ANALYSIS OF INDETERMINATE STRUCTURES

BY COMBINING REDUNDANTS

by

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ANALYSIS OF STATICALLY INDETERMINATE STRUCTURES

BY COMBINING REDUNDANTS

By SURENDRA B. DAULAT¹ SYNOPSIS

In the analysis of highly indeterminate structures, the task of setting up and solving the elastic equations becomes timeconsuming. Several methods have been developed to reduce the work considerably. Analysis by combining redundants is one of them. Each method has its own advantages. Two illustrative examples are solved to compare the combined redundant method with the method of consistent deformations.

INTRODUCTION

In the study of indeterminate structural analysis the usual procedure is to solve simultaneous equations obtained by the method of consistent deformations. The number of equations involved is equal to the degree of indeterminancy of the structure. For high degrees of indeterminancy, the solution of the simultaneous ecuations becomes a tedious task. The combined redundant method eliminates the time-consuming procedure of solving these simultaneous equations. The purpose of this study was to become familiar with this method of analysis and its applications to structural analysis problems. To illustrate this method, three numerical examples are solved. First, a truss with three degrees

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of indeterminancy (7), second, a flexural beht with three degrees of indeterminancy, and third, a truss, indeterminate to the eighth degree. To compare this method with another method, problems 2 and 3 are also solved by the method of consistent deformations.

The usual procedure for the analysis of indeterminate structures is to select a statically determinate structure by removal of redundant forces. This is done by cutting through the members, inserting hinges and removing reaction components. The number of these redundant forces is always n for an n-times indeterminate case. A set of simultaneous equations results:

E SijXj + Sio = O

where X_j are the redundants. The subscript i and j are integers from 1 to n. The terms \hat{k} ; are coefficients depending only on the size and shape of the original structure and the terms \hat{k} are the loading terms which depend on geometry and on the load (4).

The primary purpose of the method of combining redundants is to get a system of linear equations which can very easily be solved. This involves the determination of the coefficients of the "combined redundants". After these coefficients have been found, member stresses corresponding to the given condition of loading or distortion may be evaluated. These coefficients are independent of the type of loading a structure has to bear (10).

OUTLINE OF THE LETHOD

The 'm' times statically indeterminate structure is first reduced to a statically determinate form by removing a number of redundants equal to the degree of indeterminancy. The system of

simultaneous equations thus obtained is:

$$x_{1} \delta_{11} + x_{2} \delta_{12} + \dots + x_{m} \delta_{1m} - \Delta_{1} = 0$$

$$\cdot x_{1} \delta_{21} + x_{2} \delta_{22} + \dots + x_{m} \delta_{2m} - \Delta_{2} = 0$$

Eq.1.

$$\cdot x_{1} \delta_{m1} + x_{2} \delta_{m2} + \dots + x_{m} \delta_{mm} - \Delta_{m} = 0$$

The proposed method, the orthogonalization of the above equations, gives the following form.

$$\begin{array}{ccc} \chi_{1}^{\prime} \delta_{11} & -\Delta_{1}^{\prime} = 0 \\ \chi_{2}^{\prime} \delta_{22}^{\prime} & -\Delta_{2}^{\prime} = 0 & \dots & \text{Eq. 2}, \end{array}$$

To establish this pattern of equations, the principle of virtual work will be used. It states that during any virtual displacement of an elastic body the net work done by all the forces is zero. The internal and external forces must be in equilibrium and the virtual displacement must be small and compatible with the condition of the constraints. (12).

For example, assume a truss which is three times statically indeterminate Fig.l(a). If the final stress in a member is denoted by S, then by the principle of superposition

 $S = x_1s_1 + x_2s_2 + x_3s_3 + S_0$ (3) where s_1, s_2, s_3 are the bar stresses due to unit value of redundants x_1, x_2, x_3 and S_0 is the bar stress due to external loading on the statically determinate structure. If the constant term L/AE for each bar is represented by \prec , then the corresponding elongation is given by

 $\Delta_{L} = \ll S = \ll (x_1 S_1 + x_2 S_2 + x_3 S_3 + S_6) - \cdots + (4)$

Considering the case of Fig. 1(c) where a unit load is applied at point H, the bar stress s₁ results. The system will be in equilibrium due to this force. If a small displacement is given to the truss, it causes a virtual displacement in all the members of the truss. By the principle of virtual work, the net work done by all the forces due to this virtual displacement is zero. Internal and external forces must be in equilibrium and the virtual displacement must be small and compatible with the condition of the constraints (12). The following equation may be written:

 $\sum \alpha S_{s_1} = 0 = x_1 \sum \alpha S_1^2 + x_2 \sum \alpha S_1 S_2 + x_3 \sum \alpha S_1 S_3 + \sum \alpha S_1 S_0$(A)

This equation contains three unknown redundant forces, x_1 , x_2 , x_3 . Applying the redundant forces as shown in Fig. 1(d) and Fig. 1(e) the bar stresses s_2 and s_3 will result. Two similar equations could be obtained as given below.

 $\sum \alpha \leq S_{1} = 0 = \chi_{1} \geq \alpha \leq S_{2} + \chi_{2} \geq \alpha \leq S_{2}^{2} + \chi_{3} \geq \alpha \leq S_{2} \leq S_{3} + \leq \alpha \leq S_{3} \leq 0$ (B)

 $\sum_{\alpha} S_{S_{3}} = 0 = x_{1} \sum_{\alpha} s_{3} s_{1} + x_{2} \sum_{\alpha} s_{5} s_{2} + x_{3} \sum_{\alpha} s_{3}^{2} + \sum_{\alpha} s_{5} s_{0}$(c)

This summation extends over all the members of the truss. The above three equations are set up in the form given in equation (1), and can be solved simultaneously for the unknown redundant forces x_1, x_2, x_3 . In the combined redundant method, the redundants chosen from Fig. 1(c) to Fig. 1(e) are combined with each other in such a manner that the quantity $\leq s_i s_j$ becomes zero, where $i \neq j$. Thus reducing the system of equations to the form given in equation (2). This method is called the combined unit redundants method as the unit redundants are combined with each other to obtain the form

of equation (2). It is to be noted that only one of the combined redundants will have a unit value. The bar stresses due to 'n' combined redundants will be denoted by S_n . The first combined redundant will be equal to one; Fig. 1(c) and corresponding bar stresses are given by

$$S_1 = S_1$$
 ... 5(a)

The second combination of redundants from Fig. 1(c) and Fig. 1(d) is made in such a way that bar stresses due to these forces are

$$S_2 = C_{21}S_1 + S_2$$
 ... 5(b)

The third combination of Fig. 1(c), (d), (e) such that bar stresses are

$$S_3 = C_{31}S_1 + C_{32}S_2 + S_3$$
 ... 5(c)

The coefficients $\rm C_{21},\ C_{31},$ and $\rm C_{32}$ are constants, yet to be determined (12).

