



# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

MADE AT THE COLLEGE POWER PLANT  
 MANHATTAN, KANSAS.  
 ON THE IDEAL ENGINE  
 DATE MAY 21, 1902

## PRIMING LOG. (SEPARATING CALORIMETER.)

OBSERVERS:  
 GLICK FOCKELE  
 MURRAY S COLE.

No.	Time.		Scale reading.		Amount of water in steam.	Weight on scales.				Condensed steam.	Quality of steam.	Remarks.
	Start.	Stop.	Start.	Stop.		Start.		Stop.				
						oz.	oz.	lbs.	oz.			
1	1-21:15	1:29:15	4.3	5.9	1.6	17 - 12 $\frac{3}{4}$	18 - 7 $\frac{1}{2}$	0 - 11 $\frac{1}{4}$	87 $\frac{1}{2}$ %			
2	1:50	1:53	2.6	3.6	1.0	19 - 9 $\frac{3}{4}$	20 - 10 $\frac{1}{2}$	0 - 16 $\frac{3}{4}$	93%			
3	2:19	2:22	2.0	2.6	0.6	19 - 1 $\frac{1}{2}$	20 - 9	1 - 7 $\frac{1}{2}$	97 $\frac{1}{2}$ %			
4	2:47	2:50	5.2	6.2	1.0	19 - 19	20 - 19 $\frac{1}{4}$	1 - 0 $\frac{1}{4}$	94%			
5	3:21	3:24	3.5	4.6	1.1	18 - 15 $\frac{3}{4}$	20 - 0 $\frac{1}{2}$	1 - 0 $\frac{3}{4}$	94%			
6	3:56	3:59	7.4	7.8	0.4	18 - 12 $\frac{1}{4}$	19 - 14	1 - 1 $\frac{3}{4}$	98%			
7	4:32	4:35	6.6	7.6	1.0	20 - 2	21 - 8	1 - 6	95 $\frac{1}{2}$ %			
								AVERAGE	99.3%			



# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

## LOG OF ENGINE TRIAL.

TEST MADE AT THE COLLEGE POWER PLANT  
MANHATTAN, KANSAS.  
 ON THE IDEAL ENGINE  
 DATE MAY 31, 1902

CONSTANTS OF ENGINE.

Diameter of cylinder... 1.3 in. Area of piston H. E. .... 132.72 sq. in.  
 Diameter of piston rod... 1.76 in. Area of piston C. E. .... 130.49 sq. in.  
 Length of stroke ..... 1 ft. Engine constant for H. E., .004022  
 Engine constant for C. E., .003954

OBSERVERS:

GLICK FOCKELE  
MURRAY S. COLE

No. Card.	Time.	R. P. M.	M. E. P.		I. H. P.			Brake Load (Dynamo).		B. H. P.	Remarks.
			H. E.	C. E.	H. E.	C. E.	Total.	Volts.	Amperes.		
1	8: — AM	277	12.83	12.39	14.295	13.578	27.873	220	85	25.11	
2	8: 15	277	12.54	12.975	13.97	14.22	28.19	220	85	25.11	
3	8: 30	280	14.434	14.14	16.256	15.66	31.916	220	90	26.59	
4	8: 45	281	10.55	11.56	11.925	11.738	23.663	220	75	22.11	
5	9: —	277	11.595	13.044	12.92	14.295	27.215	219	85	24.95	
6	9: 15	281	10.64	11.66	12.026	12.96	24.986	220	68	20.05	
7	9: 30	277	9.325	9.906	10.39	10.855	21.245	219	60	17.72	
8	9: 45	280	9.621	9.468	10.835	10.987	21.322	220	60	17.69	
9	10: —	281	8.822	10.59	9.971	11.77	21.741	220	28	8.257	
10	10: 15	280	3.6445	5.976	4.1045	6.18	10.2845	219	62.5	17.93	
11	10: 30	280	9.481	11.536	10.679	12.8	23.479	221	62.5	18.51	
12	10: 45	277	10.00	11.322	11.143	12.46	23.603	220	62.5	18.43	
13	11: —	280	8.565	10.375	8.646	11.49	20.136	220	59	17.4	
14	11: 15	280	10.735	11.765	12.09	13.03	25.12	219	72.5	21.28	
15	11: 30	280	13.235	14.119	14.905	15.64	30.545	220	87	25.66	
16	11: 45	281	10.994	13.404	12.422	14.9	27.322	220	76	22.41	
17	12: —	278	11.178	11.323	12.50	12.45	24.95	220	68	20.05	
18	12: 15 P.M	273	11.195	12.538	12.294	13.54	25.834	220	70	20.64	
19	12: 30	285	2.985	6.269	3.422	7.0678	10.4898	220	22	6.488	No LOAD EXCEPT AT THE DAIRY.
20	12: 45	281	2.187	5.394	2.472	5.995	8.467	220	21	6.192	
21	1: —	281	3.0075	5.564	3.3995	6.184	9.5835	220	23	6.782	
22	1: 15	283	2.5565	5.564	2.91	6.228	9.138	220	18	5.308	
23	1: 30	279	5.294	6.765	5.9405	7.463	13.4035	220	32	9.437	POWER FOR SHOPS THROWN ON
24	1: 45	277	11.26	19.26	12.546	20.11	32.656	218	75	21.91	
25	2: —	277	8.089	9.8465	9.012	10.79	19.802	220	57	16.81	
26	2: 15	281	8.8215	9.8465	9.97	10.945	20.915	220	57	16.81	
27	2: 30	280	8.746	9.323	9.85	10.325	20.275	220	57	16.81	
28	2: 45	280	8.746	9.767	9.85	10.818	20.768	220	57	16.81	
29	3: —	277	8.455	9.184	9.42	10.073	19.493	220	56	16.52	
30	3: 15	279	8.384	10.295	9.409	10.36	20.769	220	58	17.1	
31	3: 30	277	8.746	9.622	9.745	10.543	20.288	220	60	17.69	
32	3: 45	277	8.601	9.765	9.584	10.7	20.284	220	58	17.1	
33	4: —	280	9.826	9.926	11.18	10.995	22.175	220	57	16.81	
Maximum,		285	14.434	19.26	16.256	20.11	32.656	221	90	26.54	
Minimum,		273	2.187	5.394	2.472	5.995	8.467	218	18	5.308	
Total,			305.196	333.502	330.0115	359.5468	689.5583	7255	1964	561.614	
Average,			9.248	10.106	10.0003	10.895	20.8953	219.85	59.51	17.18	
AVERAGE EFFICIENCY		82%									

# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

MADE AT THE COLLEGE POWER PLANT  
 MANHATTAN, KANSAS.  
 ON THE IDEAL ENGINE  
 DATE MAY 31, 1902

## PRIMING LOG. (SEPARATING CALORIMETER.)

OBSERVERS:  
 GLICK FOCKELE  
 MURRAY COLE

No.	Time.		Scale reading.		Amount of water in steam.	Weight on scales.		Condensed steam.	Quality of steam.	Remarks.
	Start.	Stop.	Start.	Stop.		Start.	Stop.			
1	8:49	8:52	1.5 <sup>oz.</sup>	2.8 <sup>oz.</sup>	1.3 <sup>oz.</sup>	17 - 13	18 - 8 <sup>1/4</sup>	0 - 11 <sup>1/4</sup>	89.2%	
2	9:55:30	9:59:30	2.8	4.0	1.2	19 - 3 <sup>1/4</sup>	20 - 6 <sup>1/2</sup>	1 - 3 <sup>1/4</sup>	99%	
3	10:53	10:56	2.6	3.6	1.0	21 - 3 <sup>1/4</sup>	22 - 3 <sup>1/2</sup>	0 - 13	93%	
4	11:55:30	11:58:30	2.5	3.5	1.0	20 - 4 <sup>1/2</sup>	21 - 4 <sup>1/4</sup>	0 - 15 <sup>3/4</sup>	99%	
5	1:9:30	1:12:30	2.5	3.0	0.8	18 - 6 <sup>1/2</sup>	19 - 6 <sup>1/2</sup>	0 - 16	96%	
6	2:5	2:8	4.2	4.8	0.6	19 - 2	19 - 14 <sup>1/2</sup>	0 - 12 <sup>1/2</sup>	95.5%	
7	2:49	2:52	3.3	4.4	1.1	19 - 4 <sup>1/2</sup>	20 - 7 <sup>1/4</sup>	1 - 2 <sup>3/4</sup>	99.5%	
8	3:48	3:51	2.3	3.5	1.2	18 - 3 <sup>3/4</sup>	19 - 8	1 - 4 <sup>1/4</sup>	99.5%	
								AVERAGE	93.8%	

# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

## LOG OF ENGINE TRIAL.

TEST MADE AT THE COLLEGE POWER PLANT

MANHATTAN, KANSAS.

ON THE IDEAL ENGINE

DATE JUNE 2, 1902

### CONSTANTS OF ENGINE.

Diameter of cylinder.... 1.2 in. Area of piston H. E. .... 132.23 sq. in.  
 Diameter of piston rod.. 1/8 in. Area of piston C. E. .... 130.49 sq. in.  
 Length of stroke ..... 1 ft. Engine constant for H. E., .004022  
 Engine constant for C. E., .003954.

OBSERVERS:

GLICK FOCKELE  
MURRAY COLE.