With these new bar stresses S_1 , S_2 , and S_3 , the values of the redundant forces X_1 , X_2 and X_3 can be obtained and by the principle of superposition the final stresses are given by

$$S = S_{1}X_{1} + S_{2}X_{2} + S_{3}X_{3} + S_{6}$$
 ... (6)

It is worth noting that the redundant forces x_1 , x_2 and x_3 are quite different from x_1 , x_2 and x_3 . The corresponding elongation is given by

$$\Delta_{L} = \alpha S = \alpha (X, S, + X_2 S_2 + X_3 S_3 + S_0)$$

Applying the virtual work principle, the set of equations similar to equations (A), (B), and (C) can be written as

$$\Sigma \propto SS_{1} = 0 = X_{1} \Sigma \propto S_{1}^{2} + X_{2} \Sigma \propto S_{1} S_{2} + X_{3} \Sigma \propto S_{1} S_{3} + \Sigma \propto S_{1} S_{0} - \dots - (A_{1})$$

$$\Sigma \propto SS_{2} = 0 = X_{1} \Sigma \propto S_{2} S_{1} + X_{2} \Sigma \propto S_{2}^{2} + X_{3} \Sigma \propto S_{2} S_{3} + \Sigma \propto S_{2} S_{0} \dots (B_{1})$$

$$\Sigma \propto SS_{3} = 0 = X_{1} \Sigma \propto S_{3} S_{1} + X_{2} \Sigma \propto S_{3} S_{2} + X_{3} \Sigma \propto S_{3}^{2} + \Sigma \propto S_{3} S_{0} \dots (C_{1})$$

This summation extends over all the members of the truss. Actually, as per the scheme, one needs to establish a condition such as that

 $\sum \propto S_1 S_2 = 0 \qquad \dots \qquad 7(a)$ $\sum \propto S_1 S_3 = 0 \qquad \dots \qquad 7(b)$ $\sum \propto S_2 S_3 = 0 \qquad \dots \qquad 7(c)$

This can be done by suitably choosing the constant terms of equations (5). We have from equation 7 (a) $\Sigma \ll S_1 S_2 = O$ Substituting the value of S_2 from equation 5 (b)

Ex S, ((21 S1 + S2) = 0

02 C21 ZX S, 2+ ZX S, S2=0

$$\therefore C_{2i} = -\frac{\sum \propto S_i S_i}{\sum \propto S_i^2} \qquad \dots \qquad \otimes (\alpha)$$

Similarly, substituting the value of S_3 from equation 5 (c) in equation 7 (b)

 $\sum \propto S_1 \left(C_{31} S_1 + C_{52} S_2 + S_3 \right) = 0$ $o_{\mathcal{E}} \left(S_{31} \sum \alpha S_1^2 + C_{32} \sum \alpha S_1 S_2 + \sum \alpha S_1 S_3 = 0$ $b_{ut} \sum \alpha S_1 S_2 = 0 \quad \text{from equation 7(a)}$ $\therefore C_{51} = - \frac{\sum \alpha S_1 S_3}{\sum \alpha S_2^2} \quad \dots \quad 8(b)$ and finally substituting the value of S, in equation 7(c)

$$\Sigma \propto S_{2} \left(\mathbb{C}_{31} S_{1} + \mathbb{C}_{32} S_{2} + S_{3} \right) = 0$$

or $C_{31} \Sigma \propto S_{2} S_{1} + \mathbb{C}_{32} \Sigma \propto S_{2}^{2} + \Sigma \propto S_{2} S_{3} = 0$
but $\Sigma \propto S_{2} S_{1} = 0$ from equation 7(a)
 $\therefore C_{32} = -\frac{\Sigma \propto S_{2} S_{3}}{\Sigma \propto S_{2}^{2}} \qquad \dots 8(c)$

In order to compute the value of constant C_{1j} , first one needs to know the bar stresses s_1 , s_2 and s_3 due to a unit value of redundants applied as shown in Fig. 1(c), (d), and (e). The first combined redundant is chosen equal to one and thus the bar stress $S_1^z = s_1$. With this value of S_1 known, the coefficient C_{21} can very easily be obtained from the equation $\delta(a)$. Once the value of coefficient C_{21} is known, the magnitude of S_2 can very easily be found from the equation 5(b) and thus the coefficients C_{31} and C_{32} can be found from equation $\delta(b)$ and $\delta(c)$ and therefore the magnitude of S_3 may be computed. Once the coefficients C_{1j} are known, the bar stresses (due to combined redundants) S_1 , S_2 and S_3 can be found from equation (5) with the combined redundants as shown in Fig. 2. The bar stresses produced are S_1 , S_2 and S_3 With these values of bar stresses, the equations (A_1) , (B_1) and (C_3) reduce to

 $X_{2} \ge \propto S_{1}^{2} + \ge \ll S_{1} \le 0 \qquad \dots \qquad 9(a)$ $X_{2} \ge \propto S_{2}^{2} + \ge \ll S_{2} \le 0 \qquad \dots \qquad 9(b)$ $X_{3} \ge \propto S_{3}^{2} + \ge \ll S_{3} \le 0 \qquad \dots \qquad 9(b)$

The unknown x_1 , x_2 and x_3 , thus obtained, will give final bar stresses by substitution into equation (6). It is worth noting that the constants C_{ij} and the bar stresses S_n are independent of the type of loading, that is, with a different type of loading, one needs to calculate only equation (6) and (9). The remaining values are unaltered. (12)

GENERAL THEORY

To generalize the above methods, let there be a "m" times statically indeterminate structure. Let the internal stresses produced due to redundants x_1, x_2, \ldots, x_m be s_1, s_2, \ldots, s_m . The structure is made statically determinate by removing the redundant forces. In order that no external work will be done by interior redundants, they are cut or hinged so that equal and opposite forces or moments can be applied at points infinitely close together. Therefore, the only external work that can be done by the combined unit redundant system during its virtual displacement is due to yielding of supports. (12)

As shown in equation (5), the internal stresses S_1, S_2 . S_n produced by combining unit redundants can be put in the following pattern:

 $S_{1} = S_{1}$ $S_{2} = C_{21}S_{1} + S_{2}$ $S_{3} = C_{31}S_{1} + C_{32}S_{2} + S_{3}$ $\cdots \cdots \cdots (10)$ $S_{m} = C_{m_{1}}S_{1} + C_{m_{2}}S_{2} + \cdots + C_{m(m-2)}S_{m-2} + C_{m(m-1)}S_{m-1} + S_{m}$

The internal stresses $s_1, s_2 \dots s_m$ are produced due to external forces like axial forces, shear forces, couples or any combination of such functions. (12)

The coefficients $C_{ij} = -\frac{\int \ll S_j s_i d_v}{\int \ll S_j^2 d_v}$ for i > jhere \ll is the constant term $\approx \frac{L}{A_E}$

where L - is the length of the member

A - is the Cross Section area of the member

E - is modulus of elasticity for a given member

With the arrangement shown in equation (10)

 $\int \ll S_i S_j d_v = 0 \quad \text{for } i \neq j. \text{ This results in an orthogonal form of the linear equations. By the principle of superposition, the internal stresses in the actual structure is given by$

 $S \in X, S, +X, S, +X, S_3 + \cdots + X_m S_m + S_o - (11)$ where S_0 is the stress due to external loading on the statically determinate structure.

Corresponding elongation is given by

 $\delta = \propto S + \delta_t = \propto (X, S, +X, S_2 + X_3 S_3 + \cdots X_m S_m + S_0) + \delta_t$ where δ_t represents internal strain due to temperature changes, due to loosening of connection, etc. For flexural members, that is, members subjected to pure bending, the term δ_t is neglected as its magnitude is negligible in comparison with stresses due to bending.

Applying the principle of virtual work, the unknowns $X_1, X_2 \cdots X_m$ can be determined from the following equations.

$$X_{1}\int_{\mathcal{A}}S_{1}^{2}d_{V} + \int_{\mathcal{A}}S_{0}S_{1}d_{V} + \int_{\mathcal{S}}\delta_{t}S_{1}d_{V} + \sum R_{1}P\Delta_{P} = 0$$

$$X_{2}\int \propto S_{2}^{2} dv + \int \propto S_{0} S_{2} dv + \int \delta_{t} S_{2} dv + \Sigma R_{2p} \Delta_{p}^{\pm} O \qquad (12)$$

$$X_{m} \int \propto S_{m}^{2} dv + \int \propto S_{0} S_{m} dv + \int \delta_{t} S_{m} dv + \Sigma R_{m} \rho \Delta_{p}^{\pm} O$$

where $\Sigma \hat{R}_{if} \Delta_{\rho}$ denotes the external work done due to settlement of the pth support, which is equal to the external force R_{ip} at the pth support due to combined redundant at point 1 multiplied by the settlement Δ_{ρ} of the pth support. Let the external force or moment at p, for the system s_n , be r_{np} and for the system S_n , be R_{np} . It can be seen that these are the same for the system S_1 as for the system s_1 that is $R_{1p} = r_{1p}$. Writing these equations in a form similar to equation (10)

from equation (13) values of R_{ip}; i=1,2...n, can be determined and if substituted in equation (12), unknown redundants $X_1, X_2, ...$ X_n can be determined. (12)

Actually the terms representing strain due to temperature changes and support settlements are neglected since these cause a stress which is of opposite nature to the stress due to live-loads. Moreover, its magnitude is negligible as compared to that due to live loads.

COMBINED REDUNDANTS FOR FLEXURAL STRUCTURES

The method of combined redundants applied to structures in pure bending, compares favourably with result obtained by the elastic center method. For such structures the work done by the axial force andby shear or torsional force is neglected as it is of very small magnitude in comparison with the work done by bending. For the sake of illustration, let us assume a bent with fixed end support as shown in Fig. 3(a).

The given structure is made statically determinate as shown in Fig. 3(b). The three unknown redundants are moment, horizontal force and vertical force. These forces and the moment are applied at the free end 'A'. These forces at the free end will cause moments at each section of the given bent.

Let the moments at any point be $m_1 = 1$; $m_2 = x$; $m_3 = y$. Now as per equation (10), the combined unit redundants will be

$$M_{1} = m_{1} = 1$$

$$M_{2} = C_{21} M_{1} + m_{2} = C_{21} + m_{2} = C_{21} + \pi$$

$$M_{3} = C_{31} M_{1} + C_{32} M_{2} + m_{3} = C_{31} + C_{32} (C_{21} + \pi) + \frac{1}{2}$$
(14)

Here the coefficient \textbf{C}_{ij} , (i \neq j) has a meaning similar to that for truss members.

$$C_{21} = -\frac{\int M_1 m_2 \frac{ds}{EI}}{\int M_1^2 \frac{ds}{EI}}.$$

where 1/21 is constant for flexural member. From equation (14)

$$c_{21} = -\frac{\int m_1 m_2 \frac{d_s}{EI}}{\int m_1^2 \frac{d_s}{EI}} = -\frac{\int x \frac{d_s}{EI}}{\int \frac{d_s}{EI}} \qquad (.15)a$$

Coefficient C

$$31 - \frac{\int_{m_1^2} \frac{c_{l_1}}{E_1}}{\int_{m_2^2} \frac{d_s}{E_1}} = -\frac{\int_{m_1^2} \frac{d_s}{E_1}}{\int_{m_1^2} \frac{d_s}{E_1}}$$

$$g_{32} = -\frac{\int_{m_1^2} \frac{d_s}{E_1}}{\int_{m_2^2} \frac{d_s}{E_1}} = -\int_{m_1^2} \int_{m_1^2} \frac{d_s}{E_1}$$

umerator is $= -\int_{m_1^2} \int_{m_1^2} \frac{d_s}{E_1}$

For C_{32} , the numerator is $= \int (C_{21} + \infty) \Im \frac{d_S}{EI}$.

$$= \left\{ \left\{ -\frac{\int x \frac{ds}{EI}}{\int \frac{ds}{EI}} + x \right\} y \frac{ds}{E} \right\}$$
$$= \left\{ -\frac{\int x \frac{ds}{EI} + x \int \frac{ds}{EI}}{\int \frac{ds}{EI}} \right\} y \frac{ds}{E}$$
$$= \left\{ -\frac{\int x \frac{ds}{EI} + x \int \frac{ds}{EI}}{\int \frac{ds}{EI}} \right\} y \frac{ds}{E}$$

and the denominator is

$$= \int (c_{21} + x)^{2} \frac{ds}{ET}$$

$$= \int \left\{ Cc_{21} \right\}^{2} + 2 \cdot (c_{21} \cdot x + x^{2}) \frac{ds}{ET}$$

$$= \int \left\{ \left(-\frac{x \int \frac{ds}{ET}}{\int \frac{ds}{ET}} \right)^{2} - \frac{2x \int \frac{ds}{ET}}{\int \frac{ds}{ET}} \cdot x + x^{2} \right\} \frac{ds}{ET}.$$

$$c_{32} = -\frac{\left\{\left(\int x \frac{d_s}{E_1}\right)^2 - 2\int x \frac{d_s}{E_1} \cdot \int \frac{d_s}{E_1} \cdot x \cdot x + x^2 \int \left(\frac{d_s}{E_1}\right)^2\right\} \frac{d_s}{E_1}}{\left\{\int \left(x \frac{d_s}{E_1}\right)^2 - \int x^2 \frac{d_s}{E_1}}$$
$$= \frac{\left\{\int \left(x \frac{d_s}{E_1}\right)^2 - \int x^2 \frac{d_s}{E_1}\right\}}{\left(\int x \frac{d_s}{E_1}\right)^2 - \int \left(x^2 \frac{d_s}{E_1}\right) \left(\int y \cdot \frac{d_s}{E_1}\right)\right\} / \int \frac{d_s}{E_1}}{\left\{\int \left(x \frac{d_s}{E_1}\right)^2 - \int \left(x^2 \frac{d_s}{E_1}\right)\right\} / \int \frac{d_s}{E_1}}$$

Multiplying the denominator and numerator by (-1)

$$C_{32} = -\frac{\int x y \frac{ds}{EI} - \left(\int x \frac{ds}{EI}\right) \left(\int y \frac{ds}{EI}\right) \left(\int \frac{ds}{EI}\right)}{\int x^2 \frac{ds}{EI} - \left(\int x \frac{ds}{EI}\right)^2 \left(\int \frac{ds}{EI}\right)}$$
(15)0

With these known values of coefficients C_{21} , C_{31} and C_{32} the moments M_1 , M_2 and M_3 can be found from the equation (14) and the multiplier X_1 , X_2 and X_3 can very easily be found from equation (12) where $S_1 = M_1$; $S_2 = M_2$ etc. M_0 is the bending moment due to the external load on the statically determinate structure. The integration of the bending moment is over the whole structure.

Actually the bending moment diagram is split up into various parts, rectangles, triangles, trapezium etc. for which ready-made integration formulas are available (6).

PHYSICAL MEANING OF THE METHOD

To demonstrate the physical meaning of the given method, assume a continuous beam with four spans as shown in Fig. 4(a). This structure is statically indeterminate to the third degree.

For the sake of illustration let the three redundants be the reactions applied at points 1, 2, and 3. By removing the intermediate supports, a simply supported beam will result with its deflection curve as shown in Fig. 4(b). Thisrepresents the state of S_0 , that is the deflection due to applied loading. Applying a unit redundant at point (1), the deflection curve will be as shown in Fig. 4(c). Fig. 4(d) represents the deflection curve which is due to the combination of redundants at the point (1) and (2). It may be observed that the selection of the coefficient C_{21} is such as to cause the deflection at point (1) to be equal to zero. The deflection curve shown in Fig. 4(e) and 4(d) plus that due to the unit redundant at point (3). Thus in Fig. 4(e) the coefficients C_{31} and C_{32} nullify the deflection at points (1) and (2).

Now, from state S_1 , S_2 and S_3 , only state S_1 causes the .flection at point (1) but the multiplier X_1 is so chosen that the deflection at point (1) becomes equal to zero (due to state S_0). The state S_2 causes the deflection at point (2) but multiplier X_2 nullifies the deflection at point (2) (due to states $S_0 + X_1 S_1$). Similarly the scheme is extended for other redundants. The superposition of all the different states is shown in Fig. 4(f) (Ref. 12).