No. Card.	Time.	R. P. M.	M. E. P.		I. H. P.			Brake Load (Dynamo).		B. H. P.	Remarks.	
			H. E.	C. E.	H. E.	C. E.	Total.	Volts.	Amperes.			
1	7:30 A.M.	281	8.018	8.746	9.062	9.722	18.784	220	52.5	15.48		
2	7:45	281	6.852	8.892	7.744	9.884	17.628	220	50	14.74		
3	8:—	280	7.726	8.892	8.701	9.848	18.549	220	50	14.74		
4	8:15	277	8.309	9.329	9.258	10.223	19.481	220	55	16.22		
5	8:30	277	10.295	10.148	11.970	11.112	22.582	220	65	19.16		
6	8:45	277	10.205	10.642	11.370	11.440	22.810	220	67.5	19.91		
7	9:—	279	9.766	11.080	10.960	12.228	23.188	220	65	19.16		
8	9:15	280	7.872	9.184	8.866	10.173	19.039	218	52.5	15.34		
9	9:30	277	8.746	10.205	9.746	11.182	20.928	220	60	17.69		
10	9:45	277	9.329	10.158	10.395	11.130	21.525	220	62.5	18.43		
11	10:—	277	7.872	8.309	8.772	9.1045	17.8765	220	55	16.22		
12	10:15	275	10.205	11.080	11.335	12.050	23.385	218	70	20.45		
13	10:30	277	8.235	8.411	9.143	9.216	18.359	220	55	16.22		
14	10:45	279	12.500	11.970	14.030	12.660	26.690	220	75	22.12		
15	11:—	280	9.559	11.177	10.765	12.380	23.145	222	70	20.83		
16	11:15	277	9.411	11.177	10.487	12.247	22.734	220	60	17.69		
17	11:30	279	11.765	12.648	13.204	13.958	27.162	220	70	20.64		
18	11:45	278	9.912	10.642	11.085	11.702	22.787	220	80	23.54		
19	12:—	280	12.648	13.235	14.245	14.660	28.905	220	85	25.11		
20	12:15 P.M.	277	10.442	10.295	11.637	11.280	22.917	218	67	19.57	WISTLE BLEW FOR NOON	
21	12:30	281	2.041	5.394	2.307	5.996	8.303	220	20	5.898		
22	12:45	277	2.943	5.147	3.280	5.640	8.920	220	45	13.97		
23	1:—	281	2.9155	4.3735	3.2955	4.861	8.1565	220	23	6.783		
24	1:15	281	1.7495	4.665	1.9775	5.185	7.1625	220	20	5.898	LOAD THROWN ON AGAIN.	
25	1:30	281	4.082	6.706	4.6145	7.454	12.0685	216	35	10.13		
26	1:45	280	9.184	9.912	10.343	10.980	21.323	220	65	19.16		
27	2:—	277	9.330	10.250	10.397	11.230	21.627	220	60	17.69		
28	2:15	279	7.581	9.767	8.509	10.780	19.289	220	50	14.74		
29	2:30	279	7.247	9.855	8.132	10.855	18.987	220	50	14.74		
30	2:45	280	8.840	10.000	9.956	10.075	20.031	220	57.5	17.45		
31	3:—	281	7.682	9.275	8.682	10.310	19.992	220	52.5	15.48		
32	3:15	280	8.746	11.225	9.843	11.325	21.168	220	70	20.64		
33	3:30	277	12.320	12.900	13.730	14.135	27.865	220	75	22.12		
	Maximum,	281	12.648	13.235	14.245	14.66	28.905	222	85	25.11		
	Minimum,	277	1.7495	5.147	1.9775	4.861	7.1625	216	20	5.898		
	Total,		274.329	316.189	308.341	345.0255	653.366	7252	1890	568.009		
	Average,		8.313	9.581	9.343	10.455	19.798	219.7	57.27	17.212		
	AVERAGE EFFICIENCY		86.9%									

# DEPARTMENT OF MECHANICAL ENGINEERING, K. S. A. C.

MADE AT THE COLLEGE POWER PLANT  
MANHATTAN, KANSAS.  
 ON THE IDEAL ENGINE  
 DATE JUNE 2, 1902.

## PRIMING LOG. (SEPARATING CALORIMETER.)

OBSERVERS:

GLICK FOCKELE  
MURRAY S. COLE.

No.	Time.		Scale reading.		Amount of water in steam.	Weight on scales.				Condensed steam.	Quality of steam.	Remarks.
	Start.	Stop.	Start.	Stop.		Start.		Stop.				
			oz.	oz.	oz.	lbs.	oz.	lbs.	oz.	lbs.	oz.	
1	7:48	7:51	3.7	4.0	0.3	19	14	21	9	1	11	99%
2	7:55	7:58	4.4	4.7	0.3	19	3½	20	5	1	1½	98.5%
3	8:56	8:59	2.3	2.8	0.5	19	12½	21	2¾	1	6¼	98%
4	10:5	10:8	1.6	2.5	0.9	18	2½	19	11½	1	9	96.5%
5	11:5	11:8	1.7	2.8	1.1	20	9¾	21	15	1	5¼	95%
6	12:5	12:8	1.3	1.9	0.6	19	19¼	21	4¾	1	6½	97.5%
7	1:7	1:10	3.1	3.5	0.4	20	2½	21	5½	1	3	98%
8	2:20	2:23	2.3	2.9	0.6	19	0½	20	8½	1	8	97.5%
										AVERAGE		97.5%