RELATION TO OTHER METHODS

The application of this method to flexural structures has been established. It can be shown that this method is very closely related to the well-known elastic center and column analogy methods.

To illustrate this, consider the same bent as shown in Fig. 3(a). The structure is made statically determinate by making end 'A' free. Applying three redundants for moment, horizontal force and vertical force such that $m_1 = 1$; $m_2 = x$ and $m_3 = y$. Rewriting the equation (1⁴).

$$M_{1} = m_{1} = 1$$

$$M_{2} = C_{21}M_{1} + m_{2} = C_{21} + x$$

$$M_{3} = C_{31}M_{1} + C_{32}M_{2} + m_{3} = C_{31} + C_{32}(C_{21} + x) + y$$
The coefficients:
$$C_{21} = -\frac{\int x \frac{d_{s}}{E_{1}}}{\int \frac{d_{s}}{E_{1}}}$$

$$C_{31} = -\frac{\int y \cdot \frac{d_{s}}{E_{1}}}{\int \frac{d_{s}}{E_{1}}} - \left(\int x \cdot \frac{d_{s}}{E_{1}}\right) \left(\int y \cdot \frac{d_{s}}{E_{1}}\right) / \left(\int \frac{d_{s}}{E_{1}}\right)}$$

$$C_{32} = -\frac{\int x \cdot y \frac{d_{s}}{E_{1}} - \left(\int x \cdot \frac{d_{s}}{E_{1}}\right) \left(\int \frac{d_{s}}{E_{1}}\right) / \left(\int \frac{d_{s}}{E_{1}}\right)}{\int x^{2} \frac{d_{s}}{E_{1}} - \left(\int x \cdot \frac{d_{s}}{E_{1}}\right)^{2} / \left(\int \frac{d_{s}}{E_{1}}\right)}$$

In the elastic center method, the point of application of redundants is so chosen that

$$\int x_{i} \frac{ds}{EI} = 0$$

$$\int \frac{y_1}{EI} \frac{ds}{EI} = 0$$

$$\int x_1 y_1 \frac{ds}{EI} = 0$$

where x_1 and y_1 are the distances to the center of gravity of the closed structure.

 $\int X_{1} \frac{d_{S}}{EI} \text{ and } \int y_{1} \frac{d_{S}}{EI} \text{ represent the statical moments of the area}$ about Y-axis and X-axis respectively from the fixed point 'A' Fig. (3); (1).

Therefore, coefficient
$$-C_{21} = \int_{\frac{C_1}{E_1}}^{\frac{C_2}{E_1}} = \overline{x} \dots 15(a_1)$$

where ds represents the length of thesmall element and l/EI the width of that small element.

Therefore
$$\int \frac{cl_s}{EI} = Area ext{ of the whole structure.}}$$

Similarly $-C_{31} = \frac{\int y \frac{cl_s}{EI}}{\int \frac{ds}{EI}} = y$ $15(b_1)$

From equation 15)c) it can be seen that the numerator

 $\int x \ y \ \frac{cl_S}{E_I} - \left(\int \frac{x}{E_I}\right) \left(\int \frac{y \ cl_S}{E_I}\right) / \left(\int \frac{cl_S}{E_I}\right) / \left(\int \frac{cl_S}{E_I}\right) \ represents the product of inertia about the X_2-Y_2 axis and the denominator$

$$\int \frac{\chi^2 \operatorname{cls}}{E \operatorname{I}} - \left(\int \frac{\pi}{E \operatorname{I}} \right)^2 \left/ \left(\int \frac{\operatorname{cls}}{E \operatorname{I}} \right) \right|$$
 represents the moment of

inertia about the Y2-axis.

t

hus
$$-c_{32} = \frac{\int x_2 y_1 \frac{c_1}{E_1}}{\int x_2^2 \frac{c_2}{E_1}} = \frac{I_{x_2} y_2}{I_{y_1}} \qquad \dots \qquad 15(c_1)$$

If the redundant forces and moment are applied at the origin of principal axis then,

$$C_{21} = - \frac{\int x_1 \frac{cl_s}{E_1}}{\int \frac{cl_s}{E_1}} = 0$$

$$C_{31} = - \frac{\int y_1 \frac{ds}{E_1}}{\int \frac{cl_s}{E_1}} = 0$$

$$C_{32} = - \frac{\int x_1 y_1 \frac{ds}{E_1}}{\int \frac{cl_s}{E_1}} = 0$$

The coefficient C_{32} becomes zero on the centroidal axis if the given structure is symmetrical as shown in Fig. 3 (d). Thus the elastic center method becomes a special case of the proposed method.

Rewriting equation (14) we have,

With these known values of M_1 , M_2 and M_3 , multipliers X_1 , X_2 and X_3 can very easily be found from equation (9) that is,

$$X_{1} = -\frac{\int M_{o} \frac{cl_{s}}{E_{1}}}{\int \frac{cl_{s}}{E_{1}}} \dots \dots 16(a)$$

$$X_{2} = -\frac{\int M_{b} M_{o} \frac{cl_{s}}{E_{1}}}{\int M_{b} \frac{M_{o} \frac{cl_{s}}{E_{1}}}{E_{1}}} \dots \dots 16(b)$$

$$X_{3} = -\frac{\int M_{b} M_{o} \frac{cl_{s}}{E_{1}}}{\int M_{3}^{2} \frac{cl_{s}}{E_{1}}} \dots \dots 16(c)$$

The final moment at any point is given by

To show the relation of the method of combining redundants with the well-known column analogy method:

$$\int M_{0} \frac{ds}{E_{1}} = P \quad : \quad \int \frac{ds}{E_{1}} = A$$

$$\int x_{2} y_{2} \frac{ds}{E_{1}} = I_{x_{2}y_{2}} \quad : \quad \int x_{2}^{2} \frac{ds}{E_{1}} = Iy_{2} \quad : \quad \int y_{2}^{2} \frac{ds}{E_{1}} = I_{x_{2}}$$

$$\int M_{0} x_{2} \frac{ds}{E_{1}} = M_{y_{2}} \quad : \quad \int M_{0} y_{2} \frac{ds}{E_{1}} = M_{x_{2}}$$
(18)

Simplifying equation (16)

From the equation 15(a,)

$$X_{2} = -\frac{\int M_{0} x_{2} \frac{d_{S}}{E_{1}}}{\int x_{3}^{2} \frac{d_{S}}{E_{1}}} = -\frac{M_{y_{2}}}{I_{y_{2}}} \dots 19(b)$$

$$X_{3} = -\frac{\int ((z_{31} + (z_{32} (z_{21} + (z_{32} x + y)) M_{0} \frac{d_{S}}{E_{1}}))}{\int ((z_{31} + (z_{32} (z_{21} + (z_{32} x + y)) \frac{d_{S}}{E_{1}}))} \frac{M_{0} \frac{d_{S}}{E_{1}}}{\int (z_{31} + (z_{32} (z_{21} + (z_{32} x + y)) \frac{d_{S}}{E_{1}}))}$$

$$X_{3} = -\frac{\int \{-y + (-\frac{I_{x_{2}y_{2}}}{I_{y_{2}}})(-\bar{x}) + (-\frac{I_{x_{2}y_{2}}}{I_{y_{2}}}) \frac{M_{0} \frac{d_{S}}{E_{1}}}{x + y}\}}{\int \{-\bar{y} + (-\frac{I_{x_{1}y_{2}}}{I_{y_{2}}})(-\bar{x}) + (-\frac{I_{x_{1}y_{2}}}{I_{y_{2}}})x + y\}^{2} \frac{d_{S}}{E_{1}}}$$

$$\mathbf{x}_{3} = -\frac{\left\{\frac{M_{x_{2}}-I_{x_{2}}\mathbf{y}_{2}}{I_{y_{2}}}, \frac{M_{y_{2}}}{I_{y_{2}}}\right\}}{\left\{\frac{I_{x_{2}}-2I_{x_{2}}\mathbf{y}_{2}}{I_{y_{2}}}+\frac{I_{x_{2}}\mathbf{y}_{2}}{I_{y_{2}}}\right\}} = -\left\{\frac{M_{x_{2}}-I_{x_{2}}\mathbf{y}_{2}}{I_{x_{2}}-I_{x_{2}}\mathbf{y}_{2}}\right\} \mathbf{19(0)}$$

Substituting into equation (17) from equations (14) and (19) we obtain the following equation.

$$\begin{split} \mathbf{M} &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \frac{\mathbf{M}_{y_{2}}}{\Gamma_{y_{2}}} \cdot \mathbf{x}_{2} - \left\{ \frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{1}}{T_{y_{1}}} \cdot \mathbf{M}_{y_{2}}}{\Gamma_{y_{1}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}}} \right\} \left(\mathbf{y}_{2} - \frac{\overline{\mathbf{1}}_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}} \cdot \mathbf{x}_{2} \right) \\ &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \left\{ \frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{1}}{T_{y_{2}}}}{\frac{\Gamma_{y_{2}}}{T_{y_{2}}}} \right\} - \frac{\mathbf{M}_{y_{2}} \mathbf{x}_{2}}{\Gamma_{y_{1}}} \left\{ \frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}}}{\frac{\Gamma_{y_{2}}}{T_{y_{2}}}} \cdot \frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}}}{\frac{\Gamma_{y_{2}}}{T_{y_{2}}}} \right) \left(\frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}}}{\frac{\Gamma_{y_{2}}}{T_{y_{2}}}} \cdot \frac{\mathbf{M}_{y_{2}} \mathbf{x}_{2}}{T_{y_{2}}} \right) \\ &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \left\{ \frac{\mathbf{M}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}} \mathbf{M}_{y_{2}} \mathbf{y}_{2} - \frac{\mathbf{M}_{y_{2}} \mathbf{x}_{2}}{T_{y_{2}} \cdot \mathbf{x}_{2}} (\mathbf{I}_{x_{2}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}}) + \mathbf{M}_{x_{1}} \cdot \mathbf{I}_{x_{2} \mathbf{y}_{1}} \cdot \mathbf{x}_{2} \right) \\ &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \left\{ \frac{\mathbf{M}_{x_{1}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{1}}} \mathbf{M}_{y_{2}} \mathbf{y}_{2}}{T_{x_{1}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}{T_{y_{2}}} \mathbf{M}_{y_{2}} \mathbf{y}_{2}} \cdot \mathbf{T}_{x_{1}} \mathbf{y}_{1} \mathbf{x}_{1} \mathbf{y}_{1} - \mathbf{T}_{x_{1}} \mathbf{y}_{2}} \right) \\ &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \left\{ \frac{\mathbf{M}_{x_{1}} - \frac{\Gamma_{x_{1}} \mathbf{y}_{2}}{T_{x_{1}} - \frac{\Gamma_{x_{1}} \mathbf{y}_{2}}{T_{y_{1}}} \mathbf{M}_{y_{2}} \mathbf{y}_{2}} \cdot \mathbf{T}_{x_{1}} \mathbf{M}_{y_{2}} \mathbf{y}_{2} \cdot \mathbf{T}_{x_{1}} \mathbf{T}_{x_{1}} \mathbf{y}_{1} - \mathbf{T}_{x_{1}} \mathbf{y}_{2}} \right\} \\ &= \mathbf{M}_{0} - \frac{\mathbf{p}}{\mathbf{A}} - \left\{ \frac{\mathbf{M}_{x_{1}} - \frac{\Gamma_{x_{1}} \mathbf{y}_{2}}{T_{x_{1}} - \frac{\Gamma_{x_{2}} \mathbf{y}_{2}}} \mathbf{T}_{x_{1}} \mathbf{M}_{y_{2}} \mathbf{y}_{2}} \cdot \mathbf{T}_{x_{2}} \mathbf{T}_{y_{2}} \mathbf{T}_{y_{2}}$$

$$= M_{0} - \frac{p}{A} - \left\{ \frac{M_{x_{x}} - \frac{I_{x_{x}}y_{y} - M_{y_{z}}}{I_{x_{x}} - I_{x_{x}}y_{y} + 2}}{I_{x_{y}} \right\}^{2} - \left[\frac{M_{y_{x}}x_{1}x_{x} + M_{y_{z}}x_{2} \cdot \frac{I_{x}}{I_{y_{y}}}}{I_{y_{y}} + M_{x_{x}}I_{xy_{x}}x_{x}} - \frac{I_{x}^{-1}y_{y} + M_{x_{y}}I_{xy_{x}}x_{y}}{I_{x_{y}} - I_{x_{x}}y_{y}} \right]^{2} - \frac{I_{x_{x}}^{-1}y_{y} - I_{x_{x}}y_{y}}{I_{x_{y}} - I_{x_{x}}y_{y}}}{I_{x_{x}} - I_{x_{x}}y_{y}} - \frac{I_{x_{x}}^{-1}y_{y} - I_{x_{x}}y_{y}}{I_{x_{x}} - I_{x_{x}}y_{y}}} - \frac{I_{x_{x}}^{-1}y_{y}}{I_{x_{x}} - I_{x_{x}}y_{y}}} - \frac{I_{x_{x}}^{-1}y_{y}}}{I_{x_{x}} - I_{x_{x}}y_{y}}} - \frac{I_{x_{x}}^{-1}y_{y}}}{I_{x}} - \frac{I_{x_{x}}^{-1}y_{y}}} - \frac{I_{x_{x}}^{-1}y_{y}}}{I_{x_{x}} - I_{x}}} - \frac{I_{x_{x}}^$$

$$= M_{0} - \frac{P}{A} - \left\{ \frac{M_{x_{k}} - \frac{I_{x_{k}} y_{k}}{T_{y_{k}}} M_{y_{k}}}{I_{x_{k}} - \frac{I_{x_{k}} y_{k}}{T_{y_{k}}}} \right\} y_{2} - \left\{ \frac{M_{y_{k}} - \frac{I_{x_{k}} y_{k}}{T_{x_{k}}}}{I_{y_{k}} - \frac{I_{x_{k}} y_{k}}{T_{x_{k}}}} \right\} x_{2}$$

This is the general form of the column analogy method for unsym-

metrical bent. Substituting $I_{x_{k_{1}}y_{2}} = O$ that is, considering the symmetrical frame of Fig. 3(d)

$$^{M} = {}^{M}_{0} - \frac{p}{A} - \frac{M_{\chi_{\lambda}}}{I_{\chi_{\lambda}}} \cdot y_{2} - \frac{M_{y_{\lambda}}}{I_{\chi_{\lambda}}} \cdot x_{2}$$

Thus the method of combined redundants reduces to the column analogy method.

ILLUSTRATIVE PROBLEMS

To illustrate the mothod of combining redundants, several problems will be solved, involving the application of this method:

PROBLEM: 1

A truss shown in Fig. 5(a) with 12^k load applied at point H, acting downward (7). Required: Ear stresses due to the given load: Solution: The given frame is statically indeterminate to the third degree. The truss is made statically determinate by removing the intermediate support and cutting bars CH and CF.

The combined redundants are shown in Fig. 5 the required computations are framed in Table 1.

^S1 = ^S1

$${}^{C}_{21} = -\frac{\sum \propto S_{1} s_{2}}{\sum \propto S_{1}^{2}} = -\frac{\sum \propto S_{1} s_{2}}{\sum \propto s_{1}^{2}} = \frac{0.85}{3.25} = 0.262$$

$${}^{S}_{2} = {}^{C}_{21} {}^{S}_{1} + {}^{S}_{2} = 0.262 {}^{S}_{1} + {}^{S}_{2}$$

$$c_{31} = -\frac{\sum \propto S_1 s_3}{\sum \propto S_1^2} = -\frac{\sum \propto s_1 s_3}{\sum \propto s_1^2} = 0.262$$

$$c_{32} = -\frac{\sum \propto S_2 s_3}{\sum \propto S_2^2} = -\frac{\sum \propto (0.262 s_1 + s_2) S_3}{\sum \propto (0.262 s_1 + s_2)^2} = -0.1105$$

$$s_3 := c_{31}s_1 + c_{32}s_2 + s_3 = c_{31}s_1 + c_{32}(c_{21} + s_2) + s_3$$

$$= 0.262 s_1 - 0.1105 (0.262 s_1 + s_2) + s_3$$

With these known values of C_{21} , C_{31} , C_{32} and hence, S_1 , S_2 and S_3 , unknown redundants could be computed as follows.

$$X_{1} = -\frac{\sum \propto S_{1} S_{0}}{\sum \propto S_{1}^{2}} = -\frac{\sum \propto S_{1} S_{0}}{\sum \propto S_{1}^{2}} = \frac{22 \cdot 88}{3 \cdot 254} = 7.0650$$

$$X_{2} = -\frac{\sum \propto S_{2} S_{0}}{\sum \propto S_{1}^{2}} = \frac{20 \cdot 71}{3 \cdot 775} = 5.448$$

$$X_{3} = -\frac{\sum \propto S_{3} S_{0}}{\sum \propto S_{3}^{2}} = -\frac{1 \cdot 32}{3 \cdot 736} = -0.353$$

The final bar stresses due to the given loading are shown in column 22 of Table 1. For the sake of convenience, $\frac{L}{A}$ has been chosen equal to unity.

(L same for all members)

PROBLEM: 2

A flexural bent is shown in Fig. 6(a) with both ends fixed. The loading is as shown.

Required: Horizontal reaction due to given load and the bending moment diagram.

Solution: The given bent is statically indeterminate to the third

degree. The end A is freed and the following three redundant forces are applied; counter-clockwise moment X_1 , horizontal force X_2 and vertical force X_3 at A. The moment diagrams for these forces are shown in Fig. 6(c), (d) and (e). Fig. 6(b) is the moment diagram due to the applied load on the statically determinate structure.

With the use of formulas given in Tafel der Werte (6), the necessary coefficients can be obtained.

(Detail calculations are not shown)

EI
$$\delta_{10} = 2,295,000$$
 kip ft.
EI $\delta_{30} = -4,360,000$ kip ft.
EI $\delta_{10} = 78,000$ kip ft.
EI $\delta_{11} = 182,250$ kip ft.
EI $\delta_{31} = -141,750$ kip ft.
EI $\delta_{33} = 234,000$ kip ft.
EI $\delta_{21} = 4,725$ kip ft.
EI $\delta_{31} = -4,500$ kip ft.
EI $\delta_{31} = 150$ kip ft.

$$C_{21} = -\frac{\int M_1 m_2 \frac{d_5}{d_1}}{\int M_1 m_2 \frac{d_5}{d_2}} = -\frac{4,725}{150} = -31.5$$

$$C_{31} = -\frac{\int M_1 m_3 \frac{d_5}{d_1}}{\int m_1^2 \frac{d_5}{d_1}} = -\frac{4,500}{150} = 30$$

$$C_{32} = -\frac{\int M_2 m_3 \frac{d_5}{d_1}}{\int M_1^2 \frac{d_5}{d_1}} = -\frac{\int ((2_1 M_1 + m_2)m_3 \frac{d_5}{d_1} + 450 \times 31/5 - 141/50)}{\int ((2_1 M_1 + m_2)^2 \frac{d_5}{d_1} - \int ((2_2 M_$$

$$X_{1} = -\frac{\int M_{1}M_{0} \frac{d_{5}}{E_{L}}}{\int M_{1}^{2} \frac{d_{5}}{E_{I}}} = -\frac{7800}{150} = -520$$

$$X_{2} = -\frac{\int M_{2}M_{0} \frac{d_{5}}{E_{I}}}{\int M_{2}^{2} \frac{d_{5}}{E_{I}}} = -\frac{\int (C_{21}m_{1}M_{0} + m_{2}M_{0}) \frac{d_{5}}{E_{I}}}{\int ((c_{21}^{2}m_{1}^{2} + 2C_{21}m_{1}m_{2} + m_{2}^{2}) \frac{d_{5}}{E_{I}}}$$

$$= -\frac{31\cdot5 \times 78000 + 22.95000}{(31\cdot5)^{2} \times 150 + 2(-31\cdot5)(4725) + 182250}$$

= 4.85

$$X_{3} = -\frac{\int (C_{31}M_{1} + m_{3})M_{0} \frac{ds}{E_{1}}}{\int (C_{31}M_{1} + m_{3})^{2} \frac{ds}{E_{1}}}$$

= $\frac{30 \times 78000 + (-4360000)}{900 \times 150 + 60(-4500) + 234000}$

Final Moments:

$$\begin{split} M_{\Lambda} &= X_{1}M_{1} + X_{2}M_{2} + X_{3}M_{3} + M_{0} \\ &= -520(1) + 4.85 \quad (31.5x1 + 0) + 20.4 \quad (30x1 + 0) + 0 \\ &= -60 \; \text{kip ft.} \end{split}$$
$$\begin{split} M_{D} &= X_{1}M_{1} + X_{2}M_{2} + X_{3}M_{3} + M_{0} \\ &= 50 \; \text{kip ft.} \end{split}$$

$$M_{\rm B} = -520 + 4.85 (-31.5 + 45) + 20.4 (30 + 0)$$

= 156.5 kip ft.

 $M_{C} = -520 + 4.85 (-31.5 + 45) + 20.4 (30 - 60) + 1200$ = 131.5 kip ft.

Horizontal Reaction at A = $\frac{156.5 + 60}{45}$ = 4.82 Horizontal Reaction at D = $\frac{131.5 + 84.5}{45}$ = 4.82

The final moment diagram is shown in Fig. 6(f) To compare the solution by the method of combined redundants to the solution by the method of consistent deformations the same problem can be worked solving the resulting equations by the triangular method.

The pattern of the equations is:

$$150 x_1 + 4725x_2 + (-4500)x_3 = -78,000 \dots (I)$$

$$4725 x_1 \quad 182,250 x_2 + (-141,750)x_3 = -2,295,000 \dots (II)$$

$$-4500 x_1 + (-141,750)x_2 + (234,000)x_3 = 4,360,000 \dots (III)$$
Solving by the triangular method: (3)

$$\begin{cases} 150 \ x_1 + \frac{14725}{2} \ x_2 - \frac{14500}{3} \ x_3 = -78,000 \\ \left\{ \frac{14725}{150} \ (1) + (11) \right\}; \ 33250 \ x_2 \ 250 \ x_3 = 165,000 \\ \left\{ \frac{14500}{150} \ (1) + (111) \right\}; \ 250x_2 \ 99,000 \ x_3 = 2,020,000 \end{cases}$$
Solving the last two equations

$$(\frac{-250}{33250}); \qquad 33250 x_2 + 250 x_3 = 165,000$$

and $x_2 = \frac{250 x_2 + 999000 x_3 = 2,020,000}{x_3 = 20.4}$

Substituting these values of x, and x, in (I)

$$x_1 = -59.5$$

Moments at A = x_1 (1) = -9.5 kip ft.
Moments at D = -1200 + 59.5 + 60x20.4 = 84.0 kip ft.
Moments at B = 59.5 + (045)x4.8 = -156.5 kip ft.
Moments at C = -1200 + (-1) (-59.5) -45 (4.8) + 60 (20.4)
= -132 kip ft.

which gives the same bending moment diagram as shown in Fig. 6(f).

A truss shown in Fig. 7(a) with 8k load applied at point (14), acting downward.

Required: Bar stresses due to given load. (5) page 401. Solution: The method of combined redundants for higher degree of indeterminancy can be demonstrated by this problem. The given truss is statically indeterminate eight degrees internally. The truss is made statically determinate by cutting one of the diagonal members in each panel Fig. 7(b). The forces in the members due to the applied loads and the redundant forces are shown in Fig. 7(b) and (c) respectively. The systematic calculations for various combined redundant coefficients and the final bar stresses are tabulated in Table 2.

It is to be noted that calculations up to column (31) remain unaltered for different loadings.

In order to compare the combined redundant method with the method of consistent deformations the same problem is solved using Guass's Elimination Method to solve the resulting equations. (Table 4)

The equations are:

79.96	x ₁ +	20.45 2	^c 2 +	140.08	-	0
20.45	xil +	93.86 2	^c 2 +	20.45 x3 + 169.60 :	=	0
20.45	x2 +	112.86	×3+	20.45 x ₁ + 193.39	=	0
20.45	x3 +	123.68	x4+	20.45 x5+206.9	=	0
20.45	x1+ +	123.68	x5+	20.45 x6+206.9	=	0
20.45	×5 +	112.86	x6+	20.45 x7-11.30	-	0
20.45	x ₆ +	93.86	x ₇ +	20.45 x ₈ -508.60	-	0
20.45	x7 +	79.96	x ₈ -	339.40	=	0

The solution of simultaneous equation gives:

$$\begin{array}{rcl} x_8 & = & \frac{225 \cdot 7}{75 \cdot 3^2} = & 2 \cdot 93 \\ x_7 & = & \frac{500 \cdot 875 - 20 \cdot 45 \times 2 \cdot 93}{90 \cdot 0^4} = & 4 \cdot 86 \\ x_6 & = & \frac{41 \cdot 30 - 20 \cdot 45 \times 4 \cdot 86}{109 \cdot 38} = & - & 0 \cdot 53 \\ x_5 & = & -\frac{176 \cdot 7 + 20 \cdot 45 \times 0 \cdot 53}{109 \cdot 38} = & - & 1 \cdot 37 \\ x_4 & = & -\frac{176 \cdot 20 + 20 \cdot 45 \times 1 \cdot 37}{119 \cdot 81} = & - & 1 \cdot 23 \\ x_3 & = & -\frac{162 \cdot 59 + 20 \cdot 45 \times 1 \cdot 23}{108 \cdot 135} = & - & 1 \cdot 25 \\ x_2 & = & -\frac{133 \cdot 60 + 20 \cdot 45 \times 1 \cdot 25}{88 \cdot 635} = & - & 1 \cdot 21 \\ \end{array}$$

The detailed calculations are given in Table 3. The bar stresses by both methods check fairly well.

CONCLUSION

From the illustrative examples it can be observed that the combined redundant method eliminates the task of solving the simultaneous equations, which result from application of the method of consistent deformations. This is replaced by calculation of the coefficients of the combined redundants.

In Table 1, for different loading systems, the calculations remain unaltered except for columns 15 to 22. Therefore, by the principle of superposition the maximum bar stresses can be obtained by suitable combinations of loading systems. If the same problem were to be solved for maximum bar stresses by some other method, for example, consistent deformations, then, for each system of loading, the magnitude of the redundant forces must be calculated by solving the simultaneous equations each time. This is a very tedious and time-consuming approach. Thus it can be seen that this method has an advantage if the maximum bar stresses are to be calculated.

For a flexural structure, the calculations involved by the combined redundant method are the calculations for coefficients of combined redundants. Thus, the problem of solving simultaneous equations is eliminated. The redundant forces in this method, of combined redundants, are quite different from those obtained by the consistent deformation method.

It was pointed out by Steven J. Fenves² A.M. ASCE (8), that in common types of trusses, if the number of members greatly exceeds the number of redundants, the combined redundant method does not have any computational advantage. Moreover, for higher degree of indeterminancy the accuracy is affected by this method, the same author, Steven J. Fenves, discovered that, for computer programming, this method involves fewer operations than by ordinary methods, that is, matrix inversion by the computer.

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В С



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Fig. 4

FIG. 3



FIG. 5: PROBLEM NO. 1 METHOD OF COMBINED REDUNDANTS







FIG. 6: PROBLEM NO. 2: FLEXURAL MEMBERS BY COMBINING REDUNDANTS









Table 1. Third Degree Redundant Frame by the Method of Combining Redundants

:+0°039 -0.487 +0.417 +3.775 +0.027 +0.027 +0.027 25 SS +0°20+ +0.64 +0.16 :+0°01 :+0.01 +0.01 +0.64 10 0 +1.0 0 -0.118 S2 S3 :+0.059 :-0.164 :+0°64 0 0 0 თ 0 0 0 0 S₁s₃ :+0°225 :-0.625 -0°82 :-0.45 ස 0 0 0 0 0 0 0 0 :+0.197 :-0°08 :-0°08 :-0.098 :-0.698 :+0.164 : **-**0°098 :-0°098 :+0.164 :+0.836 :-0.164 :-0.403 -0°80 -0.80 ~ s~ 0 :+1°0 1 +0.164:-0,098 :-0°098 :+0.197 :+0.197 +0.164 :-0.164 c21⁵1 :-0.164 0 9 0 .0 1 S15 :+0.225 :-0-625 :-0°45 0 :-0.85 0 0 0 5 0 0 0 0 :+3.254 +0.141 +0.391 :+0.563 :+0.563 +0,141 :+0.391 :+0.141 :+0°141 :+0.391 2'S +0.39 0 4 0 0 0 0.60 -0°80 -0.60 -0°80 +1.0 +1°0 3 5 0 0 0 0 0 0 0 0 1 -0.80 -0.60 -0.80 -0.60 +1.0 -1°0 0 0 0 2 s's 0 0 0 s1#S1 :+0°625 -0.375 -0°625 -0°375 :-0.375 -0.375 -0.625 +0.625 +0.75 +0°22 0 0 1 ... AB BC 8 DE ΕF FG GH Η βĢ BH DG ß DF CF

Table 1 (concl.)

-1.49 -3.16 +7.56 +0.30 +1.51 =0,92 -0.69 -1.69 -4.14 +5°52 -5,99 +0.26 +3°58 +2°05 -0.35 S 22 1 4.42: +0.90: -0.06: -0.03: -0.89: +1.92: +5.31: +1.08: +0.16: +0.27: -0°08: -2.66: -0.54: +0.03: -0.54: +0.26: -0.54: +0.03: -4.42: +4.61: +0.10: -7.03: -1.85: -1.65: +4.42: +0.90: -0.06: -3.84: +0.01: -0.32 +0°30: +0°0+ -0.35: x₃s3 -4.41: +5.48 X₁S₁: X₂S₂: +1.82; -1.09; +5.31; -2.22; -4.42° -0.90° -4,41: 0 0 -2.66: -2.66: -2°66: 19 0 s S S S -0.22: -0.20: -1.54 -4.72: -0.14: -0°55: -0°67: -0.59: -3.13: +0.97: -0.62: +3.21: -9°6 : +0°93: +1°32: 0 0 0 C sos: -0°62: =0.84: -0.22 : -22.88:-20.71: 17 0 0 0 C -3°37° -0.164*0.0181*+0.854*+0.73 *+ 3.75* -2.35* sos1: -3°37° -2.35: -0.84: -2.54: -2°54 +2°35: 16 0 :+0.164+0.0181:+0.146:+0.021:=11.25: :+0.197:0.0446:+0.242:+0.059:- 4.50: +0,197:0.0218:-0.425:+0.181:- 4.50: -0.098:0.0109:-0.087:+0.008:+ 2.25: -0.098:0.0771:-0.021:+0.004:+ 6.75: -0.098:0.0109:-0.087:+0.008:+ 6.75: -0.164:0.0924:-0.256:+0.065:- 3.75: \$\$\$.0885:+0.089:+0.008:+12.0 \$ +0.164:0.0181:+0.146:+0.021:- 3.75: -0.098:0.0109:-0.687:+0.472:+ 2.25: ം 15 0.0885:-0.712:+0.507 :--.1105:0.1105:+0.012: :+3°736: -0.80 :+0.64 :+1 •0 255 14 ŝ :+1°0 1 :c3151 :c3252 : 0 12 0 ... 11 0 0 0 BAR AB BC 8 DE EF FG ů ЪF HO GH H BG Ð СF

EIGHTH DECREE REDUNDANT FRAME BY COMBINING REDUNDANTS : TABLE 2

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3-5 3-6 3-6 3-6 3-4	8 32 324 24 24	-3.0 1.5 -2.0 -2.0 2.5	6 6 8 8 1.0 1.0	2.88 2.88 20.45 20.45 20.45 24.0 24.0	0203 <u>7</u> 20.45	140.08 14.40 -7.20 51.20 51.20 60.0
5-7 6-8 5-6 7-8 5-8 7-6	5.76 5.76 32.0 32.0 34.30 34.30	-4.5 3.0 -2.0 -2.0 2.5	6 8 8 1.0 1.0	93.86 2.08 20.45 20.45 34.30 34.30	₩ ₃ ₩ ₄ ½ 20.45	169.60 15.57 -10.38 51.20 51.20 85.80 0
7-9 8-10 7-8 9-10 7-10 9-8	4.97 4.97 32.0 32.0 40.0 40.0	-6.0 -4.5 -2.0 -2.5 0	6 8 8 8 1.0 1.0	112.86 1.79 1.79 20.45 20.45 40.0 40.0	U4U5 A 20.45	193.39 17.90 -13.40 51.20 51.20 100.0 0
9-11 10-12 11-12 9-10 11-10 9-12	4.97 32.0 32.0 40.0 40.0	-7.5 6.0 -2.0 -2.0 0 2.5	6 8 8 1.0 1.0	123.68 1.79 20.45 20.45 40.0 <u>40.0</u>	20.45	206.90 20.40 -17.90 51.20 0 100.0
11-13 12-14 13-14 11-12 13-12 11-14	5.76 5.76 32.0 34.30 34.30	-9.0 7.5 6.0 -2.0 2.5	6 6 8 8 1.0 1.0	2.08 2.08 20.45 20.45 34.30 34.30	0607 <u>7</u> 20.45 0708 <u>1</u>	206.90 31.10 -25.90 -153.50 51.20 0 <u>05.80</u>
13-15 14-16 15-16 13-14 15-14 13-16	8 32 21 21 21 21 21	-4.5 9.0 6.0 0 -7.5	6 8 8 1.0 1.0	2.88 2.88 20.45 20.45 24.0 24.0 93.86	20.45	-11.30 21.6 -43.2 -153.60 -153.60 -153.60 -180.0 -508.60

Table 3. Eighth degree redundant frame by consistent deformation method.

TABLE 3 (concl.).

BAR	:	L/A	:	s	:	Un	:	υ ² n	L	:	υ,υ,,	L:	sUn	L
15-17 16-18 17-18 15-16 17-16 15-18		14.40 14.40 17.45 32.0 19.20 19.20		0 4.5 0 6.0 0 7.5		6 8 8 1.0 1.0		5. 5. 20. 19. <u>19.</u> 79.	18 18 15 65 20 96				0 - 38 0 -153 -153 -144 -144 -339	90 60

Method	
Elimination	
Gauss's	
4.	
Table	

Gau. s's Elimination Method	×	×2 :::	× 53	× *	×	× × ×	× 8 	× =	K,
<u>38848858</u>	: 79.96 20.45	20.45 93.86 20.45	20.45 112.86 20.45	20,45 123,68 20,45	20.45 123.68 20.45 1	20.45 20.45 20.45 20.45 20.45 33.85 20.45 20.45	20°45 20°45 20°57 20°57	140.08: 193.08: 206.90: 206.90: 206.90: 339.40:	
EQUATION	×	×~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	×	×*	 ב	× • · · × • · · ×	* 8 X	×	Check
Equation (1) Sum $I \in Eq_{\circ}$ (1) Equation (2) Multi Sum I by2555	79,96 79,96 20,45 -20,45	20.45 20.45 93.86 =5.225	20.45				7777	140.08 140.08 169.60 +36.0 +	-34 .84 10.325
Equation (3) Equation (3)		88.635 20.45 0	20.45 112.860	20°45			TT	133.60" 193.39	24.515 39.63
Multi Sum II by231 Equation (4)	0	=20.45 0	-4°725 108,135 20,45	0 20.45 123.68	20°45		I I	30.80 162.59 206.90	5.625 36.005 -42.32
Aulti Sum I by O Aulti Sum II by O Aulti Sum III by189 Sum IV	0	00	0 -20.45 0	-3.87 119.81	20°45		Ţ	0 0 +30 ° 7 176 ° 20	0 0 6,38 -35,94
Equation (5) Multi Sum 1 by O Multi Sum 11 by O Multi Sum 111 by O	0	00	00	20•45 0	123 。 68	20 . 45		00°°00	-42.32 0 0
Witt Sum IV by1715 Sum V by1715 Equation (6) Wuiti Sum I by O				-20°45 0	-3.51 120.17 20.45	20.45 112.86 20.45	ī	30.2 176.7 11.30	6.24 -36.08 165.06
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MTION	V by 0 by -0.17 n (7) by 0 1 by 0 11 by 0 V by 0	Dy 0 1 by187 n (8) by 0 1 by 0 V by 0 by 0 by 0	II by -0.2. III by -0.2.
EQU	1t1 Sum V1t1 Sum VSum VISum VIEquatio1t1 Sum I1t1 Sum I1t1 Sum I1t1 Sum I	tti Sum V Sum VII Equatio Equatio Eti Sum I Lti Sum I Lti Sum I ti Sum I ti Sum V	ti Sum V ti Sum V Sum V

ANALYSIS OF INDETERMINATE STRUCTURES BY COMBINING REDUNDANTS

by

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B.E. (Civil), (Hons.), S. V. UNIVERSITY, GUJARAT-STATE, INDIA 1961

AN ABSTRACT OF A MASTER'S REPORT

submitted in partial fulfillment of the

requirements for the degree

MASTER OF SCIENCE

Department of Civil Engineering

KANSAS STATE UNIVERSITY Menhattan, Kansas

1964

Approved by:

Major Professor

ABSTRACT

In the analysis of highly indeterminate structures, the tesk of setting up and solving the elastic equations becomes timeconsuming. Several methods have been developed to reduce the work considerably. This paper illustrates the method of combining redundants. By suitably combining the redundant forces, en orthogonalized form of simultaneous equations is achieved. These can easily be solved for unknown redundant forces. Three illustrative examples are solved. To compare this method with another method, two of the problems are also solved by the method of consistent deformations.

This method of combining redundents has great advantage when the bar stresses are to be calculated for different loading conditions. For flexural structures, it is well suited. This method does not show any computational advantage, when the structure is highly indeterminate. In such cases, accuracy is also affected since more operation of multiplications are involved.